# RULES FOR THE SURVEY AND CONSTRUCTION OF STEEL SHIPS

GUIDANCE FOR THE SURVEY AND CONSTRUCTION OF STEEL SHIPS

Part C

**Hull Construction and Equipment** 

Rules for the Survey and Construction of Steel Ships
Part C 2023 AMENDMENT NO.1
Guidance for the Survey and Construction of Steel Ships
Part C 2023 AMENDMENT NO.1

Rule No.29 / Notice No.28 30 June 2023 Resolved by Technical Committee on 25 January 2023



An asterisk (\*) after the title of a requirement indicates that there is also relevant information in the corresponding Guidance.

# RULES

# RULES FOR THE SURVEY AND CONSTRUCTION OF STEEL SHIPS

Part C

**Hull Construction and Equipment** 

#### 2023 AMENDMENT NO.1

Rule No.29 30 June 2023

Resolved by Technical Committee on 25 January 2023

An asterisk (\*) after the title of a requirement indicates that there is also relevant information in the corresponding Guidance.

Rule No.29 30 June 2023 AMENDMENT TO THE RULES FOR THE SURVEY AND CONSTRUCTION OF STEEL SHIPS

"Rules for the survey and construction of steel ships" has been partly amended as follows:

# Part C HULL CONSTRUCTION AND EQUIPMENT

# Part 1 GENERAL HULL REQUIREMENTS

# Chapter 1 GENERAL

- 1.1 General
- 1.1.2 Application
- **1.1.2.1** General

Sub-paragraph -1 has been amended as follows.

- 1 The requirements in **Part** C apply to ships constructed of welded steel structures, composed of stiffened plate panels, and having a length  $\cancel{\underline{L}}$  (as defined in 2.1.2, Part A) of not less than 90 m, and intended for unrestricted service.
- 1.4 Symbols and Definitions
- 1.4.2 Primary Symbols and Units
- 1.4.2.2 Ship's Main Data

Unless otherwise specified, the symbols of a ship's main data and their units used in Part C are those defined in Table 1.4.2-2.

Table 1.4.2-2 has been amended as follows.

Table 1.4.2-2 Ship's Main Data

Symbol	Meaning	Unit
$L_C$	Ship length, but to be taken as 90 m where not greater than 90 m (See 1.4.3.1-1)	m
(Omitted)		

#### 1.4.3 Definitions

# 1.4.3.1 Principal Particulars

Sub-paragraph -1 has been amended as follows.

- 1 The ship length  $L_C$  is to be in accordance with the following (1) to (3), but is to be taken as 90 m in cases where not greater than 90 m:
- (1) The ship length  $L_C(m)$  is to be the distance measured on the waterline at the scantling draught  $T_{SC}$  from the forward side of the stem to the after side of the rudder post for ships with a rudder post or to the centre of the rudder stock for ships without a rudder post.  $L_C$  is to be not less than 96% but need not exceed 97% of the extreme length on the waterline at the scantling draught  $T_{SC}$ .
- (2) In ships without a rudder stock (e.g., ships fitted with azimuth thrusters), the ship length  $L_C$  is to be taken as equal to 97 % of the extreme length on the waterline at the scantling draught  $T_{SC}$ .
- (3) In ships with an unusual stem or stern arrangements, the ship length is to be deemed appropriate by the Society on a case-by-case basis.

# **Chapter 3 STRUCTURAL DESIGN PRINCIPLES**

# 3.3 Net Scantling Approach

# 3.3.3 Corrosion Model for Strength Assessment

# 3.3.3.1

Table 3.3.3-1 has been amended as follows.

Table 3.3.3-1 Assessment of Corrosion Applied to the Gross Scantlings

Structural requirement	Property/analysis type	Applied corrosion addition
	(Omitted)	
	FE model <sup>(1)</sup>	$0.5~t_c$
Strength assessment by cargo hold analysis	Buckling strength	$t_c$
	(Omitted)	
	FE model <sup>(<u>al</u>)</sup>	$0.5~t_c$
Torsional strength assessment by whole ship analysis	Buckling strength	$t_c$
	(Omitted)	
Fatigue strength assessment (finite element analysis)	Finite element model <sup>(1)</sup>	$0.25\ t_c$
Torsional fatigue strength assessment by whole ship analysis	Finite element model <sup>(21)</sup>	$0.25t_c$
Whole ship analysis (other than torsional yield strength assessment and fatigue strength assessment)	Finite element model <sup>(21)</sup>	0.5 t <sub>c</sub>
(Notes)  (1) No consideration of corrosion addition is requi  (≟1) No consideration of corrosion addition is requi	8	

<sup>(&</sup>lt;u>21</u>) No consideration of corrosion addition is required for members not affecting strength assessment results.

# 3.5 Minimum Requirements

# 3.5.2 Slenderness Requirements

Paragraph 3.5.2.1 has been amended as follows.

# 3.5.2.1 Application

All structural members are to meet the slenderness requirements specified in. Where structural members are deemed by the Society as having effectiveness equivalent to those compliant with the requirements in 3.5.2, such structural members are to be deemed compliant with 3.5.2. 3.5.2, except for those listed below:

- Bilge plates within the cylindrical part of the ship and the radius gunwale
- Structure members in superstructures and deck houses in cases where such members do not contribute to longitudinal strength.

Pillars in superstructures and deckhouses are to comply with the applicable slenderness and proportion requirements specified in 3.5.2.

- Where structural members are deemed by the Society as having an effectiveness equivalent to those compliant with 3.5.2, such members are to be deemed compliant with 3.5.2.
- 23 Notwithstanding -1 above, thickness of shell plating, watertight deck, bulkhead and web of girder and stiffness of stiffener need not to comply with 3.5.2, provided that buckling strength requirements specified in 5.3 and 8.6.2 are satisfied.

#### 3.5.2.2 Thickness of Various Structural Members

1 The thickness t (mm) of various structural members is to satisfy the following criteria:

$$t \ge \frac{b}{C} \sqrt{\frac{\sigma_Y}{235}}$$

b: For plating, b is to be taken as the plate breadth (mm)

For webs, b is to be taken as the web depth (mm). However, where the stiffener is provided on the web, b may be taken as the maximum breadth taking the stiffener into account.

For face plates, b is to be taken as the half breadth of the face plate (mm)

For circular section pillars, b is to be taken as their mid-thickness radius (mm)

C: Slenderness coefficient as specified in Table 3.5.2-1

Table 3.5.2-1 has been amended as follows.

Table 3.5.2-1 Slenderness Coefficients

TYPE OF STRUCTURAL MEMBER	С
Shell plating <del>, watertight bulkheads and girder webs</del>	Ü
Upper decks	
Inner bottom plating	100
Girder webs	
Watertight <sup>(1)</sup> and ₩non-tight bulkheads	
Watertight decks <sup>(1)</sup>	125
Non-watertight decks within cargo hold regions	
Non-watertight decks outside cargo hold regions	<u>150</u>
Angles and T-section stiffener webs	75
Webs of bulb sections	45
Webs of pillars	35
Flat bars	22
Face plates of stiffeners and girders	15
Face plates of pillars	12
Circular section pillars	50
Note:	
(1) This includes deep tank boundaries.	

# 3.5.2.4 Stiffness of Stiffeners

Sub-paragraph (3) has been amended as follows.

The scantlings of stiffeners are to comply with one of the following (1) to (3) as applicable depending on the structure:

(1) For longitudinal stiffeners

The value of the moment of inertia  $I_{st}$  ( $cm^4$ ) is to satisfy the following:

$$I_{st} \ge 1.43\ell^2 A_{eff} \frac{\sigma_Y}{235}$$

 $A_{eff}$ : Sectional area  $(cm^2)$  of stiffeners with attached plating, taking into account the effective breadth specified in 3.6.3

 $\sigma_Y$ : Specified minimum yield stress ( $N/mm^2$ ) of attached plating

(2) For non-longitudinal stiffeners attached to watertight members

The value of the moment of inertia  $I_{st}$  ( $cm^4$ ) is to satisfy the following:

$$I_{st} \ge 0.72\ell^2 A_{eff} \frac{\sigma_Y}{235}$$

 $A_{eff}$ : Sectional area ( $cm^2$ ) as specified in (1) above

 $\sigma_Y$ : Specified minimum yield stress  $(N/mm^2)$  as specified in (1) above

(3) For non-longitudinal stiffeners attached to a non-watertight member

The value of the moment of inertia  $I_{st}$  (cm<sup>4</sup>) is to satisfy (2) above. However, in the case of flat bar stiffeners, it is also acceptable if gross scantling depth  $h_{w-gr}$  (mm) of flat bar stiffeners satisfies the following, where the stiffener may be a flanged stiffener with an equivalent moment of inertia:

$$h_{w-gross} \ge \frac{\ell}{12} \times 10^3$$

# 3.5.2.9 Edge Reinforcement in Way of Openings

Sub-paragraph -1 has been amended as follows.

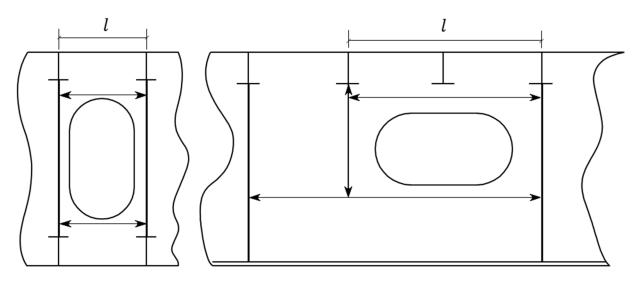
1 The depth of web  $h_{w-gr}$  (mm) of edge stiffeners fitted in way of openings (See Fig. 3.5.2-1) is to satisfy the following:

$$h_{w-gr} \ge \max\left(50, 0.05l\sqrt{\frac{\sigma_Y}{235}}\right)$$

<u>l</u>: Length of edge stiffener in way of opening as defined in Fig. 3.5.2-1 (m)

2 The thickness of the web and flange of the edge stiffener are to satisfy in 3.5.2.2 and 3.5.2.3.

Fig. 3.5.2-1 Typical Edge Reinforcements



Chapter 4 LOADS

# 4.4 Loads to be Considered in Local Strength

# 4.4.2 Maximum Load Condition

# 4.4.2.4 Internal Pressure Due to Liquid Loaded

1 Static pressure  $P_{ls}$  ( $kN/m^2$ ) acting on tanks or ballast holds loaded with liquids is to be as specified in **Table 4.4.2-5**.

Table 4.4.2-5 has been amended as follows.

Table 4.4.2-5 Internal Static Pressure  $P_{ls}$  in Tanks and Ballast Holds Loaded with Liquids

T 1 - 1 - 1	Static pressure $P_{ls}$ ( $kN/m^2$ )	
Type of tank or hold	$z \le z_{top}$	$z > z_{top}$
Cargo tanks fully loaded with liquid cargo (excluding liquefied gas)	$\rho_L g(z_{top} - z) + P_{PV}$	0
Cargo tanks fully loaded with liquefied gas and liquefied gas fuel tanks	$\rho_L g(z_{top} - z) + P_0$	0
Ballast holds	$ ho_L g(z_{top}-z)$	0
Ballast tanks and other tanks	$\rho_L g(z_{top} - z + 0.5h_{eff}) - P_{BAL}$	0

#### Notes:

 $\rho_L$ : Density of liquid loaded ( $t/m^3$ ), as specified in Table 4.4.2-6

 $z_{top}$ : Z coordinate of the highest point (m) of the tank, excluding small hatchways (m)

#### har: Height of air pipe or overflow pipe above the top of the tank (m)

 $P_{PV}$ : Design vapour pressure ( $kN/m^2$ ), but not to be taken less than 25  $kN/m^2$ 

 $P_0$ : Design vapour pressure ( $kN/m^2$ ). For cargo tanks, this value is not to be less than the MARVS specified in 1.1.4, Part N. For liquefied gas fuel tanks, this value is not to be less than the MARVS specified in 2.2.1, Part GF.

 $P_{BAL}$ : Hydrostatic pressure  $(kN/m^2)$  at ballast draught  $T_{BAL}$  (m) considered as a component for offsetting internal pressure, as specified in **Table 4.4.2-7**. For members subject to simultaneous external and internal pressures, the <u>actual</u> hydrostatic pressure <u>value</u> is to be considered. For <u>other</u> members <u>other than it</u>, the value is to be <u>taken as</u> 0.

#### 4.4.3 Testing Condition

Paragraph 4.4.3.1 has been amended as follows.

#### 4.4.3.1 External Pressure

External pressure  $P_{ST-dx}$  (kN/m<sup>2</sup>) acting on the hull is to be the hydrostatic pressure corresponding to the draught described in the test plan approved by the Society in accordance with the requirements in 2.1.5, Part B. The following (1) and (2) are to be considered as external pressure acting on the hull.

- (1) Case 1 ( $P_{ST-ex1}$ ): Hydrostatic pressure ( $kN/m^2$ ) corresponding to the draught described in the test plan at Classification Surveys during Construction approved by the Society in accordance with 2.1.5, Part B.
- (2) Case 2 ( $P_{ST-ex2}$ ): Hydrostatic pressure ( $kN/m^2$ ) corresponding to the draught  $T_{BAL}$  or  $0.33T_{SC}$ , whichever is smaller. However, where the draught of pressure tests at Special Surveys is given in the loading manual, the hydrostatic pressure corresponding to said draught may be considered instead.

#### 4.4.3.2 Internal Pressure

Sub-paragraphs -1 and -3 have been amended as follows.

- 1 Internal pressure  $\frac{P_{ST-in}}{P_{ST-in1}}$  and  $P_{ST-in2}$   $(kN/m^2)$  acting on the hull and tanks is to be in accordance with Table 4.4.3-1.
- When a hydrostatic test is conducted under a condition exceeding the pressure specified in -1 above, the actual pressure which occurs in the test is to be used.
- 3 When external pressure is considered as a component offsetting the internal pressure in a member subjected to simultaneous internal and external pressures, the external pressure specified in 4.4.3.1 may be considered, identical cases are to be combined.

Table 4.4.3-1 has been amended as follows.

Table 4.4.3-1 Internal Pressure  $P_{ST-in1}$  and  $P_{ST-in2}$  in Testing Condition

		1 31 = th
Position ur	nder consideration	Internal pressure $P_{ST-in1} P_{ST-in1}$ and $P_{ST-in2} (kN/m^2)$
<u>Case 1</u>	$z \le z_{ST}$	$\frac{P_{ST=in} - \rho g(z_{ST} - z)}{P_{ST=in1} = \rho g(z_{ST} - z)}$
	$z > z_{ST}$	0
G 2	$z \le z_{top}$	$P_{ST-in2} = \rho g(z_{top} + h_{air} - z) + 25$
Case 2	$z > z_{top}$	<u>0</u>
NI-4		

Notes:

 $z_{ST}$ : Height  $\underline{(m)}$  of water head  $\underline{(m)}$  in  $\underline{\text{for}}$  hydrostatic test, as specified in Table 4.4.3-2

 $\underline{z_{top}}$ : Z coordinate of the highest point (m) of the tank, excluding small hatchways

 $h_{air}$ : Height (m) of air pipe or overflow pipe above the top of the tank

(1) Compartments to be assessed are ballast tanks only.

Table 4.4.3-2 Design Testing Water Head Height  $z_{ST}$  (Omitted)

# **4.4.3.3** Vertical Bending Moment

(Omitted)

# 4.5 Loads to be Considered in Strength of Primary Supporting Structures

#### 4.5.3 Harbour Condition

#### 4.5.3.2 Internal Pressure

Sub-paragraphs -2 and -4 have been amended as follows.

- 1 Loads due to cargoes, ballast and other loaded materials are to be considered as the internal pressure acting on the hull.
- 2 Internal pressure  $P_{PT-in}$  ( $kN/m^2$ ) acting on tanks loaded with liquids and ballast holds is to be in accordance with the following formula<u>e</u>:

$$\frac{P_{DT-IM} = P_{IC}}{P_{IC}}$$

For ballast tanks,  $P_{PT-in} = P_{ls} + \rho_L g h_{air}$ 

For other tanks,  $P_{PT-in} = P_{ls}$ 

 $P_{ls}$ : Static pressure  $(kN/m^2)$ , as specified in 4.4.2.4-1

 $\rho_L$ : Density of liquid loaded ( $t/m^3$ ), as specified in Table 4.4.2-6

 $h_{air}$ : Height of air pipe or overflow pipe above the top of the tank (m)

3 Internal pressure  $P_{PT-in}$  ( $kN/m^2$ ) acting on cargo holds loaded with dry bulk cargoes in the fully loaded and partially loaded conditions is to be in accordance with the following formula:

$$P_{PT-in} = P_{bs}$$

 $P_{bs}$ : Static pressure ( $kN/m^2$ ), as specified in **4.4.2.5-1** 

4 Internal pressure  $P_{xs}$   $(kN/m^2)$  not corresponding to -2 and -3 above is to be the value calculated by dividing the weight of the loaded material (kN) by the area  $(m^2)$  in the range subject to the said loaded material, and is to be considered as a line load or a point load depending on the type of the loaded material.

# 4.6 Loads to be Considered in Strength Assessment by Cargo Hold Analysis

#### 4.6.2 Maximum Load Condition

# 4.6.2.5 Internal Pressure Due to Liquid Loaded

1 Static pressure  $P_{ls}$  ( $kN/m^2$ ) acting on tanks and ballast holds loaded with liquids is to be in accordance with **Table 4.6.2-11**.

Table 4.6.2-11 has been amended as follows.

Table 4.6.2-11 Static Pressure  $P_{ls}$  in Tanks and Ballast Holds Loaded with Liquids

Tong of Asologia 11	Static pressure $P_{ls}$ ( $kN/m^2$ )	
Type of tank or hold	$z \le z_{top}$	$z > z_{top}$
Cargo tanks fully loaded with liquid cargo (excluding liquefied gas)	$\rho_L g(z_{top} - z) + P_{PV}$	0
Cargo tanks fully loaded with liquefied gas and liquefied gas fuel tanks	$\rho_L g(z_{top} - z) + P_0$	0
Ballast holds	$ ho_L gig(z_{top}-zig)$	0
Ballast tanks and other tanks	$ \rho_L g(z_{top} - z + 0.5h_{eff}) $	0

Notes:

 $\rho_L$ : Density of liquid loaded ( $t/m^3$ ), as specified in Table 4.6.2-12

 $z_{top}$ : Z coordinate of the highest point  $\underline{(m)}$  of tank, excluding small hatchways  $\underline{(m)}$ 

har: Height of air pipe or overflow pipe above the top of the tank (m)

 $P_{PV}$ : Design vapour pressure  $(kN/m^2)$ , not to be less than 25  $kN/m^2$ 

 $P_0$ : Design vapour pressure ( $kN/m^2$ ). For cargo tanks, <u>this value is</u> not to be less than <u>the MARVS</u> specified in **1.1.4**, **Part N**. For liquefied gas fuel tanks, this value is not to be less than the MARVS specified in **2.2.1**, **Part GF**.

#### 4.6.3 Harbour Condition

#### 4.6.3.3 Internal Pressure

Sub-paragraphs -2, -4 and -5 have been amended as follows.

- 1 Loads due to cargoes, ballast and other cargoes are to be considered as internal pressure acting on the hull.
- 2 Internal pressure  $P_{PT-in}$  ( $kN/m^2$ ) acting on tanks and ballast holds loaded with liquids is to be in accordance with the following formula:

$$\frac{P_{PT-IN}-P_{IC}}{P_{IC}}$$

For ballast tanks,  $P_{PT-in} = P_{ls} + \rho_L g h_{gir}$ 

For other tanks,  $P_{PT-in} = P_{ls}$ 

 $P_{ls}$ : Static pressure  $(kN/m^2)$ , as specified in 4.6.2.5-1

 $\rho_L$ : Density of liquid loaded ( $t/m^3$ ), as specified in Table 4.6.2-12

 $h_{air}$ : Height of air pipe or overflow pipe above the top of the tank (m)

3 Internal pressure  $P_{PT-in}$  ( $kN/m^2$ ) acting on cargo holds loaded with dry bulk cargoes in the full and partial load conditions is to be in accordance with the following formula:

$$P_{PT-in} = P_{bs}$$

 $P_{hs}$ : Static pressure  $(kN/m^2)$ , as specified in **4.6.2.6-1** 

4 <u>Characteristics</u> Container load  $F_{in\_hold}$  (kN) acting on cargo holds  $F_{in\_hold}$  (kN) and the container load  $F_{on\_deck}$  (kN) acting on hatch coamings, etc.  $F_{on\_deck}$  (kN) loaded with container cargoes are to be in accordance with the following formulae:

$$F_{in\_hold} = F_{cs}$$
$$F_{on\_deck} = F_{cs}$$

 $F_{cs}$ : Static load (kN) as specified in **4.6.2.7-2** 

Internal pressure  $P_{xs}$  ( $kN/m^2$ ) due to loaded materials not corresponding to -2 to -4 above is to be calculated by dividing the weight of the <u>loaded</u> material (kN) by the area ( $m^2$ ) in the range subject to the <u>said loaded</u> materials and is to be considered as a line load or a point load depending on the type of the <u>eargoloaded material</u>.

# 4.7 Loads to be Considered in Fatigue

#### 4.7.2 Cyclic Load Condition

# 4.7.2.5 Internal Pressure Due to Liquid Loaded

1 Static pressure  $P_{ls}$  ( $kN/m^2$ ) acting on tanks and ballast holds loaded with liquids is to be in accordance with **Table 4.7.2-7**.

Table 4.7.2-7 has been amended as follows.

Table 4.7.2-7 Static Pressure  $P_{ls}$  in Tanks and Ballast Holds Loaded with Liquids

T C. 1 11 11	Static pressure $P_{ls}$ ( $kN/m^2$ )	
Types of tank and hold	$z \le z_{top}$	$z > z_{top}$
Cargo tanks fully loaded with liquid cargo (excluding liquefied gas)	$\rho_L g(z_{top} - z) + P_{PV}$	0
Cargo tanks fully loaded with liquefied gas and liquefied gas fuel tanks	$\rho_L g(z_{top} - z) + P_0$	0
Ballast holds	$ \rho_L g(z_{top} - z) $	0
Ballast tanks and other tanks	$\rho_L g(z_{top} - z + 0.5h_{eff})$	0

#### Notes:

 $\rho_L$ : Density of liquid loaded ( $t/m^3$ ), as specified in Table 4.6.2-12

 $z_{top}$ : Z coordinate of the highest point (m) of the tank, excluding small hatchways (m)

#### hat: Height of air pipe or overflow pipe above the top of the tank (m)

 $P_{PV}$ : Design vapour pressure  $(kN/m^2)$  but not to be less than 25  $kN/m^2$ 

 $P_0$ : Design vapour pressure ( $kN/m^2$ ). For cargo tanks, <u>this value is</u> not to be less than <u>the MARVS</u> specified in **1.1.4**, **Part N**. For liquefied gas fuel tanks, <u>this value is</u> not to be less than <u>the MARVS</u> specified in **2.2.1**, **Part GF**.

#### 4.8 Loads to be Considered in Additional Structural Requirements

#### 4.8.2 Maximum Load Condition

# **4.8.2.1** General

Sub-paragraph -3 has been added as follows.

- 1 Slamming loads acting on the bottom structure are to be in accordance with **4.8.2.2**.
- 2 Loads due to bow impact are to be in accordance with 4.8.2.3.
- 3 Sloshing loads are to be in accordance with 4.8.2.4.

#### 4.8.2.4 Sloshing Loads

- Loads to be considered in tank structures for the sloshing specified in 10.9 are to be in accordance with 4.8.2.4. The sloshing loads to be considered for filling ratios are to be in accordance with Table 4.8.2-10. These requirements apply to tank structures where the ratio of tank height to tank length is not less than 1/4 and not more than 4.0 for sloshing loads due to pitch, and where the ratio of tank height to tank breadth is not less than 1/4 and not more than 4.0 for sloshing loads due to roll.
- Where the relationship between the natural period of tanks in which liquid height is considered and the natural period of ship motions corresponds to the following (1) and (2), sloshing loads may be partially omitted from consideration. In applying the following (1) and (2),  $T_{tk-X}$  and  $T_{tk-Y}$  may be calculated using the tank length or breadth without considering the effects of girders and/or wash bulkheads instead of using equivalent tank length  $\ell_e(m)$  or equivalent tank breadth  $b_e(m)$ . These natural periods of tanks are to be calculated in accordance with the following (3).
- (1) Where the period of longitudinal oscillation  $T_{tk-X}(s)$  of cargo tanks is not within the range of  $\pm 20$  % of pitch period  $T_{\phi\_slh}$  specified in **Table 4.8.2-11** and not within  $\pm 1.5$  seconds from the same period, sloshing loads due to pitch need not be considered. In calculating  $T_{\phi\_slh}$ , the parameters for the ballast condition are to be used.
- Where the period of transverse oscillation  $T_{tk-y}(s)$  of cargo tanks is not within the range of  $\pm 20$  % of roll period  $T_{\theta}$  specified in 4.2.2.1 and not within  $\pm 1.5$  seconds from the same period, sloshing loads due to roll need not be considered. In calculating  $T_{\theta}$ , the parameters for the ballast condition are to be used.
- (3)  $T_{tk-X}(s)$  and  $T_{tk-Y}(s)$  are to be in accordance with the following formulae. The periods of tanks are to be calculated for each 10 % of the filling ratio  $(0.1f_r)$ .

$$T_{tk-X} = \frac{2\pi}{\sqrt{\frac{\pi}{\ell_e} \cdot g \cdot \tanh\left(\frac{\pi}{\ell_e} h_{lc}\right)}}$$

$$T_{tk-Y} = \frac{2\pi}{\sqrt{\frac{\pi}{b_e} \cdot g \cdot \tanh\left(\frac{\pi}{b_e} h_{lc}\right)}}$$

$$\ell_e : \text{Equivalent tank length } (n)$$

 $\underline{\ell_e}$ : Equivalent tank length (m) for calculating the natural period as specified in Table 4.8.2-12

 $\underline{b_e}$ : Equivalent tank breadth (m) for calculating the natural period as specified in Table 4.8.2-12

 $h_{lc}$ : Liquid height under consideration (m)

Table 4.8.2-10 Filling Ratios and Sloshing Loads

Filling ratio $f_r$	Sloshing load
$0.2 \le f_r < 0.4$	Sloshing loads for low filling ratio
$0.4 \le f_r < 0.7$	Sloshing loads for middle filling ratio
$0.7 \le f_r \le 0.9$	Sloshing loads for high filling ratio
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#### Notes:

 $f_r$ : Filling ratio of liquid cargo tanks, as given by the following formula:

 $f_r = h_{lc}/h_{tk}$ 

 $h_{lc}$ : Liquid height under consideration (m)  $h_{tk}$ : Maximum tank height (m) Table 4.8.2-11 Pitch Period and Angular Acceleration to be Considered for Sloshing

$\underline{\text{Period}}(s)$	Pitch angular acceleration (rad/s²)	
$T_{\phi\_slh} = 2\pi \sqrt{\frac{L_C B T_{LC} C_{B\_LC} K_{yy}^2 + A_{\phi}}{g L_c^3 B (2.2 C_{W\_LC}^2 - 1.8 C_{W\_LC} + 0.6)/12}}$	$a_{5\_slh} = \phi_{slh60} \left(\frac{2\pi}{T_{\phi_{slh}}}\right)^2$	

Notes:

Ballast condition is considered.

 $K_{yy}$ : Radius of gyration (m) around Y-axis, to be taken as:

 $K_{yy} = 0.25 L_c$ 

 $A_{\phi}$ : Added moment of inertia of pitch, to be taken as:

$$A_{\phi} = \frac{\pi L_{C}^{3} B^{2} C_{W\_LC}^{2}}{48 (3 - 2C_{W\_LC}) (3 - C_{W\_LC})} \left( -1.8 \frac{T_{LC}}{L_{C}} + 0.835 \right)$$

 $\phi_{slh60}$ : Maximum pitch angle (rad) under wave angle of 60 deg, to be taken as:

 $\phi_{slh60} = (0.037T_{LC}^{0.91} + 0.11)\phi$ 

 $\phi$ : Pitch angle (rad), as specified in Table 4.2.2-2

Table 4.8.2-12 Equivalent Tank Length and Equivalent Tank Breadth

	$\ell_e$ and $b_e$
Equivalent tank length	$\ell_e = \frac{(1 + n_{WT}  \alpha_{WT}) (1 + f_{wf}  \alpha_{wf})}{(1 + n_{WT}) (1 + f_{wf})} \ell_{tk-h}$
Equivalent tank breadth	$b_{e} = \frac{(1 + n_{WL} \alpha_{WL}) (1 + f_{grd} \alpha_{grd})}{(1 + n_{WL}) (1 + f_{grd})} b_{tk-h}$

Notes:

n<sub>wT</sub>: Number of transverse wash bulkheads in tanks under consideration

n<sub>WL</sub>: Number of longitudinal wash bulkheads in tanks under consideration

 $\alpha_{WT}$ : Coefficient related to transverse wash bulkheads, to be taken as (1) (See Fig. 4.8.2-4):

$$\underline{\alpha_{WT}} = \frac{A_{OWT}}{A_{tk-t-h}}$$

 $A_{OWT}$ : Total area of opening ( $m^2$ ) in transverse section in way of the wash bulkhead below the liquid height under consideration  $h_{lc}$ 

 $\underline{A_{tk-t-h}}$ : Total transverse cross sectional area of the tank  $(m^2)$  below the liquid height under consideration  $h_{lc}$   $\underline{\alpha_{WL}}$ : Coefficient related to longitudinal wash bulkheads, to be taken as<sup>(2)</sup>:

$$\underline{\alpha_{WL}} = \frac{A_{OWL}}{A_{tk-l-h}}$$

 $A_{OWL}$ : Total area of opening  $(m^2)$  in longitudinal section in way of the wash bulkhead below the liquid height under consideration  $h_{lc}$ 

 $A_{tk-l-h}$ : Total longitudinal cross sectional area of the tank  $(m^2)$  below the liquid height under consideration  $h_{lc}$   $\alpha_{wf}$ : Coefficient related to transverse girders, to be taken as<sup>(3)</sup> (See Fig. 4.8.2-4):

$$\underline{\alpha_{wf}} = \frac{A_{O-wf-h}}{A_{tk-t-h}}$$

 $A_{Q-wf-h}$ : Total area of opening  $(m^2)$  in the transverse section in way of girders below the liquid height under consideration  $h_{lc}$ 

 $\alpha_{ard}$ : Coefficient related to longitudinal girders, to be taken as<sup>(4)</sup>:

$$\underline{\alpha_{grd}} = \frac{A_{O-grd-h}}{A_{tk-l-h}}$$

 $A_{Q-grd-h}$ : Total area of opening  $(m^2)$  in the longitudinal section in way of girders below the liquid height under consideration  $h_{IC}$ 

 $f_{wf}$ : Coefficient to consider the number of transverse girders and wash bulkheads in the tank, to be taken as:

$$f_{wf} = \frac{n_{wf}}{1 + n_{WT}}$$

<u>n<sub>wf</sub></u>: Number of transverse girders, excluding wash bulkheads, in the tank

fgrd: Coefficient to consider the number of longitudinal girders and wash bulkheads in tanks, to be taken as:

$$\underline{f_{grd}} = \frac{n_{grd}}{1 + n_{WL}}$$

nard: Number of longitudinal girders, excluding wash bulkheads, in the tank

 $\ell_{tk-h}$ : Maximum value of tank length at the liquid height  $h_{lc}$  (m)

 $b_{tk-h}$ : Maximum value of tank breadth at the liquid height  $h_{lc}$  (m)

(1) For tanks whose shape changes along their length and/or with transverse wash bulkheads of different shapes,  $\alpha_{WT}$  is to be taken as the average of all transverse wash bulkheads in the tank, as given by the following formula:

$$\underline{\alpha_{WT}} = \frac{\sum_{i=1}^{n_{WT}} \frac{A_{OWT_i}}{A_{tk-t-h_i}}}{n_{WT}}$$

(2) For tanks whose shape changes along their breadth and/or with longitudinal wash bulkheads of different shapes,  $\alpha_{WL}$  is to be taken as the average of all longitudinal wash bulkheads in the tank, as given by the following formula:

$$\underline{\alpha_{WL}} = \frac{\sum_{i=1}^{n_{WL}} \frac{A_{OWL_i}}{A_{tk-L-h_i}}}{n_{WL}}$$

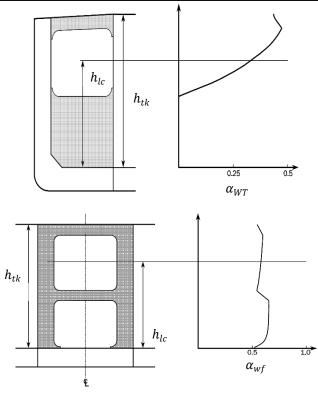
(3) For tanks whose shape changes along their length and/or with transverse girders of different shapes,  $\alpha_{wf}$  is to be taken as the average of all transverse girders in the tank, as given by the following formula:

$$\underline{\alpha_{wf}} = \frac{\sum_{i=1}^{n_{wf}} \frac{A_{O-wf-h_i}}{A_{tk-t-h_i}}}{n_{wf}}$$

(4) For tanks whose shape changes shape along their breadth and/or with longitudinal girders of different shape,  $\alpha_{grd}$  is to be taken as the average of all longitudinal girders in the tank, as given by the following formula:

$$\alpha_{grd} = \frac{\sum_{i=1}^{n_{grd}} \frac{A_{O-wf-h_i}}{A_{tk-t-h_i}}}{2}$$

Fig. 4.8.2-4 Coefficients for Wash Bulkheads and Girders



- Impact pressure caused by liquid cargo impacting tank boundaries and internal structures with high velocity is to be treated as sloshing loads. In this 4.8.2.4, the impact pressure is replaced with an equivalent pressure for plates and replaced with an equivalent bending moment for stiffeners. For tanks deemed necessary by the Society, consideration of loads may be required to be based on advanced methods such as numerical analysis or model tests.
- 4 Sloshing loads to be considered for plate panels are to be in accordance with the following (1) and (2).
- Equivalent pressures  $P_{slh-p}$  ( $kN/m^2$ ) obtained in accordance with **Table 4.8.2-13** are to be considered as sloshing loads due to pitch.
- (2) Equivalent pressures  $P_{slh-r}$  ( $kN/m^2$ ) obtained in accordance with **Table 4.8.2-14** are to be considered as sloshing loads due to roll.

Table 4.8.2-13 Equivalent Pressure for Plate Panels and Sloshing Loads Due to Pitch

Relevant ship motion	Equivalent pressure $(kN/m^2)$
<u>Pitch</u>	$P_{slh-p} = \frac{F_{slh-p}}{C_{slh1} \cdot \min(1000, C_{slh2})} \cdot 10^{6}$

#### Notes:

C<sub>slh1</sub>, C<sub>slh2</sub>: Coefficients related to member and panel length depending on the type of stiffened system, to be taken as:

 $C_{slh1} = b$ ,  $C_{slh2} = a$  for plate panels of stiffened system A

 $C_{slh1} = a$ ,  $C_{slh2} = b$  for plate panels of stiffened system B

 $C_{slh1} = b_f$  or  $b_w$ ,  $C_{slh2} = l$  for vertically corrugated bulkheads

Stiffened system A <sup>(1)</sup>: Transverse bulkheads, transverse wash bulkheads, front and aft walls of tanks with vertically stiffened systems; vertical girders of vertically stiffened systems attached to longitudinal bulkheads or tank side walls; tank top plates of longitudinally stiffened systems; horizontal girders stiffened in parallel to depth direction of webs which are attached to transverse bulkheads or transverse wash bulkheads or front and aft walls of tanks

Stiffened system B<sup>(2)</sup>: Transverse bulkheads, transverse wash bulkheads, front and aft walls of tanks with horizontally stiffened systems; vertical girders of horizontally stiffened systems attached to longitudinal bulkheads or tank side walls; tank top plates of transverse stiffened systems; horizontal girders in perpendicular to depth direction of webs which are attached to transverse bulkheads or transverse wash bulkheads or front and aft walls of tanks; cross-ties in transverse direction

a: Length (mm) of the longer side of the plate panel

b: Length (mm) of the shorter side of the plate panel

 $b_f$ ,  $b_w$ : Width (mm) of the flange and web of corrugated bulkheads respectively, as specified in 10.9.2.1

θ: Angle (rad) of corrugated bulkheads, as specified in 10.9.2.1

l: Height (mm) of corrugated bulkheads, as specified in 7.2.7.3

 $F_{slh-p}$ : Equivalent impact force (kN), to be taken as:

$$F_{slh-p} = \rho_L \cdot C_{slh1} \cdot \ell_{tk}^{1.5} \cdot C_d \cdot C_{SS} \cdot \alpha_{5\_slh} \cdot C_{slh3} \cdot 10^{-3}$$

 $\rho_I$ : Maximum design cargo density ( $t/m^3$ ) in considered  $h_{Ic}$ . Table 4.4.2-6 may be applied correspondingly.

 $\ell_{tk}$ : Maximum tank length (m)

 $\underline{C_d}$ : Coefficient depending on aspect ratio of the tank, as given by the following formula:

$$C_d = 0.65 + 0.35 \tanh\left(4 - \frac{1.5\ell_{tk}}{h_{tk}}\right)$$

 $h_{tk}$ : Maximum tank height (m)

 $C_{SS}$ : Coefficient, as given by the following formula:

$$C_{SS} = \min\left(0.3 + \frac{L_C}{325}, 1.0\right)$$

 $\underline{a_{5\_slh}}$ : Pitch angular acceleration ( $rad/s^2$ ), as specified in Table 4.8.2-11. The parameters for the ballast condition are to be used.

 $\underline{C_{slh3}}$ : Coefficient related to members under consideration and the distance from the centre of gravity of the ship to the tank, to be taken as:

 $C_{SIh3} = C_{h1}(0.0104|x_{TG} - x_G| + 1.0)$ 

 $C_{h1}$ : Parameter depending on  $h_{lc}$ , as specified in Table 4.8.2-15.

 $x_{TG}$ : X coordinate (m) at the volumetric centre of gravity of the tank under consideration

 $x_G$ : X coordinate (m) at the centre of gravity of the ship, to be taken as  $x_G = 0.45L_C$ . Where deemed appropriate by the Society, the value may be defined by the designer.

(1) See Fig. 10.9.3-1

(2) See Fig. 10.9.3-2

Table 4.8.2-14 Equivalent Pressure for Plate Panels, Sloshing Load Due to Roll

Relevant ship motion	Equivalent pressure (kN/m²)
<u>Roll</u>	$P_{Slh-r} = \frac{F_{Slh-r}}{C_{Slh1} \cdot \min(1000, C_{Slh2})} \cdot 10^{6}$

#### Notes:

Csth1: Coefficients related to member and panel length depending on the type of stiffened system, to be taken as:

 $C_{slh1} = b$ ,  $C_{slh2} = a$  for plate panels of stiffened system A

 $C_{slh1} = a$ ,  $C_{slh2} = b$  for plate panels of stiffened system B

 $C_{slh1} = b_f$  or  $b_w$ ,  $C_{slh2} = l$  for vertically corrugated bulkheads

Stiffened system A <sup>(1)</sup>: Longitudinal bulkheads, longitudinal wash bulkheads, tank side walls with vertically stiffened systems; vertical girders of vertically stiffened systems attached to transverse bulkheads or front and aft walls of tanks; tank top plates of transverse stiffened systems; horizontal girders stiffened in parallel to depth direction of webs which are attached to longitudinal bulkheads or longitudinal wash bulkheads or front and aft walls of tanks

Stiffened system B<sup>(2)</sup>: Longitudinal bulkheads, longitudinal wash bulkheads, front and aft walls of tanks with longitudinally stiffened systems; vertical girders of horizontally stiffened systems attached to transverse bulkheads or front and aft walls of tanks; tank top plates of longitudinally stiffened systems; horizontal girders stiffened in perpendicular to depth direction of webs attached to longitudinal bulkheads or longitudinal wash bulkheads or tank side walls; cross-ties in longitudinal direction

 $a, b, b_f, b_w, \theta, l$ : As specified in Table 4.8.2-13

 $F_{slh-r}$ : Equivalent impact force (kN), to be taken as:

 $F_{slh-r} = \rho_L \cdot C_{slh1} \cdot b_{tk}^{1.5} \cdot a_4 \cdot C_{slh3} \cdot 10^{-3}$ 

 $\rho_L$ : As specified in Table 4.8.2-13

 $b_{tk}$ : Maximum tank breadth (m)

 $a_4$ : Roll angular acceleration ( $rad/s^2$ ), as specified in 4.2.3.4. The parameters for the ballast condition are to be used.

 $C_{slh3}$ : Coefficient related to members under consideration, to be taken as:

 $C_{clh3} = C_{h1}$ 

 $C_{h1}$ : Parameter depending on  $h_{lc}$ , as specified in Table 4.8.2-15

(1) See Fig.10.9.3-1

(2) See Fig.10.9.3-2

<u>Table 4.8.2-15</u> Parameters for Sloshing Loads

	<u>C<sub>h1</sub></u>				
Member to be assessed	Low filling ratio $0.2 \le f_r < 0.4$	Middle filling ratio $0.4 \le f_r < 0.7$	High filling ratio $0.7 \le f_r \le 0.9$		
Front and aft walls / side walls of tanks     Transverse bulkheads / longitudinal bulkheads     (including corrugated bulkheads)	8.63	16.1	22.3		
<ul><li>Transverse wash bulkheads</li><li>Longitudinal wash bulkheads</li></ul>	3.23	<u>4.61</u>	4.22		
- Tank top plates <sup>(1)</sup>	<u>1.18</u>	<u>11.0</u>	<u>8.63</u>		
<ul> <li>Vertical girders attached to front and aft walls / side walls of tanks</li> <li>Vertical girders attached to longitudinal bulkheads / transverse bulkheads</li> </ul>	3.63	6.28	4.80		
<ul> <li>Horizontal girders attached to front and aft walls / side walls of tanks</li> <li>Horizontal girders attached to longitudinal bulkheads / transverse bulkheads</li> </ul>	For $h_{hg} \le 0.5$ , $3.14h_{hg} + 0.68$ For $h_{hg} > 0.5$ , $-1.37h_{hg} + 2.935$	For $h_{hg} \le 0.5$ , $1.57h_{hg} + 0.20$ For $h_{hg} > 0.5$ , $-0.39h_{hg} + 1.18$	$0.88h_{hg}$ +0.10		
- Cross-ties	<u>3.24</u>	4.61	4.22		
- Sloping plates above side walls <sup>(2)</sup>	<u>NA</u>	$\alpha = 0$ :11.0 $\alpha = 30$ :1.97 $\alpha = 90$ :16.0	$\alpha = 0$ :8.63 $\alpha = 30$ :3.92 $\alpha = 90$ :22.3		
- Sloping plates below side walls <sup>(2)</sup>	$\alpha = 0$ :5.89 $\alpha = 30$ :5.89 $\alpha = 90$ :8.63	$\alpha = 0$ :4.91 $\alpha = 30$ :4.91 $\alpha = 90$ :16.1	<u>N4</u>		
Notes:					

- f<sub>r</sub>: Filling ratio of liquid cargo tank, as specified in Table 4.8.2-10.
- α: Acute angle (deg) of inclination angle to the horizontal plane of the panel under consideration

 $h_{hg}$ : Ratio of the distance (m) from tank bottom plate to the horizontal girders under consideration to maximum tank height  $h_{tk}$  (m)

- (1) The parameters are for the plate panels within the range of  $0.3\ell_{tk}$  from transverse bulkheads / front and aft walls of tank and within the range of  $0.3b_{tk}$  from longitudinal bulkheads / tank side walls. Definitions of  $\ell_{tk}$  and  $b_{tk}$  are specified in Table 4.8.2-13 and Table 4.8.2-14.
- (2) Intermediate values of  $\alpha$  are to be obtained by linear interpolation.
- 5 Sloshing loads to be considered for stiffeners are to be in accordance with the following (1) and (2).
- (1) Where -4 above is applied and the stiffeners are attached to the plate panels with  $C_{slh1} = b$  and  $C_{slh2} = a$  (stiffeners in parallel to the direction a), the equivalent bending moments specified in the following formulae  $M_{slh-p}$  and  $M_{slh-r}$  (kN-m) are to be considered.

 $M_{slh-n} = F_{slh-n} \ell_{slh}$  for sloshing loads due to pitch

 $M_{slh-r} = F_{slh-r} \ell_{slh}$  for sloshing loads due to roll

 $F_{slh-p}$ ,  $F_{slh-r}$ : As specified in -4 above

 $\ell_{slh}$ : Equivalent lever (m), to be taken as:

 $\ell_{slh} = f_{bd}\ell_{bda}$ 

 $f_{bd}$ : Coefficient considering boundary conditions, as specified in Table 4.8.2-16

 $\ell_{bdg}$ : Effective bending span (m) of stiffeners, as specified in 3.6.1.2

(2) Where -4 above is applied and the stiffeners are attached to the plate panels with  $C_{sth1} = a$ 

and  $C_{slh2} = b$  (stiffeners in parallel to the direction a), the equivalent bending moments specified in the following formulae  $M_{slh-p}$  and  $M_{slh-r}$  (kN-m) are to be considered.

 $M_{slh-p} = 0.083 \cdot F_{slh-p} \cdot \ell_{bdg}$  for sloshing loads due to pitch

 $M_{slh-r} = 0.083 \cdot F_{slh-r} \cdot \ell_{bda}$  for sloshing loads due to roll

 $F_{slh-p}$ ,  $F_{slh-r}$ : As specified in -4 above

 $\ell_{bdg}$ : As specified in (1) above

Table 4.8.2-16 Coefficient Considering Boundary Conditions

Table 4.6.2-10 Coefficient Considering Boundar	<u>j conditions</u>
<u>Member</u>	$f_{bd}$
- Front and aft walls / side walls of cargo tanks including	
corrugated walls	
- Transverse bulkheads / longitudinal bulkheads (including	0.31
corrugated bulkheads)	
- Tank top plates	
- Sloping plates above and below side walls	
- Transverse wash bulkhead, longitudinal wash bulkhead	
- Vertical girders attached to front and aft walls of tanks /	
transverse bulkheads	<u>0.43</u>
- Vertical girders attached to tank side walls / longitudinal	
<u>bulkheads</u>	
- Horizontal girders attached to front and aft walls of tanks /	
transverse bulkheads	1.70
- Horizontal girders attached to side walls of tanks / longitudinal	1.70
<u>bulkheads</u>	
- Cross-ties	0.39

6 For vertically corrugated bulkheads, in addition to the requirements of -4 above, the equivalent bending moments  $M_{slh-p}$  and  $M_{slh-r}$  obtained from the following formulae are to be considered as loads (kN-m) for obtaining the section modulus.

 $M_{slh-n} = F_{slh-n} \ell_{slh}$  for sloshing loads due to pitch

 $\underline{M_{slh-r}} = F_{slh-r}\ell_{slh}$  for sloshing loads due to pitch

 $F_{slh-p}$ ,  $F_{slh-r}$ : As specified in -5 above. The value of  $C_{slh1}$  is to be the value of 1/2 pitch (mm) specified in 7.2.7.2.

 $\ell_{slh}$ : Equivalent lever (m), to be taken as:

 $\ell_{slh} = f_{hd}\ell$ 

f<sub>hd</sub>: Coefficient considering boundary conditions, as specified in Table 4.8.2-16

 $\ell$ : Bending span (m) of corrugated bulkheads, as specified in 7.2.7.3

7 Hull girder load (*kN-m*) to be considered in longitudinal members is to be in accordance with absolute value of the following formulae, whichever is greater.

 $M_{V-HG} = M_{SV max} + C_{slh-V}M_{WV-h}$ 

 $M_{V-HG} = M_{SV min} + C_{slh-V}M_{WV-s}$ 

<u>M<sub>SV\_max</sub></u>: Permissible maximum vertical still water bending moment (kN-m) specified in 4.3.2.2

 $\underline{M_{SV\_min}}$ : Permissible minimum vertical still water bending moment (kN-m) specified in 4.3.2.2

 $\underline{M_{WV-h}}$ : Vertical wave bending moment (kN-m) in the hogging condition, to be taken as:  $\underline{M_{WV-h}} = 0.19C_1C_2L_C^2BC_{B1}$ 

 $\underline{M_{WV-s}}$ : Vertical wave bending moment (kN-m) in the sagging condition, to be taken as:  $\underline{M_{WV-s}} = -0.11C_1C_2L_C^2B(C_{B1} + 0.7)$ 

 $C_2$ : As specified in **Table 4.4.2-14**. Intermediate vales are to be linear interpolation.  $C_{slh-V}$ : 0.5 for sloshing loads due to pitch, 0.2 for sloshing loads due to roll

# Chapter 6 LOCAL STRENGTH

#### 6.1 General

Title of Paragraph 6.1.3 has been amended as follows.

# 6.1.3 <u>Net Scantling Approach</u>

Paragraph 6.1.3.1 has been amended as follows.

#### **6.1.3.1** General

The required scantlings specified in this chapter are net scantlings, unless otherwise specified.

# 6.2 Design Load Scenarios and Loads of the Ship to Be Assessed

#### 6.2.1 General

#### 6.2.1.1

- 1 Unless otherwise specified, a strength assessment is to be carried out for the maximum load condition, the testing condition and flooded condition.
- 2 For longitudinal hull girder structural members, hull girder loads due to longitudinal bending are to be considered in addition to lateral loads on plates and stiffeners.
- 3 Lateral loads are, in principle, assumed to act on one side of plates and stiffeners. However, where any loads are constantly also acting on the other side, such loads may be taken into account.
- 4 Plates constituting boundaries of watertight compartments not intended to carry liquids and stiffeners supporting such plates, excluding shell plating, stiffeners attached to shell plating, weather deck plating and stiffeners attached to weather deck plating, are to be subjected to lateral loads under flooded conditions.

# 6.2.2 Assessment Design Load Scenarios and Loads for Members to Be Assessed

#### 6.2.2.1

For the plates constituting the boundaries of compartments and the stiffeners supporting such plates listed in **Table 6.2.2-1**, the strength assessment specified in this chapter is to be carried out considering the lateral loads and hull girder loads specified in the table. For members or compartments corresponding to multiple conditions, the strength assessment is to be carried out for all applicable loads.

#### Table 6.2.2-1 has been amended as follows.

Table 6.2.2-1 Assessment Design Load Scenarios and Loads for Members or Compartments to Be Assessed

		Load					
Compartments <u>or</u>	Design load scenario	Lateral		Load	Refer to the	he following:	
		load	Load type	component	Lateral load (P)	Hull girder load $(M_{V-HG}, M_{H-HG})$	
Outer shell (including stiffeners)		External pressure	Seawater	Static + dynamic loads	4.4.2.2-1		
Cargo tanks, ballast tanks, ballast holds and other tanks			Internal	Liquid loaded	Static + dynamic loads	4.4.2.2-2	
Cargo holds <sup>(1)</sup>	Maximum load conditions	pressure	Dry bulk cargoes	Static + dynamic loads		4.4.2.9	
Cargo holds <sup>(2)</sup>	conditions		Others	Static + dynamic loads			
Weather decks (including stiffeners)		Others	Green sea, unspecified loads	Green sea load, static + dynamic loads	Greater of the pressures specified in 4.4.2.2-3 and -4		
Internal decks <sup>(2)</sup> (including stiffeners)			Cargoes	Static + dynamic loads	4.4.2.2-3		
Members constituting compartments subject to hydrostatic testing	Testing condition	Internal pressure	Seawater	Static loads	4.4.3.2	4.4.3.3	
Compartments not carrying liquids <sup>(3)</sup> Transverse and longitudinal bulkheads	Flooded conditions	Internal pressure	Seawater	-	4.4.4.1	4.4.4.2	

#### (Notes)

- (1) For ships of single-side skin construction intended for carrying cargoes other than liquids, the outer shell (including stiffeners) may be excluded from the assessment.
- (2) For ships carrying cargoes other than bulk and liquid cargoes with the cargoes properly fastened or otherwise held in position so that the cargo loads can be deemed as acting only on the inner bottom plating and internal deck, the assessment need be performed only for the inner bottom plating and the internal deck.
- (3) Not required to be applied to shell plating, stiffeners attached to shell plating, weather deck plating and stiffeners attached to weather deck plating.

#### 6.3 Plates

#### **6.3.2** Plates

Paragraph 6.3.2.1 has been amended as follows.

# **6.3.2.1** Bending Strength

Plate thickness is to be not less than the largest of the values obtained by the following formula under all applicable design load scenarios specified in Table 6.2.2-1. Application of gross or net scantlings in the values obtained from the following is specified in Table 6.3.2-1:

$$t = C_{Safety}C_{Aspect}\sqrt{\frac{4}{1.15C_a\sigma_Y}}\sqrt{\frac{|P|b^2}{f_P}}\times 10^{-3}(mm)$$

 $\sigma_{\rm Y}$ : Specified minimum yield stress (N/mm<sup>2</sup>)

b: Length (mm) of the shorter side of the plate panel

a: Length (mm) of the longer side of the plate panel

 $\alpha$ : Aspect ratio to be taken as a/b.

*f<sub>P</sub>*: Strength coefficient as given in **Table 6.3.2-1** for eases in the maximum load condition and the testing condition or **Table 6.3.2-2** for eases in flooded conditions.

P: Lateral pressure  $(kN/m^2)$  corresponding to each Design load scenario specified in Table 6.2.2-16.3.2-1, to be calculated at the load calculation point specified in 3.7.

 $C_a$ : Coefficient of axial force effect as specified in Table 6.3.2-32 when  $\alpha \ge 2$  or Table 6.3.2-43 when  $\alpha < 2$ .

Correction coefficient for the aspect ratio of the plate panel as given in **Table**6.3.2-1 for eases in the maximum load condition and the testing condition or

Table 6.3.2-2 for eases in flooded conditions.

 $C_{Safety}$ : Safety factor taken as 1.0.

 $\sigma_{BM}$ : Axial stress (N/mm<sup>2</sup>) due to hull girder bending as specified in **6.2.3.1**.

Table 6.3.2-1 has been amended as follows.

Table 6.3.2-1 Definitions of C<sub>Aspect</sub> and f<sub>p</sub> for Assessment in Maximum Load Condition and Testing Condition

	8	
<del>Member</del>	C <sub>aspect</sub>	<del>f</del> <del>f</del> ∉
Longitudinal hull girder structural members	1.0	<del>12</del>
Other members	$\frac{1.07 - 0.28 \left(\frac{b}{a}\right)^{\frac{2}{a}}}{\text{but } 1.0 \text{ for } \alpha > 2}$	<del>12</del>

Table 6.3.2-1 Application of Gross or Net Scantlings and Each Parameter in the Evaluation for Each Design Load Scenario

Application of Lateral load  $C_{Aspect}$ Design load scenario Member  $f_{P}$ gross or net  $P(kN/m^2)$ scantlings Longitudinal hull  $P_{ex}$ ,  $P_{in}$ ,  $P_{dk}$  and  $P_{GW}$ girder structural 1.0 To be in accordance with members Maximum load condition Net scantling **4.4.2.2-1** to **-4** corresponding 12  $1.07 - 0.28 \left(\frac{b}{a}\right)^2$ to compartments/members to Other members be assessed in Table 6.2.2-1 but 1.0 for  $\alpha > 2$ Longitudinal hull girder structural 1.0  $P_{ST-in1}$ members Gross To be in accordance with Case 1 12 scantling 4.4.3.2 Other members but 1.0 for  $\alpha > 2$ Testing Longitudinal hull condition girder structural 1.0 members  $P_{ST-in2}$ To be in accordance with Net scantling 16 Case 2 4.4.3.2 Other members Longitudinal hull girder structural 1.0  $P_{FD-in}$ members Flooded condition Net scantling To be in accordance with 16 4.4.4.1 Other members

Table 6.3.2-2 has been deleted, Table 6.3.2-3 has been renumbered to Table 6.3.2-2.

Table 6.3.2-2 Definitions of C<sub>Acroset</sub> and f<sub>B</sub> for Assessment in Flooded Conditions

Member	<del>C<sub>Aspect</sub></del>	<del>∫</del> #
Longitudinal hull girder structural members	<del>1.0</del>	<del>16</del>
Other members	$\frac{1}{\sqrt{1+\left(\frac{b}{a}\right)^2}}$	<del>16</del>

Table 6.3.2- $\frac{3}{2}$  Definition of  $C_a$  (for  $\alpha \ge 2$ )

Memb	$C_a$	
Longitudinal hull	Longitudinal framing system	$\sqrt{1-\left(rac{\sigma_{BM}}{\sigma_{Y}} ight)^{2}}$
girder structural members	Transverse framing system	$1.0 - \frac{ \sigma_{BM} }{\sigma_{Y}}$
Other members		1.0

Table 6.3.2-4 has been renumbered to Table 6.3.2-3.

Table 6.3.2-43 Definition of  $C_a$  (for  $\alpha < 2$ )

Member		$C_a$	ζ	η
Longitudinal hull	Longitudinal framing system	$\left[ \left( \left  \sigma_{BM} \right  \right ^{\zeta} \right]^{\eta}$	2	$\frac{b}{a}$
girder structural members	Transverse framing system	$\left[1-\left(\frac{ \partial_{BM1} }{\sigma_{Y}}\right)\right]$	$2\frac{b}{a}$	1
Other members			1.0	

Paragraph 6.3.3 has been amended as follows.

#### 6.3.3 Longitudinal and Transverse Bulkheads Corrugated Bulkheads

#### 6.3.3.1 Plane Bulkheads

The thickness of plane bulkheads is to be in accordance with the requirements in 6.3.2.1.

# **6.3.3.21** Thickness of Corrugated Bulkheads

The thickness of the flanges and webs of corrugated bulkheads in the maximum load condition and the testing condition under all applicable design load scenarios specified in Table 6.2.2-1 is to be the largest of the values obtained by the following formula. Application of gross or net scantlings in the values obtained from following formula is specified in Table 6.3.3-1:

$$t = C_{Safety} \sqrt{\frac{4}{1.15\sigma_Y}} \sqrt{\frac{|P| \frac{b_\#}{b_\#} \underline{b}^2 \gamma}{\frac{12}{2} f_P}} \times 10^{-3} (mm)$$

C<sub>Safety</sub>: Safety factor taken as 1.0 as specified in Table 6.3.3-1.

 $\sigma_{\rm Y}$ : Specified minimum yield stress (N/mm<sup>2</sup>)

P: Lateral pressure  $(kN/m^2)$  corresponding to each design load scenario specified in Table

6.2.2-1 Table 6.3.3-1, to be calculated at the load calculation point specified in 3.7.

- <u>b</u>: Width (mm) of the flange (face plate) and web, respectively, to be taken as  $b_f$  or  $b_w$  (mm) in Fig. 6.3.3-1.
- γ: Coefficient taken as follows: as specified in Table 6.3.3-1.

$$\frac{\alpha + \beta^{\frac{2}{3}}}{\alpha + \beta}$$
where  $\alpha = \frac{t_{\overline{w}}^{\frac{3}{3}}}{t_{\overline{x}}^{\frac{2}{3}}}$ ,  $\beta = \frac{b_{\overline{w}}}{b_{\overline{x}}}$ 

 $t_{\perp}$  and  $t_{\perp}$ : Thickness (mm) of the flange and web, respectively.

 $b_{\pm}$  and  $b_{\pm}$ : Width (mm) of the flange and web, respectively (See Fig. 6.3.3-1).

 $f_p$ : Strength coefficient given in Table 6.3.3-1.

2 The thickness of the flange and web of corrugated bulkheads in flooded conditions is to be not less than the value obtained by the following formula:

$$t = C_{\text{safety}} \sqrt{\frac{4}{1.15\sigma_{\text{F}}}} \sqrt{\frac{f_{\text{F}}}{f_{\text{F}}}} \times 10^{-3} \text{ (mm)}$$

 $\sigma_{\underline{\mathbf{v}}}$ : Specified minimum yield stress (N/mm<sup>2</sup>)

P: Lateral pressure (kN/m²) in the flooded conditions specified in Table 6.2.2-1, to be calculated at the load calculation point specified in 3.7.

b: Width (mm) of the flange (face plate) and web, respectively, to be taken as  $b_{\neq}$  or  $b_{\neq}$  (mm) in Fig. 6.3.3-1.

f<sub>□</sub>: Strength coefficient given in Table 6.3.3-1.

Csarety: Safety factor given in Table 6.3.3-2.

 $t_{\neq}$  and  $t_{w}$ : Thickness (mm) of the flange (face plate) and web, respectively

32 Notwithstanding -1 and -2 above, horizontal corrugated bulkheads are to be as deemed appropriate by the Society.

Fig. 6.3.3-1 Measurement Method of b

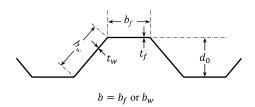


Table 6.3.3-1 has been amended as follows.

Table 6.3.3-1 Value of  $f_{\mu}$ Flooded condition

Flange

Web

16

Table 6.3.3-1 Application of Gross or Net Scantlings and Each Parameter in the Evaluation for Each Design Load Scenario

Design load scenario gro		Application of gross or net scantlings	Lateral load P (kN/m²)	¥	<u>Member</u>	$\underline{C_{Safety}}$	$f_{P}$
<u>Maximum load</u>	condition	Net scantling	$P_{ex}$ , $P_{in}$ , $P_{dk}$ and $P_{GW}$ To be in accordance with  4.4.2.2-1 to -4  corresponding to  compartments/members to  be assessed in Table 6.2.2-1	$\frac{\alpha + \beta^3}{\alpha + \beta}$	Flange and Web	<u>1.0</u>	<u>12</u>
<u>Testing</u>	$\frac{\text{Case1}}{\text{Scantling}} \frac{\frac{P_{ST-in1}}{\text{To be in accordance with}}}{\frac{4.4.3.2}{\text{Case1}}}$	To be in accordance with	$\frac{\alpha + \beta^3}{\overline{\alpha + \beta}}$	Flange and Web	<u>1.0</u>	<u>12</u>	
<u>condition</u>	Case 2	Net scantling $\frac{P_{ST-in2}}{\text{To be in accordance with}}$ $\frac{1.0}{4.4.3.2}$		1.0	Flange	<u>1.15</u>	$8\left[1+\left(\frac{t_w}{t_f}\right)^2\right]$
	<u>Case 2</u>		1.0	Web	<u>1.07</u>	<u>16</u>	
Flooded condition Net scar			P <sub>FD</sub> in	1.0	<u>Flange</u>	<u>1.15</u>	$8\left[1+\left(\frac{t_w}{t_f}\right)^2\right]$
		Net scantling	To be in accordance with 4.4.4.1	<u>1.0</u>	<u>Web</u>	<u>1.07</u>	<u>16</u>

Notes:

$$\alpha = \frac{t_w^3}{t_f^3}$$
,  $\beta = \frac{b_w}{b_f}$ 

 $t_f$  and  $t_w$ : Thickness (mm) of the flange and web, respectively

 $b_f$  and  $b_w$ : Width (mm) of the flange and web, respectively (See Fig. 6.3.3-1)

Table 6.3.3-2 has been deleted.

Table 6.3.3-2 Value of Cearant

Flange -	<del>1.15</del>
Web	<del>1.07</del>

#### 6.4 Stiffeners

#### 6.4.1 General

# 6.4.1.1 Application

Sub-paragraph -2 has been amended as follows.

- 1 Stiffeners subject to lateral loads are to be in accordance with the requirements in **6.4.2**.
- 2 Notwithstanding -1 above, the side frames within the cargo region are to be in accordance with 6.4.3.

#### 6.4.2 Stiffeners

Paragraph 6.4.2.1 has been amended as follows.

# **6.4.2.1** Bending Strength

For all applicable design load scenarios specified in **Table 6.2.2-1**, the section modulus of stiffeners is to be not less than the value obtained from the following formulae. <u>Application of gross or net scantlings in the values obtained from following formula is specified in **Table 6.4.2-5**:</u>

Table 6.4.2-1 has been amended as follows.

Table 6.4.2-1 Required Section Modulus for Stiffeners Subject to Lateral Pressure

Tuore	5. 1.2 1 Ite	quired Section Woulding for Stiffeners Subject to Euteral Pressure	
Design	Design load scenario Required value		
Maximum load condition Testing condition		$Z = \frac{C_{Safety}C_{VB}f_{bdg} P s\ell_{bdg}^{2}}{12f_{f}C_{s}\sigma_{Y}} (cm^{3})$	
Flooded conditions		$Z = \frac{C_{Safety} f_{bdg-P}  P  s \ell_{bdg}^{2}}{16 f C_{s} \sigma_{Y}} (cm^{3})$	
$\ell_{bdg}$ :	Effective bending	g span (m) as specified in 3.6.1.2.	
s:	Spacing (mm) of	stiffeners	
$f_{bdg}$ :	Elastic bending moment distribution factor according to the type of end connection of stiffe		
	specified in Tabl	e 6.4.2-2.	
$f_{bdg-P}$ :	Plastic bending moment distribution factor according to the type of end connection of stiffener		
	specified in Table 6.4.2-3.		
$C_{VB}$ :	Coefficient taken	a as follows:	
	1.0: For stiffener	s arranged horizontally	
	1.2: For stiffener	s not arranged horizontally	
P:	Lateral pressure $(kN/m^2)$ corresponding to each Design load scenario specified in Table 6.2.2-16.4.2-5,		
	be calculated at the load calculation point specified in 3.7.		
$C_s$ :	Coefficient for axial force effect as specified in Table 6.4.2-4.		
$C_s$ : $f_f$ : $f$ : $C_{Safety}$ :	Coefficient taken as 1.25 for flat bars or 1.0 for other members.		
<i>f</i> :	Shape coefficien	t taken as 1.5 for flat bars or 1.2 for other members.	
$C_{Safety}$ :	Safety factor taken as 1.0.		

Table 6.4.2-5, Table 6.4.2-6 and Table 6.4.2-7 have been renumbered to Table 6.4.2-6, Table 6.4.2-7 and Table 6.4.2-8, and Table 6.4.2-5 has been added as follows.

Table 6.4.2-5 Application of Gross or Net Scantlings and P in the Evaluation for Each Design

Load Scenario					
Design load scenario		Application of gross or net scantlings	$\frac{\text{Lateral load}}{P \ (kN/m^2)}$		
Maximum load condition		Net scantling	P <sub>ex</sub> , P <sub>in</sub> , P <sub>dk</sub> and P <sub>GW</sub> To be in accordance with  4.4.2.2-1 to -4 corresponding to  compartments/members to be  assessed in Table 6.2.2-1		
	<u>Case 1</u>	Gross scantling	P <sub>ST-in1</sub> To be in accordance with 4.4.3.2		
Testing condition	<u>Case 2</u>	Net scantling	$\frac{P_{ST-in2}}{\text{To be in accordance with 4.4.3.2}}$		
Flooded condition		Net scantling	P <sub>FD-in</sub> To be in accordance with 4.4.4.1		

Paragraph 6.4.2.2 has been amended as follows.

# **6.4.2.2** Shear Strength of Webs

For all applicable design load scenarios specified in **Table 6.2.2-1**, the stiffener web thickness is to be not less than the value obtained from the following formula. Application of gross or net scantlings in the values obtained from following formula is specified in **Table 6.4.2-5**:

$$t_{w} = \frac{C_{Safety}C_{VS}f_{shr}|P|s\ell_{shr}}{2d_{shr}\tau_{eH}} (mm)$$

 $\tau_{eH}$ : Permissible shear stress ( $N/mm^2$ ) taken as follows:

$$\sigma_Y/\sqrt{3}$$

 $d_{shr}$ : Effective shear depth (mm) as specified in 3.6.4.2.

 $\ell_{shr}$ : Effective shear span (m) as specified in 3.6.1.3.

s: Spacing (mm) of stiffeners

 $C_{VS}$ : Coefficient taken as follows:

1.0: For stiffeners arranged horizontally

1.4: For stiffeners not arranged horizontally

 $f_{shr}$ : Shear force distribution factor according to the type of end connection of stiffeners as specified in **Table 6.4.2-56**.

 $C_{Safety}$ : Safety factor taken as 1.2.

Table 6.4.2- $\frac{5}{6}$  Definition of  $f_{shr}$ 

		73111		
One end	Other end			
	Fixed Flexibly fixed Sniped end			
Fixed	1.00	1.15	1.25	
Flexibly fixed	1.15	1.00	1.20	
Sniped end	1.25	1.20	1.00	

#### (Notes)

- (1) A fixed end connection is a structure in which a stiffener is continuous or its ends are reinforced with proper backing.
- A flexibly fixed end connection is a structure fixed to an orthogonal stiffener and not reinforced on the back with an effective supporting member, as shown in Fig. 6.4.2-1.

### **6.4.2.3** Reinforcement of Stiffeners with Struts

Sub-paragraph -2 has been amended as follows.

- 1 (Omitted)
- Where stiffeners are supported by struts provided in between the floors in a double bottom, the section modulus and web thicknesses of bottom longitudinals and inner bottom longitudinals are to be not less than the values calculated in **6.4.2.1** and **6.4.2.2** times those shown in **Table 6.4.2-67**. However, in a cargo loaded condition where  $|P_{in}| > 2|P_{ex}|$  or  $|P_{ex}| > 2|P_{in}|$ , these values are to be obtained from the formula shown in **Table 6.4.2-78** instead of **Table 6.4.2-67**.
  - $P_{in}$ : Lateral pressure  $(kN/m^2)$  acting on the inner bottom plating of a hold loaded with cargo, to be calculated as specified in any of **4.4.2.4** through **4.4.2.7** as applicable depending on the load loaded in the cargo hold. For each load condition specified in **4.4.2.1**,  $P_{in}$  is to be calculated at the inner bottom plating in a location provided with struts.
  - $P_{ex}$ : External pressure  $(kN/m^2)$  acting on the bottom shell, to be calculated as specified in **4.4.2.3**.  $P_{ex}$  is to be calculated at the bottom shell in a location provided with struts for each load condition specified in **4.4.2.1**.

# 3 (Omitted)

Table 6.4.2- $\frac{67}{2}$  Correction Factor  $C_1$  for Struts

<u> </u>							
Ratio of the moment of inertia of stiffeners			≥ 1.0	≥ 1.2	≥ 1.4	≥ 1.6	≥ 1.8
$\max(I_B, I_I)/\min(I_B, I_I)$			< 1.2	< 1.4	< 1.6	< 1.8	≥ 1.0
$\mathcal{C}_1$	Bottom longitudinals	Value for section modulus (6.4.2.1)	0.625	0.670	0.700	0.725	0.745
		Value for web thickness (6.4.2.2)	0.750	0.775	0.800	0.815	0.825
	Inner bottom longitudinals	Value for section modulus (6.4.2.1)	0.625	0.670	0.690	0.720	0.740
		Value for web thickness (6.4.2.2)	0.750	0.780	0.795	0.810	0.825

Table 6.4.2- $\frac{78}{2}$  Correction Factor  $C_2$  for Struts in Ships Where  $|P_{in}| > 2|P_{ex}|$  or  $|P_{ex}| > 2|P_{in}|$ 

C	$ P_{in}  > 2 P_{ex} $	$ P_{ex}  > 2 P_{in} $			
$C_2$	Bottom longitudinals	Inner bottom longitudinals			
Value for section modulus (6.4.2.1)	$\frac{3}{4} \frac{\lambda}{\lambda + 1} \frac{P_{in}}{P_{ex}} - C_1 + \frac{1}{2}$	$\frac{3}{4} \frac{1}{\lambda + 1} \frac{P_{ex}}{P_{in}} - C_1 + \frac{1}{2}$			
Value for web thickness (6.4.2.2)	$\frac{1}{2} \frac{\lambda}{\lambda + 1} \frac{P_{in}}{P_{ex}} - C_1 + 1$	$\frac{1}{2} \frac{1}{\lambda + 1} \frac{P_{ex}}{P_{in}} - C_1 + 1$			
(Notes)					
$C_1$ : Coefficient given in Table 6.4.2-67.					
(1) Correction Factor $C_2$ is not to be less than Correction Factor $C_1$ .					

# **Chapter 7 STRENGTH OF PRIMARY SUPPORTING STRUCTURES**

Symbols has been amended as follows.

#### **Symbols**

For symbols not defined in this Chapter, refer to 1.4.

(Omitted)

 $P_{DB}$ : Reference pPressure  $(kN/m^2)$  for the double bottom to be calculated at the load calculation point specified in 7.3.1.5, as specified in 4.4, Chapter 4, Part 2 the requirements for loads to be considered for the strength of primary supporting structures specified in Chapter 4, Part 2 (Requirements by ship type)

 $P_{DS}$ : Reference pPressure  $(kN/m^2)$  for the double side to be calculated at the load calculation point specified in 7.3.1.5, as specified in 4.4, Chapter 4, Part 2 the requirements for loads to be considered for the strength of primary supporting structures specified in Chapter 4, Part 2 (Requirements by ship type)

(Omitted)

 $C_{EX}$ : Coefficient for the ratio of the effective breadth of longitudinal girders in double hull as given in **Fig. 7.3.3-1**. The method for calculation of effective breadth is given in **3.6.3.1**. However,  $C_{EX}$  is to be taken as 0 where no longitudinal girder is provided.

 $C_{EY}$ : Coefficient for the ratio of the effective breadth of transverse girders in double bottom <u>hull</u> as given in **Fig. 7.3.3-1**. The method for calculation of the effective breadth is given in **3.6.3.1**. However,  $C_{EY}$  is to be taken as 0 where no transverse girder is provided. For double bottom, read  $C_{EY}$ . For double side,  $C_{EY}$  is to be read as  $C_{EZ}$  and the transverse girder as the longitudinal girder. (Omitted)

#### 7.1 General

#### 7.1.2 Application

Paragraph 7.1.2.1 has been amended as follows.

#### 7.1.2.1 Application

1 The requirements in this Chapter apply to ships with a length  $L_{\epsilon}$  of less than 200 m are to be applied in accordance with Table 1.2.2-1.

- 2 Notwithstanding -1 above, for ships subject to Chapter 8, the requirements in Chapter 8 are to apply instead of those in this Chapter.
- 32 Girders and plates to be flange in double bottom or double side constituting a double hull are to be in accordance with the requirements in 7.3. Other girders are to be in accordance with the requirements in 7.2. (See Fig. 7.1.2-1)
- 43 Constructions of single bottom are to be according to the requirements specified in 7.2.
- 54 Notwithstanding -32 and -43 above, assessments may be carried out by direct strength calculations deemed appropriate by the Society, such as beam analysis.

Title of Paragraph 7.1.3 has been amended as follows.

# 7.1.3 <u>Net Scantling Approach</u>

# **7.1.3.1** General

The required scantlings in this chapter are to be specified as net scantlings.

# 7.2 Simple Girders

# 7.2.2 Assessment Conditions and Loads

# 7.2.2.2 Assessment Conditions and Loads for Members to Be Assessed

Table 7.2.2-1 has been amended as follows.

Table 7.2.2-1 Assessment Conditions and Loads for Members or Compartments to Be Assessed

	Typical members	Assessment condition	Loads					
Compartments/ members to be			Lateral load	Load type	Load components	Refer to:		
assessed						Load (P)	Hull girder load ( $M_{V-HG}$ , $M_{H-HG}$ )	
Girders on shell plating	Web frames, side stringers (single side skin structure)		External pressure	Seawater	Static + dynamic loads	4.4.2.2-1		
Cargo oil tanks, ballast tanks, ballast holds and other tanks	Stiffening girders, corrugated bulkheads	Maximum load condition	Internal pressure	Liquid loaded	Static + dynamic loads	4.4.2.2-2	4.4.2.9	
Cargo holds(1)	Stiffening girders, corrugated bulkheads			Dry bulk cargoes and others	Static + dynamic loads			
Single-bottomed cargo holds	Girders, floors			Unspecified cargoes	Static + dynamic loads			
Girders on deck	Deck girders deck		Others	Green sea (weather decks only), unspecified cargoes	Green sea load, static + dynamic loads	Greater of the pressures specified in 4.4.2.2-3 and -4		
Internal decks <sup>(2)</sup>	Deck girders, deck transverses			Unspecified cargoes	Static + dynamic loads	4.4.2.2-3		
Members constituting compartments subject to hydraulic testing	Stiffening girders, corrugated bulkheads	Testing condition	Internal pressure	Seawater	Static loads	P <sub>ST-in1</sub> as specified in 4.4.3.2	4.4.4.3	
Compartments not carrying liquids <sup>(3)</sup> Transverse and longitudinal bulkheads	Stiffening girders, corrugated bulkheads	Flooded condition	Internal pressure	Seawater	-	4.4.4.1	4.4.4.2	

#### (Notes)

- (1) For ships of a single side skin structure intended for carrying cargoes other than liquids, girders on the shell plating may be excluded from the assessment.
- (2) For ships carrying cargoes other than bulk and liquid cargoes with the cargoes properly fastened or otherwise held in position so that the cargo loads can be deemed as acting only on the inner bottom plating and internal deck, the assessment need only be performed for the inner bottom plating and the internal deck.
- (3) Not required for girders on shell plating and weather deck.

# 7.2.3 Bending Strength

Paragraph 7.2.3.1 has been amended as follows.

#### 7.2.3.1 Section Modulus

In each assessment condition, the section modulus of simple girders is to be not less than that obtained from the following formula:

$$Z_{n50} = C_{Safety} \frac{|M|}{\sigma_{all} - \sigma_{BM}} \times 10^3 \, (cm^3)$$

 $C_{Safety}$ : Safety factor to be taken as  $\frac{1.0}{1.1}$ 

M: Maximum moment (kN-m) of the assessment model as specified in 7.2.3.2

 $\sigma_{all}$ : Permissible bending stress (N/mm<sup>2</sup>) to be taken as follows:

$$\sigma_{all} = \frac{235}{K}$$

K: Material factor as specified in 3.2.1.2

 $\sigma_{BM}$ : Stress ( $N/mm^2$ ) due to the hull girder load at the girder to be assessed as specified in 7.2.2.3. However, for members other than longitudinal hull girder structural members,  $\sigma_{BM}$  is to be taken as 0.

# 7.2.6 Bending Stiffness

# 7.2.6.1 Depth of Girders

For the members specified in **Table 7.2.6-1**, depth is not to be less than that specified in the table. However, the depth may be reduced provided that the member has equivalent moment of inertia or deflection to the required members.

Table 7.2.6-1 has been amended as follows.

Table 7.2.6-1 Depth of Girders

Members	Depth of Girders (m)			
Web frame	$0.1$ $\ell_{bdg}$			
Web frame supporting cantilever	0.125 <i>€ℓ<sub>bdg</sub></i>			
Web frame supporting side stringer	0.125 <del>ℓ</del> ℓ <sub>bdg</sub>			
Side stringer	0.125 <i>€ℓ<sub>bdg</sub></i>			
Side stringer forward of collision bulkhead	0.2 <del>∉</del> ℓ <sub>bdg</sub>			
Web frame forward of collision bulkhead	0.2 <del>ℓ</del> ℓ <sub>bdg</sub>			
Note:				
$\ell_{bdg}$ : Effective bending span (m) of the girder as given in 3.6.1.4				

Paragraph 7.2.6.2 has been amended as follows.

#### 7.2.6.2 Moment of Inertia of Girders

For the members specified in **Table 7.2.6-2**, moment of inertia is not to be less than that obtained from following formula. However, the loads acting on the members is not distributed loads, the moment of inertia is to be deemed appropriate by the Society.

$$I = \frac{SP \ell \ell_{bdg}^4}{384E\delta} \times 10^8 \ (cm^{24})$$

S: Spacing of girders (m)

P: Loads corresponding to each assessment condition specified in Table 7.2.6-1, member as given in Table 7.2.6-2. Loads are to be calculated at the middle of  $\ell$ 

 $\ell$ : Full length of girders (m)

 $\ell_{bda}$ : Effective bending span (m) of the girder as given in 3.6.1.4

E: Young's modulus to be taken as  $206,000 (N/mm^2)$ 

 $\delta$ : Allowable values for deflection of girders as specified in **Table 7.2.6-2** 

Table 7.2.6-2 has been amended as follows.

Table 7.2.6-2 Allowable values for deflection of girders

	Loads P	Allowable values $\delta$
Girders attached to decks (except for cantilever)	Static pressure $P_{dks}$ of cargo specified in 4.4.2.7 and green sea pressure $P_{GW}$ specified in 4.4.2.8, whichever is greater.	$\ell/1340 \times 10^3$
Girders supporting stiffeners attached to watertight bulkheads	Internal Pressure $P_{FD=in}$ in flooded condition specified in 4.4.4.1	$S/670 \times 10^3$
Girders supporting stiffeners in deep tanks	Static pressure $P_{ls}$ specified in 4.4.2.4	$S/2000 \times 10^3$

#### 7.3 Double Hull Structures

#### 7.3.1 General

Paragraph 7.3.1.4 has been amended as follows.

#### 7.3.1.4 Loading Conditions

The loading conditions to be considered in double hull strength assessments are to be in accordance with  $\frac{4.4}{4.4}$ , the requirements for loads to be considered for the strength of primary supporting structures specified in Chapter 4, Part 2 (Requirements by ship type). The loading condition of which  $P_{DH} = 0$  may be waived.

## 7.3.2 Requirements for Scantlings

Paragraph 7.3.2.1 has been amended as follows.

#### **7.3.2.1** Bending Strength

In each assessment condition, the thickness of plating of double hull is to be in accordance with the following requirements (1) and (2). The thickness of plating according to these requirements is to be uniform at any point in the double hull under assessment.

(1) The thickness of bottom shell plating and inner bottom plating constituting a double bottom and that of side shell plating and side longitudinal bulkheads constituting a double side are to be not less than that obtained from the following formula:

be not less than that obtained from the following formula:
$$t_{n50} = \frac{C_{Safety}}{C_{cnd}} \frac{(1 - v^2)}{D_{DH}} \times \max \left( \frac{|M_X|}{\gamma_{CHR_{\underline{Stf}-X}} \underline{C_{bi-X}}} (\sigma_{all} - \sigma_{BM}), \frac{|M_Y|}{\gamma_{stf-y} C_{bi-y}} \underline{C_{\underline{BH}}} \sigma_{all} \right) \quad (mm)$$

 $C_{Safety}$ : Safety factor to be taken as 1.2

γ<sub>crn</sub>: Coefficient of the bending stiffness effect of girder members and stiffeners to be taken as 1.12

Coefficient of strength decrease due to transverse bending to be taken as 0.5

 $\gamma_{stf-x}$ : Coefficient of the bending stiffness effect of stiffeners in the longitudinal direction, as given in Table 7.3.2-1

 $\gamma_{stf-y}$ : Coefficient of the bending stiffness effect of stiffeners in the transverse direction, as given in Table 7.3.2-1

 $\underline{C_{bi-x}}$ : Coefficient of strength decrease due to bending in the longitudinal direction, as given in Table 7.3.2-1

 $C_{bi-y}$ : Coefficient of strength decrease due to bending in the transverse direction, as given in Table 7.3.2-1

 $M_X$ : Longitudinal bending moment (kN-m/m) per unit width in double hull as given in 7.3.3.1 (See Fig. 7.3.2-1).

 $M_Y$ : Transverse bending moment (kN-m/m) per unit length in the double bottom hull as given in 7.3.3.1. When assessing double bottoms, read  $M_Y$ . When assessing double sides,  $M_Y$  is to be read as  $M_Z$ ; *i.e.*, the vertical bending moment per unit length. (See Fig. 7.3.2-1).

 $\sigma_{BM}$ : Stress ( $N/mm^2$ ) due to hull girder bending at the member under assessment to be taken as follows:

$$\sigma_{BM} = \left[ \left| \frac{M_{V-HG}}{I_{y-n50}} (z - z_B) \right| + \left| \frac{M_{H-HG}}{I_{Z-n50}} y \right| \right] \times 10^5$$

 $M_{V-HG}$ : Vertical bending moment (kN-m) corresponding to each assessment condition as given in **4.6.2.10** for the maximum load condition and **4.6.3.5** for the harbour condition.

 $M_{H-HG}$ : Horizontal bending moment (kN-m) corresponding to each assessment condition as given in **4.6.2.10** for the maximum load condition and **4.6.3.5** for the harbour condition.

 $I_{y-n50}$ : Moment of inertia ( $cm^4$ ) of the transverse section at the midpoint of  $\ell_{DB}$  about its horizontal neutral axis (net scantlings). The corrosion addition is given in 3.3.4.

 $I_{z-n50}$ : Moment of inertia  $(cm^4)$  of the transverse section at the midpoint of  $\ell_{DS}$  about its vertical neutral axis (net scantlings). The corrosion addition is given in 3.3.4.

z: Position (m) of the double hull on the Z coordinate, which is defined as follows:

For bottom shell plating, this position is to be taken as the lowest point of the bottom shell plating of the double bottom under consideration at  $x_{DB} = 0$ .

For inner bottom plating, this position is to be taken as the lowest point of the inner bottom plating of the double bottom under consideration at  $x_{DB} = 0$ .

For side shell plating, this position is to be taken as either of the upper and lower ends of the side shell plating of the double side under consideration at  $x_{DS} = 0$ , whichever is greater in distance from  $z_n$ .

For longitudinal bulkhead, this position is to be taken as either of the upper and lower ends of the longitudinal bulkhead of the double side under consideration at  $x_{DS} = 0$ , whichever is greater in distance from  $z_n$ .

y: Position (m) of the double hull on the Y coordinate, which is defined as follows:

For bottom shell plating, this position is to be taken as the outermost point of the bottom shell plating of the double bottom under consideration at  $x_{DR} = 0$ .

For inner bottom plating, this position is to be taken as the outermost point of the inner bottom plating of the double bottom under consideration at  $x_{DB} = 0$ .

For side shell plating, this position is to be taken as the point most distant from the centreline of the side shell plating of the double side under consideration at  $x_{DS} = 0$ .

For longitudinal bulkhead, this position is to be taken as the point most distant from the centreline of the longitudinal bulkhead of the double side under consideration at  $x_{DS} = 0$ .

(2) Notwithstanding (1) above, where any of the requirements specified in 2.4.1.2-6(1) and

**2.4.1.3-1(1)** for the spacing of girders and floors in double bottom is not satisfied, the thickness of the bottom shell plating and inner bottom plating constituting a double bottom is to be not less than that obtained from the following formula. Similarly, if any of the requirements specified in **2.4.2.1(1)** and **2.4.2.2(1)** for the spacing of side transverses and side stringers is not satisfied, the thickness of the side shell plating and longitudinal bulkheads constituting a double side is to be not less than that obtained from the following formula. However,  $C_{EX} = 1.0$  where no longitudinal girders are provided, while  $C_{EY} = 1.0$  where no transverse girders are provided.

$$t_{n50} = \frac{C_{Safety}}{C_{cnd}} \frac{(1 - v^2)}{\gamma_{cm}} \times max \left( \frac{|M_X|}{\gamma_{stf-x} C_{bi-x} C_{EX} (\sigma_{all} - \sigma_{bM})}, \frac{|M_Y|}{\gamma_{stf-y} C_{bi-y} C_{EY} C_{EX} (\sigma_{all} - \sigma_{bM})} \right) (mm)$$

$$C_{Safety}: \quad \text{Safety factor to be taken as } 1.2$$

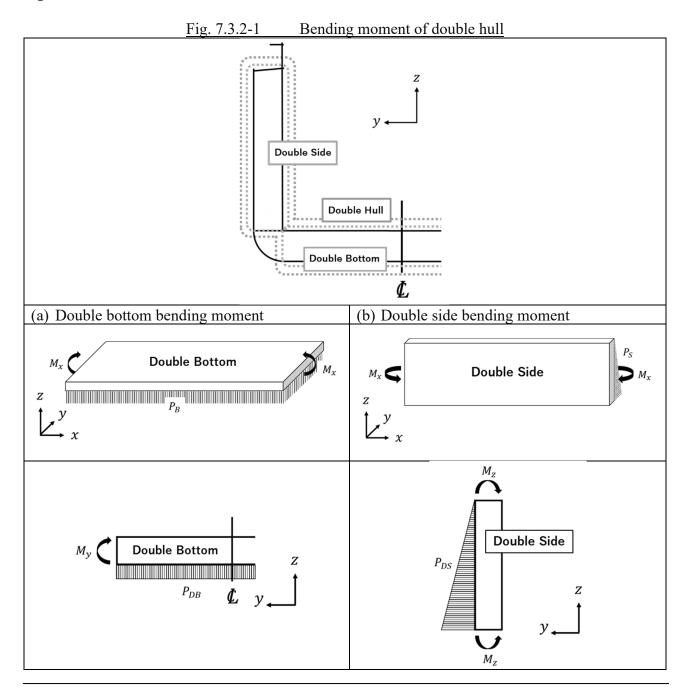
$$\gamma_{Cir}, \quad C_{M}, \quad \gamma_{stf-x}, \quad \gamma_{stf-y}, \quad C_{bi-x}, \quad C_{bi-y}, \quad M_{*X}, \quad M_{*Y} \quad \text{and} \quad \sigma_{bM}: \quad \text{As specified in } (1)$$

Table 7.3.2-1 has been added as follows.

Table 7.3.2-1 Bending Strength Parameters for Double Hull

	Longitudinal	Transverse
	framing system	framing system
$C_{bi-x}$	<u>1.0</u>	<u>0.5</u>
$C_{bi-y}$	0.5	1.0
$\gamma_{stf-x}$	1.1	1.0
$\gamma_{stf-v}$	1.0	1.1

Fig. 7.3.2-1 has been added as follows.



#### **Moments and Shear forces** 7.3.3

#### 7.3.3.1 **Moments**

Sub-paragraph (2) has been amended as follows.

(2) The transverse or vertical bending moment per unit length in the double hull is to be obtained

$$M_Y = C_{MY} P_{DH} B_{DH}^2 (kNm/m)$$

from the following formula:  $M_Y = C_{MY} P_{DH} B_{DH}^2 \quad (kNm/m)$   $C_{MY}$ : Maximum transverse bending moment coefficient to be taken as follows depending on the structural arrangement: When assessing double bottom, read  $C_{MY}$ . When assessing double side,  $C_{MY}$  is to be read as  $C_{MZ}$ 

((a) to (d) are omitted.)

Table 7.3.3-2 has been amended as follows.

Table 7.3.3-2 Moment Correction Factor  $\beta_B$  for Double Bottom

Type of structure	$\beta_B$
D1	$\frac{8P_{\rm BL}D_{\rm BB}B_{\rm BS}^{-\frac{3}{2}} + 15P_{\rm BB}D_{\rm BS}B_{\rm BB}^{-\frac{3}{2}} + 120P_{\rm BS}ED_{\rm BS}D_{\rm BB}}{12} \qquad \qquad \frac{12}{P_{\rm BB}B_{\rm BS}^{-\frac{3}{2}}}$
	$\frac{28P_{DS}D_{DB}B_{DS}^3 + 15P_{DB}D_{DS}B_{DB}^3}{240D_{DB}B_{DS} + 180D_{DS}B_{DB}} \times \frac{12}{P_{DB}B_{DB}^2}$
D2	$\frac{8P_{DS}D_{DB}B_{DS}^{3} + 15P_{DB}D_{DS}B_{DB}^{3}}{120D_{DB}B_{DS} + 180D_{DS}B_{DB}} \times \frac{12}{P_{DB}B_{DB}^{2}}$
D3	$\frac{3P_{DS}D_{DB}B_{DS}^{3} + 10P_{DB}D_{DS}B_{DB}^{3}}{60D_{DB}B_{DS} + 120D_{DS}B_{DB}} \times \frac{12}{P_{DB}B_{DB}^{2}}$
D4	$\frac{3P_{DS}D_{DB}B_{DS}^{3} + 5P_{DB}D_{DS}B_{DB}^{3}}{60D_{DB}B_{DS} + 60D_{DS}B_{DB}} \times \frac{12}{P_{DB}B_{DB}^{2}}$

Table 7.3.3-3 has been amended as follows.

Table 7.3.3-3 Moment Correction Factor  $\beta_S$  for Double Side

	b internet content accer by not be used size
Type of structure	eta s
D1	$\frac{8P_{05}D_{05}B_{05}^{-3} + 15P_{05}D_{05}B_{05}^{-3} + 120P_{05}ED_{05}D_{05}}{720ED_{05}D_{05}} \times \frac{15 + \frac{90ED_{05}}{9_{05}^{-3}}}{120D_{05}B_{05} + 180D_{05}B_{05} + \frac{720ED_{05}D_{05}}{9_{05}^{-3}}} \times \frac{P_{05}D_{05}^{-2} + \frac{15P_{05}ED_{05}}{9_{05}^{-3}}}{\frac{15P_{05}ED_{05}}{9_{05}^{-3}}}$
	$\frac{28P_{DS}D_{DB}B_{DS}^3 + 15P_{DB}D_{DS}B_{DB}^3}{240D_{DB}B_{DS} + 180D_{DS}B_{DB}} \times \frac{60}{7P_{DS}B_{DS}^2}$
D2	$\frac{8P_{DS}D_{DB}B_{DS}^{3} + 15P_{DB}D_{DS}B_{DB}^{3}}{120D_{DB}B_{DS} + 180D_{DS}B_{DB}} \times \frac{15}{P_{DS}B_{DS}^{2}}$
D3	$\frac{3P_{DS}D_{DB}B_{DS}^{3} + 10P_{DB}D_{DS}B_{DB}^{3}}{60D_{DB}B_{DS} + 120D_{DS}B_{DB}} \times \frac{20}{P_{DS}B_{DS}^{2}}$
D4	$\frac{3P_{DS}D_{DB}B_{DS}^{3} + 5P_{DB}D_{DS}B_{DB}^{3}}{60D_{DB}B_{DS} + 60D_{DS}B_{DB}} \times \frac{20}{P_{DS}B_{DS}^{2}}$

#### 7.3.3.2 Shear Forces

Sub-paragraphs (3) and (4) have been amended as follows.

((1) and (2) are omitted.)

(3) The shear force in side stringers is to be given by the following formula:

$$F = C_{f(x_{DH})}C_{f(z_{DH})}C_{FX}P_{DH}SB_{DS} (kN)$$

 $C_{f(x_{DH})}$ : To be taken as follows depending on the hatchway width:

(a) For the hatchway width is not more than 0.7*B*, or for no hatchway is provided

$$C_{f(x_{DH})} = \max\left(0.5, 2.0 \frac{|x_{DH}|}{\ell_{DS}}\right)$$

(b) For the hatchway width exceeds 0.7B

$$C_{f(x_{DH})} = \max\left(0.5, \min\left(1.0, 2.5 \frac{|x_{DH}|}{\ell_{DS}} + 0.25\right)\right)$$

$$C_{f(z_{DH})} = 1.0$$

 $C_{FX}$ : Maximum shear force coefficient for side stringers to be taken as follows:

$$C_{FX} = C_{FXS} + \beta_S (C_{FXF} - C_{FXS})$$

 $\beta_S$ : Double side moment correction factor as given in **Table 7.3.3-3**. However,  $\beta_S$  is to be taken as 1.0 where greater than 1.0.

 $C_{FXS}$  and  $C_{FXF}$ : Coefficients obtained according to the value of  $\alpha_{EQ}$  from **Tables 7.3.3-18** to **Table 7.3.3-23**, depending on the structural model. For intermediate values of  $\alpha_{EQ}$ , these coefficients are to be determined by interpolation.

(4) The shear force in side transverses is to be given by the following formula:

$$F = C_{f(x_{DH})}C_{f(z_{DH})}C_{FZ}P_{DH}SB_{DS}(kN)$$

 $C_{f(x_{DH})}$ : To be taken as follows depending on  $\alpha_{EQ}$  for the double side:

(a) For 
$$\alpha_{EQ} \ge 4.0$$

$$C_{f(x)} = 1.0$$

(b) For  $\alpha_{EQ} < 4.0$ 

$$C_{f(x)} = \min\left(1.0, \max\left(0.5, C_{AS}\left(0.5 - \frac{|x_{DH}|}{\ell_{DS}}\right) + 0.5\right)\right)$$

$$C_{AS} = \min\left(4.0, \max\left(1.667, \frac{1.0}{0.467\left(\frac{1.0}{\alpha_{EO}} - 0.25\right)} + 0.25\right)\right)$$

 $C_{f(z_{DH})}$ : To be taken as follows depending on the hatchway width:

(a) For the hatchway width is not more than 0.7B or for no hatchway is provided When hatchways are provided (D1 and D2 Type)

$$C_{f(z_{DH})} = \max\left(0.3, \min\left(1.0, 1.0 - 1.667 \frac{|z_{DH}|}{B_{DS}}\right)\right)$$

(b) For the hatchway width exceeds 0.7B When no hatchways are provided (D3 and D4 Type)

$$C_{f(z)} = \max\left(0.5, \min\left(1.0, 1.0 - 1.667 \frac{|z_{DH}|}{B_{DS}}\right)\right)$$

 $C_{FZ}$ : Maximum shear force coefficient for side transverses to be taken as follows:

$$C_{FZ} = C_{FZS} + \beta_S (C_{FZF} - C_{FZS})$$

 $\beta_S$ : Double side moment correction factor as given in Table 7.3.3-3

 $C_{FZS}$  and  $C_{FZF}$ : Coefficients obtained according to the value of  $\alpha_{EQ}$  from **Tables 7.3.3-18** to **Table 7.3.3-23**, depending on the structural model. For intermediate values of  $\alpha_{EQ}$ , these coefficients are to be determined by interpolation.

Table 7.3.3-18 has been amended as follows.

Table 7.3.3-18 Coefficients  $C_{FXS}$  and  $C_{FZS}$  at Double Side (D1) Where Boundary Condition with

Bottom Structure Is "Supported"

$\alpha_{EQ}$	0.25 or less	0.4	0.5	0.6	0.7	0.8	0.9
$C_{FXS}$	0.177	0.245	0.279	0.307	0.330	0.349	0.365
$C_{FZS}$	0.141	0.226	9.277	0.324	0.367	0.407	0.443
$\alpha_{EQ}$	1.0	1.2	1.6	2.0	2.5	4.0 or more	
$C_{FXS}$	0.474	0.786	1.574	2.498	3.745	8.412	
$C_{FZS}$	0.477	0.535	0.624	0.684	0.732	0.786	

$lpha_{EQ}$	≤ 0.25	0.4	0.5	0.6	0.7	0.8	0.9
$C_{FXS}$	0.083	0.137	0.173	0.207	0.236	0.262	0.286
$C_{FZS}$	0.082	0.127	0.153	0.178	0.200	0.219	0.238
$\alpha_{EQ}$	1.0	1.2	1.6	2.0	2.5	4.0 ≤	
$C_{FXS}$	0.308	0.340	0.386	0.415	0.435	0.524	
$C_{FZS}$	0.254	0.284	0.328	0.360	0.382	0.409	

Table 7.3.3-19 has been amended as follows.

Table 7.3.3-19 Coefficients  $C_{FXF}$  and  $C_{FZF}$  at Double Side (D1) Where Boundary Condition with Bottom Structure Is "Fixed"

_			***************************************	Bottom Str		100		
	$\alpha_{EQ}$	0.25 or less	0.4	0.5	0.6	0.7	0.8	0.9
	$C_{FXF}$	0.100	0.144	0.167	0.186	0.201	0.214	0.223
	$C_{FZF}$	0.135	0.210	0.232	0.288	0.320	0.348	0.372
	$\alpha_{EQ}$	1.0	1.2	1.0	2.0	2.5	4.0 or more	
	$C_{FXF}$	0.231	0.345	0.548	0.687	0.781	0.927	
	$C_{FZF}$	0.394	0.429	0.474	0.495	0.504	0.498	$\angle$

$lpha_{EQ}$	≤ 0.25	0.4	0.5	0.6	0.7	0.8	0.9
$C_{FXF}$	0.083	0.136	0.168	0.194	0.217	0.227	0.236
$C_{FZF}$	0.159	0.235	0.277	0.313	0.344	0.371	0.396
$\alpha_{EQ}$	1.0	1.2	1.6	2.0	2.5	4.0 ≤	
							1
$C_{FXF}$	0.242	0.247	0.242	0.245	0.249	0.235	

Table 7.3.3-22 has been amended as follows.

Table 7.3.3-22 Coefficients  $C_{FXS}$  and  $C_{FZS}$  at Double Side (D3 or D4) Where Boundary Condition with Bottom Structure Is "Supported"

$lpha_{EQ}$	0.25 or less	0.4	0.5	0.6	0.7	0.8	0.9
$C_{FXS}$	0.122	0.160	0.200	0.225	0.249	0.274	0.296
$C_{FZS}$	0.064	0.107	0.133	0.155	0.175	0.192	0.205
$\alpha_{EQ}$	1.0	1.2	16	2.0	2.5	4.0 or more	
$C_{FXS}$	0.314	0.340	0.360	0.364	0.305	0.365	
$C_{FZS}$	0.216	0.232	0.248	0.253	0.254	0.254	

$lpha_{EQ}$	≤ 0.25	0.4	0.5	0.6	0.7	0.8	0.9
$C_{FXS}$	0.107	0.158	0.188	0.215	0.239	0.260	0.278
$C_{FZS}$	0.070	0.113	0.138	0.161	0.181	0.197	0.211
$\alpha_{EQ}$	1.0	1.2	1.6	2.0	2.5	4.0 ≤	
$C_{FXS}$	0.293	0.312	0.322	0.318	0.310	0.287	
$C_{FZS}$	0.222	0.237	0.253	0.258	0.259	0.259	

Table 7.3.3-23 has been amended as follows.

Table 7.3.3-23 Coefficients  $C_{FXF}$  and  $C_{FZF}$  at Double Side (D3 or D4) Where Boundary

Condition with Bottom Structure Is "Fixed" 0.5 0.6 0.8 0.9  $\alpha_{EO}$ 0.25 or less 0.107 0.1520.175 0.195 0.211 0.228 0.239  $C_{FXF}$ 0.2850.1400.215 0.308 0.323 0.333 $C_{FZF}$ 4.0 or more  $\alpha_{EQ}$ 1.0 1.2 2.5 0.247 0.255 0.255 0.258 253  $C_{FXF}$  $C_{FZF}$ 3.350 0.350 0.350 0.350 0.350 0.350

$lpha_{EQ}$	≤ 0.25	0.4	0.5	0.6	0.7	0.8	0.9
$C_{FXF}$	0.100	0.144	0.167	0.186	0.202	0.215	0.223
$C_{FZF}$	0.135	0.209	0.250	0.282	0.305	0.321	0.332
$lpha_{EQ}$	1.0	1.2	1.6	2.0	2.5	4.0 ≤	
$C_{FXF}$	0.228	0.232	0.226	0.220	0.213	0.195	
$C_{FZF}$	0.339	0.345	0.344	0.341	0.340	0.340	

#### 7.4 Pillars, Struts, Etc.

## 7.4.2 Scantling Requirements

Paragraph 7.4.2.1 has been amended as follows.

#### 7.4.2.1 Buckling Strength Requirements (Euler Buckling)

For members subject to axial compressive loads, such as pillars or struts, their sectional area is to be not less than that obtained from the following formula:

$$A_{n50} = C_S \frac{F}{\sigma_{cr}} \times 10 \ (cm^2)$$

 $C_S$ : Safety factor to be taken as 1.4. However, when struts are placed between longitudinals in double bottom and double side,  $C_S$  is to be taken as 2.8.

F: Compressive load (kN) specified in each requirement. However, the compressive load may be obtained by direct strength analysis.

 $\sigma_{cr}$ : Buckling strength of beams and pillars or such members as struts to be taken as follows:

For 
$$\sigma_E > \frac{\sigma_Y}{2}$$
:  $\sigma_{cr} = \sigma_Y \left( 1 - \frac{\sigma_Y}{4\sigma_E} \right) (N/mm^2)$   
For  $\sigma_E \le \frac{\sigma_Y}{2}$ :  $\sigma_{cr} = \sigma_E (N/mm^2)$   
 $\sigma_E = C_{BC} \pi^2 E \left( \frac{k}{l} \right)^2 (N/mm^2)$ 

k: Minimum radius (mm) of gyration of beams and pillars or members such as struts

*l*: Distance (*mm*) from the top of the inner bottom plating, deck or any other structure, to which the lower end of pillars, struts, etc., is attached, to the bottom of the beam or deck girder supported by the pillars, struts, etc.

 $C_{BC}$ : Fixed end effect coefficient as specified in the following i) to iii):

i) For corrugated bulkheads supported at each end with a stool with a width

exceeding 2 times the depth of the corrugation

$$C_{BC}=4$$

ii) For corrugated bulkheads or cross ties supported at one end with a stool with a width exceeding 2 times the depth of the corrugation

$$C_{BC}=2$$

iii) Other cases

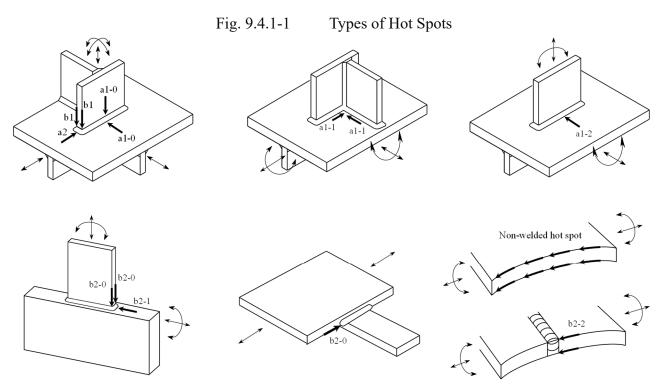
$$C_{BC}=1$$

## **Chapter 9 FATIGUE**

## 9.4 Finite Element Analysis

## 9.4.1.4 Types of Hot Spot

The types of hot spots are described in **Table 9.4.1-1**. These are defined according to their location on the plate and their orientation to the weld toe as shown in **Fig. 9.4.1-1**.



#### **9.4.2.7** Mesh Size

Sub-paragraph (4) has been renumbered Sub-paragraph (5), and Sub-paragraph (4) has been added as follows.

- 1 The size of the very fine mesh is to be in accordance with the following (1) to (4):
- (1) The very fine mesh finite element model for the evaluation of "a" type hot spot stress is to be based on the shell element of standard mesh size  $t_{gr} \times t_{gr}$ , where  $t_{gr}$  is the gross thickness (mm) of the member for evaluation of the considered hot spot.

- (2) The very fine mesh finite element model for the evaluation of "b" type hot spot stress is to be based on the shell element of standard mesh size  $10 \text{ } mm \times 10 \text{ } mm$ .
- (3) Except for the following (a) to (c), the mesh size of hot spots consisting of plates of different thicknesses may, in principle, be the thickness of the thinnest plate among those evaluated.
  - (a) When the member for which fatigue strength is considered critical is assessed as "a" type, the mesh size is to be equal to the thickness of that member.
  - (b) When the mesh size for "a" type hot spot is smaller than half the plate thickness, very fine mesh zones corresponding to the respective mesh sizes are to be prepared.
  - (c) When "a" and "b" type hot spots exist together in one structural detail and the thickness of the plate with "a" type hot spot exceeds 20 mm, very fine mesh zones corresponding to the respective mesh sizes are to be prepared.
- (4) Notwithstanding the requirements in (1) to (3) above, a mesh size which is smaller than half the plate thickness may be used for type "a" hot spots where deemed appropriate by the Society.
- (4<u>5</u>) The mesh size of very fine mesh finite element models for the evaluation of "a2" type spot is not to be greater than the thickness of the considered plate.
- 2 The very fine mesh zone is to be extending over at least 10 elements in all directions from the hot spot. The transition of the surrounding element size between the coarser mesh and the very fine mesh zone is to be done gradually. This transition mesh is to be such that a uniform mesh with regular shape gradually transitions from smaller elements to larger ones. An example of the mesh transition in way of the side frame bracket toe is shown in **Fig. 9.4.2-1**.

## 9.4.5.2 Hot Spot Locations and Stress Readout Points

1 The hot spot locations and stress readout points for welded joints are to be in accordance with the following (1) to (4) depending on the type of hot spots. Examples of the types of welded joints and the joints reproduced using shell models are shown in Fig. 9.4.5-1.

Sub-paragraphs (1) and (2) have been amended as follows.

- (1) Type a1 hot spot (See Fig. 9.4.5-2 to Fig. 9.4.5-4)
  - (a) The location of type al hot spot is the position shifted  $x_{shift}$  (mm) in the direction orthogonal to the weld line on the plates surface from the position of the node at the intersection of the shell elements of the plates. When assessing hot spot stress, the hot spot stress is obtained by reading out the stresses at positions shifted  $0.5t_{n25}$  (mm) and  $1.5t_{n25}$  (mm) in the direction orthogonal to the weld line from the hot spot location, based on the thickness of the plates for which hot spot stress is to be assessed. However, when using a mesh size smaller than half the thickness in accordance with 9.4.2.7-1(4), the hot spot stress is obtained by reading out the stress at the positions shifted  $0.4t_{n25}$  (mm) and  $1.0t_{n25}$  (mm) in the direction orthogonal to the weld line from the hot spot.
  - (b)  $x_{shift}$  is to be obtained from the following formula:

Cases other than  $\phi = 90^{\circ}$ 

When continuous plates are assessed

$$x_{shift} = \frac{t_{2n25}}{2\sin\phi} + \frac{t_{1n25}}{2\tan\phi} \ (mm)$$

When discontinuous plates are assessed +

$$x_{shift} = \frac{t_{2n25}}{2\sin\phi} \ (mm)$$

Case  $\phi = 90^{\circ}$ 

$$x_{shift} = \frac{t_{2n25}}{2} \ (mm)$$

 $t_{1n25}$ : Thickness (mm) of plate containing the hot spot to be assessed (main plate)

 $t_{2n25}$ : Thickness (mm) of plate intersecting the main plate (attached plate)

φ: Angle (deg) between attached plate and main plate

(2) Type a2 hot spot (See Fig. 9.4.5-5)

The location of type a2 hot spot is the position of the node at the intersection between the shell element of a plate and the shell element edge face of a plate, gusset plate or bracket. The hot spot stress is obtained by reading out the stress at positions shifted  $0.5t_{n25}$  (mm) and  $1.5t_{n25}$  (mm) in the direction extending to the gusset plate or bracket from the hot spot, based on the thickness of the plate for which hot spot stress is to be assessed. However, when using a mesh size smaller than half the thickness in accordance with 9.4.2.7-1(4), the hot spot stress is obtained by reading out the stress at the positions shifted  $0.4t_{n25}$  (mm) and  $1.0t_{n25}$  (mm) in the direction extending to the gusset plate or bracket from the hot spot.

- (3) Type b1 hot spot (See Fig. 9.4.5-6)
  - (a) The location of type b1 hot spot is the position shifted  $x_{shift}$  (mm) along the shell element edge face of the plate, gusset plate, or bracket from the position of the node at the intersection between the shell element edge face of a plate, gusset plate or bracket and the shell element of the plate. When assessing hot spot stress, the hot spot stress is obtained by reading out the stresses at position shifted 5 mm and 15 mm along the shell element edge face of the plate, gusset plate, or bracket from the hot spot.
  - (b)  $x_{shift}$  is taken as follows:

$$x_{shift} = \frac{t_{2n25}}{2\sin\phi} \ (mm)$$

 $\phi$ : Angle (deg) between gusset plate edge and plate intersecting the considered plate (attached plate)

 $t_{2n25}$ : Thickness (mm) of plate intersecting the considered plate (attached plate)

- (4) Type b2 hot spot (See Fig. 9.4.5-7)
  - (a) The location of type b2 hot spot is the position of the node at the intersection between the shell element edge face of a plate, gusset plate, or bracket and the shell element of a plate. When assessing the hot spot stress at the corner of plate edge or the plate edge face, the hot spot stress is obtained by reading out the stresses at positions shifted 5 mm and 15 mm along the corner of plate edge or the plate edge face from the hot spot. When assessing the hot spot at a weld toe at the corner of plate edge welded by butt joint, the hot spot stress is obtained by reading out the stresses of two elements adjoining the hot spot.
  - (b) Type b2-1 hot spot stress is to be corrected by multiplication with the following factor:

$$f_{\text{HSS}} = 0.436 \cdot \left(\frac{t_{2-n25}}{t_{1-n25}} - 1\right)^2 + 1$$

 $t_1$ : Thickness (mm) of plating to be assessed

 $t_2$ : Thickness (mm) of plating not to be assessed

Paragraph 9.4.5.4 has been amended as follows.

#### 9.4.5.4 Hot Spot Stress

- 1 The hot spot stress is to be obtained by linear extrapolation of the stress at the point on stress readout line specified in 9.4.5.2 to the hot spot location specified in 9.4.5.2.
- The hot spot stress is, in principle, to be obtained from the following formula:

$$\sigma_{HS} = 1.5\sigma_{0.5} - 0.5\sigma_{1.5} (N/mm^2)$$

 $\sigma_{0.5}$ : Stress (N/mm<sup>2</sup>) at position shifted 0.5 $t_{n25}$  (mm) or 5 mm from hot spot

 $\sigma_{1.5}$ : Stress (N/mm<sup>2</sup>) at position shifted 1.5 $t_{n25}$  (mm) or 15 mm from hot spot

3 Notwithstanding the requirements specified in -2, the hot spot stress is to be obtained from the following formula when using a mesh size smaller than half the plate thickness in accordance with the requirements in 9.4.2.7-1(4).

 $\sigma_{HS} = 1.67 \cdot (1.0\sigma_{0.4} - 0.4\sigma_{1.0}) (N/mm^2)$ 

 $\sigma_{0.4}$ : Stress (N/mm<sup>2</sup>) at position shifted 0.4 $t_{n25}$  (mm) from hot spot

 $\sigma_{1.0}$ : Stress (N/mm<sup>2</sup>) at position shifted 1.0 $t_{n25}$  (mm) from hot spot

34 Notwithstanding the requirements specified in -2 and -3 above, when the hot spot on the tank cover at the intersection of the tank cover and the upper deck of ships carrying liquefied gases in bulk (independent spherical tanks), and the hot spot on the coaming at the intersection of the coaming of the tank dome opening and the upper deck of ships carrying liquefied gases in bulk (independent prismatic tanks) are to be evaluated, the hot spot stress for  $\Delta \sigma_{ort\_j}$  is obtained from the following formula. The stress at  $x_{shift}$  position is to be obtained according to the procedure specified in -2 and -3 above:

 $\sigma_{HS} = \left[ \overline{\sigma_{membrane}(x_{shift})} + 0.60 \cdot \sigma_{bending}(x_{shift}) \right] (N/mm^2)$ 

 $\sigma_{membrane}(x_{shift})$ : Membrane stress (N/mm<sup>2</sup>) at  $x_{shift}$  position

 $\sigma_{bending}(x_{shift})$ : Bending stress  $(N/mm^2)$  at  $x_{shift}$  position obtained from the following formula:

 $\sigma_{bending}(x_{shift}) = \sigma_{surface}(x_{shift}) - \sigma_{membrane}(x_{shift})$   $\sigma_{surface}(x_{shift})$ : Surface stress (N/mm<sup>2</sup>) at  $x_{shift}$  position

## Chapter 10 ADDITIONAL STRUCTURAL REQUIREMENTS

#### 10.4 Deck Structure

#### 10.4.6 Helicopter Decks

Paragraph 10.4.6.2 has been amended as follows.

#### 10.4.6.2 Longitudinals and Beams of Helicopter Decks

The section modulus of the longitudinals and beams of a helicopter deck is not to be less than that obtained by the following formula:

$$\frac{M}{C_{total}} \times 10^{3} (cm^{2})$$

$$C_{safety} \frac{M}{\sigma_{V}} \times 10^{3} (cm^{3})$$

 $\sigma_{\rm Y}$ : Specified minimum yield stress (N/mm<sup>2</sup>)

Cimi: Safety coefficient for dynamic effect of ship motion to be taken as 1.2.

C<sub>safety</sub>: Safety factor taken as 1.25.

M: Maximum bending moment (kN-m) acting on the longitudinals and beams. This value is to be the value of (1) or (2) below, whichever is greater. However, this value is to be specified in (1) when  $\ell_1 \ge \ell$ .

(1)  $\ell_{\pm} \ge \ell$  When a load of helicopter acts (See Fig. 10.4.6-1(a))

$$\frac{M}{4} = \frac{7P\ell}{40}$$

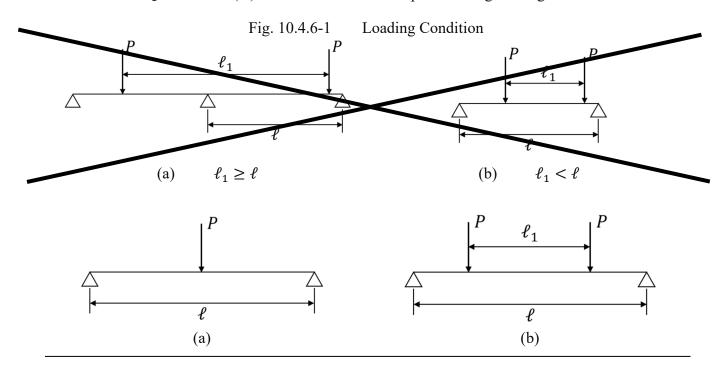
$$M = \frac{7P\ell}{40}$$
2) 
$$\frac{\ell_{\pm} < \ell \text{ When two loads of helicopter act } (See \text{ Fig. 10.4.6-1(b)})}{M = \frac{P(\ell - \ell_{1})(7\ell - 3\ell_{1})}{20\ell}}$$

$$M = \frac{P(\ell - \ell_{1})(7\ell - 3\ell_{1})}{20\ell}$$

 $\overline{P}$ : Load of helicopter (kN) (See **4.8.3.1**)

 $\ell$ : Spacing of longitudinals and beams (m)

 $\ell_1$ : Distance (m) between loads of helicopter P acting on longitudinals and beams



## 10.6 Strengthened Bottom Forward

#### **10.6.1** General

#### **10.6.1.1 Application**

Sub-paragraph -2 has been renumbered to -3, and Sub-paragraph -2 has been added as follows.

The requirements of this 10.6 are to be applied to ships having a bow draught under  $0.037L_{C230}$  in ballast condition. The ballast condition means the condition where the ballast is loaded only in ballast tanks such as the dedicated ballast tanks and separate ballast tank and in holds also used for ballasting. Where more than one ballast condition is planned for a ship, these requirements can be applied on the basis of the specific ballast condition provided in the loading manual as the ballast condition used in stormy weather. However, this provision does not include the exceptional ballast condition where the ballast is loaded in cargo oil tanks only in stormy weather to ensure the safety of the ship.  $L_{C230}$  is the length of the ship according to 1.4.2.2.

- 2 For ships subject to 10.6, assessment of strengthened bottom forward is to be carried out in accordance with 10.6.2 or 10.6.3.
- 23 The required scantlings specified in this 10.6 are to be net scantlings.
- 10.6.2 General Ships (Ship other than those with  $L_C$  of not more than 150 m,  $V/\sqrt{L_C}$  of not less than 1.4 and  $C_B$  of not more than 0.7)

Paragraph 10.6.2.1 has been amended as follows.

#### 10.6.2.1 Application

Strengthening of the bottom forward of ships other than those with  $L_{\subseteq}$  of 150 m or less,  $V/\sqrt{L_{\subseteq}}$  of 1.4 or larger, and  $C_{\sqsubseteq}$  of 0.7 or less is to be as specified in this 10.6.2. For ships other than those subject to 10.6.3, 10.6.2 is to be applied.

10.6.3 Small, Slender High-Speed Ships (Ships having  $L_C$  of not more than 150 m,  $V/\sqrt{L_C}$  of not less than 1.4, and  $C_B$  of not more than 0.7)

Paragraph 10.6.3.1 has been amended as follows.

## 10.6.3.1 Application

In ships of which  $L_G$  and  $C_B$  are not more than 150 m and 0.7 respectively, and  $V\sqrt{L_G}$  is not less than 1.4, the thickness of the shell plating at the strengthened bottom forward is to follow the requirements of this 10.6.3.1. However, ships that earry a certain amount of cargo regularly, such as Container Ships, may comply with the requirements in 10.6.2 instead.

For ships satisfying the following (1) to (3), 10.6.3 is to be applied. However, for ships always which are expected to carry a certain amount of cargo, such as container ships 10.6.2 may be applied instead.

- (1) Ships where Length  $L_c$  is not greater than 150 m
- (2)  $V/\sqrt{L_C}$  is not less than 1.4
- (3)  $C_R$  is not greater than 0.7

Section 10.9 has been added as follows.

#### 10.9 Tank Structures for Sloshing

#### **10.9.1 General**

## **10.9.1.1 Application\***

- 1 For the members of liquid cargo tank structures which satisfy the following (1) to (3), the scantlings specified in this 10.9 are considered to be satisfied when obtained using the sloshing loads specified in 4.8.2.4.
- (1) Cargo tanks with volumes of not less than  $100 m^3$
- (2) Cargo tanks designed for possible to loading at filling ratios of not less than 20 % and not more than 90 %
- (3) Where the period of longitudinal oscillation of cargo tanks is within the range of ±20 % of pitch period and within ±1.5 seconds from the same period, and where the period of transverse oscillation of cargo tanks is within the range of ±20 % of roll period and within ±1.5 seconds from the same period.
- 2 In applying -1(3) above, where only one of the conditions is applicable, only the sloshing load

due to the relevant ship motion need be considered.

- 3 In applying -1(3) above, tank natural periods are to be calculated for each 10 % of the filling ratio, and only the sloshing load due to the filling ratio corresponding to the conditions of -1(3) above need be considered.
- 4 Notwithstanding -1 above, the application of this 10.9 may be required for any tank structure deemed necessary by the Society.
- 5 Notwithstanding this 10.9, advanced methods such as numerical calculations may be required in order to determine scantling where deemed appropriate by the Society.

#### 10.9.1.2 Scantling Approach

The required scantlings specified in this 10.9 are to be net scantlings.

#### 10.9.1.3 Members to be Assessed and Loads to be Applied

This 10.9 specifies the yield strength assessment of plates on which sloshing loads act (including plate panels constituting the webs of girders), and the stiffeners attached to them. Strength assessments for their members are to be performed considering the lateral loads and hull girder loads specified in Table 10.9.1-1.

Table 10.9.1-1 Loads for Each Member to be Assessed

	1ault 10.7.	1 1	bads for E	ach Member it	oc i ibbebbea	
				Load		
Compartment to be assessed	<u>Member</u>	Lateral		Refer to the	ne following	Application
<u>be assessed</u>		load	Load type	$\frac{\text{Lateral load}}{(P_{slh}, M_{slh})}$	Hull girder load $(M_{V-HG})$	
	<u>Plates</u>			4.8.2.4-4		10.9.2.1
	Stiffener			4.8.2.4-5	4.8.2.4-7	10.9.3.1
Cargo tank	Webs of girders	Internal pressure	<u>Liquid</u> <u>loaded</u>	4.8.2.4-4		<u>10.9.4.1</u>
	Corrugated			49246		<u>10.9.2.1</u>
	<u>bulkheads</u>			4.8.2.4-6	=	<u>10.9.5.1</u>

#### 10.9.1.4 Stress Due to Hull Girder Load

The stress  $\sigma_{BM}$  (N/mm<sup>2</sup>) due to hull girder load at plates and stiffeners to be assessed is to be in accordance with the following formula.

$$\sigma_{BM} = \left| \frac{M_{V-HG}}{I_{y-n50}} (z - z_n) \right| \times 10^5$$

 $M_{V-HG}$ : Hull girder load (vertical bending moment) specified in Table 10.9.1-1(kN-m)

 $I_{y=n50}$ : Moment of inertia ( $cm^4$ ) of the hull transverse section under consideration about its horizontal neutral axis. Corrosion additions considered in the calculation are to be as specified in 3.3.4.

- z: Z coordinate (m) at the load calculation point for the member under consideration. The coordinate systems and the load calculation points are as specified in 1.4.3.6, 3.7.1 and 3.7.2 respectively.
- $z_n$ : Vertical distance (m) from the top of the keel in the transverse section under consideration to the horizontal neutral axis

#### **10.9.2** Plates

#### 10.9.2.1

The thickness of plates on which sloshing loads act is to be not less than the value obtained from the following formula.

$$\underline{t} = \frac{b}{2} \sqrt{\frac{P_{slh} \times 10^{-3}}{1.15C_a \sigma_Y}} \ (mm)$$

 $\sigma_{\rm Y}$ : Specified minimum yield stress (N/mm<sup>2</sup>)

- <u>b</u>: Length (mm) of the shorter side of the plate panel. However, it is to be taken as breadth of flange  $b_f$  (mm) or breadth of web  $b_w$  (mm) in the case of corrugated bulkheads (See Fig. 10.9.2-1)
- a: Length (mm) of the longer side of the plate panel.
- $\alpha$ : Aspect ratio, to be taken as  $\alpha/b$ .

 $P_{sth}$ : Equivalent pressure  $(kN/m^2)$  for the plate panels, as specified in Table 10.9.2-1

<u>Ca</u>: Coefficient of axial force effect as specified in Table 6.3.2-2 when  $\alpha \ge 2$  or Table 6.3.2-3 when  $\alpha < 2$ . However, it is taken as 1.0 for corrugated bulkheads.

 $\sigma_{BM}$ : Stress (N/mm<sup>2</sup>) due to hull girder bending, as specified in 10.9.1.4

Table 10.9.2-1 Equivalent Pressure for Plate Panels

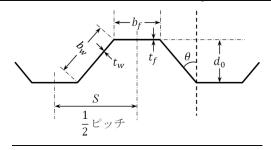
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<u>Member</u>	$\underline{P_{slh}}$
- Transverse bulkheads (including corrugated bulkheads)	
- Front and aft walls of tank	$\underline{P_{slh-p}}$
- Transverse wash bulkheads	$\frac{-\sin -p}{(4.8.2.4-4(1))}$
- Tank top plates near transverse bulkheads / front and aft walls of	(4.0.2.4-4(1))
tanks <sup>(1)</sup>	
- Longitudinal bulkheads including corrugated bulkheads	
- Tank side walls	D
- Longitudinal wash bulkheads	$\frac{P_{slh-r}}{(4.8.2.4-4(2))}$
- Tank top plates near longitudinal bulkheads / tank side walls <sup>(1)(2)</sup>	(1.0.2.1-1(2))
- Sloping plates above and below longitudinal bulkheads	

#### Notes:

Numbers in parentheses indicate section number.

- (1)  $P_{slh-p}$  applies to plate panels within the range of  $0.3\ell_{tk}$  from transverse bulkheads / front and aft walls of tanks, while  $P_{slh-r}$  applies to plate panels within the range of  $0.3b_{tk}$  from longitudinal bulkheads / tank side walls. Definitions of  $\ell_{tk}$  and  $b_{tk}$  are specified in Table 4.8.2-13 and Table 4.8.2-14.
- (2) Notwithstanding (1) above, where large sloping plates (such as plates consisting of top side tanks) are arranged between tank top plates and longitudinal bulkheads /tank side walls, the tank top plates may be excluded from the members to be assessed.

Fig. 10.9.2-1 1/2 Pitch of Corrugated Bulkheads



#### 10.9.3 Stiffeners

#### 10.9.3.1

The section modulus of stiffeners attached to plates on which sloshing loads act is to be not less than the value obtained from the following formula.

$$\underline{Z = \frac{M_{slh}}{C_s \sigma_Y} \times 10^3 (cm^3)}$$

M<sub>slh</sub>: Equivalent bending moment (kN-m), as specified in Table 10.9.3-1.

C<sub>s</sub>: Coefficient of axial force effect, as specified in Table 6.4.2-4.

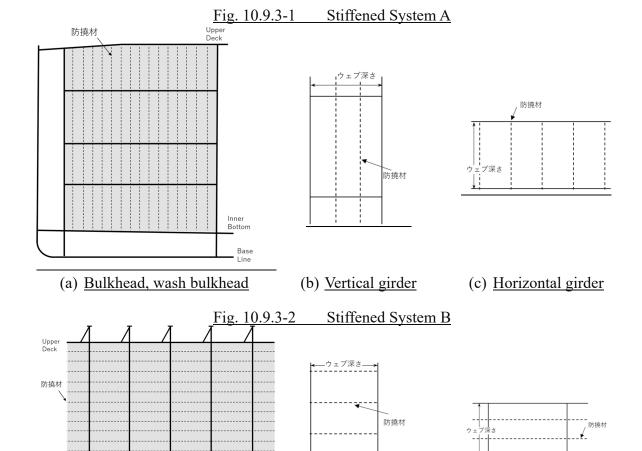
Table 10.9.3-1 Equivalent Bending Moment for Each Member to be Assessed

Member to be assessed	Stiffened system	$\underline{M_{sth}}$
Stiffeners attached to tank top plates <sup>(1) (2)</sup>	<u>Longitudinal</u>	$\frac{M_{slh-p}(4.8.2.4-5(1))}{M_{slh-r}(4.8.2.4-5(2))}$
Stiffeners attached to tank top plates	Transverse	$\frac{M_{slh-p}(4.8.2.4-5(2))}{M_{slh-p}(4.8.2.4-5(1))}$
<ul> <li>Stiffeners attached to transverse bulkheads / front and aft walls         of tank     </li> <li>Stiffeners attached to transverse wash bulkheads</li> </ul>	System A <sup>(3)</sup>	$M_{slh-p}(4.8.2.4-5(1))$
<ul> <li>Stiffeners attached to vertical girders which are attached to longitudinal bulkheads / tank side walls</li> <li>Stiffeners attached to horizontal girders which are attached to transverse bulkheads / front and aft walls of tank</li> <li>Stiffeners attached to cross-tie of transverse direction</li> </ul>	System B <sup>(4)</sup>	$M_{slh-p}(4.8.2.4-5(2))$
<ul> <li>Stiffeners attached to longitudinal bulkheads / tank side walls</li> <li>Stiffeners attached to longitudinal wash bulkheads</li> <li>Stiffeners attached to sloping plates above and below longitudinal bulkheads / tank side walls</li> </ul>	System A (3)	$\frac{M_{slh=r}}{(4.8,2.4-5(1))}$
<ul> <li>Stiffeners attached to vertical girders which are attached to transverse bulkheads</li> <li>Stiffeners attached to horizontal girders which are attached to longitudinal bulkheads / tank side walls</li> <li>Stiffeners attached to cross-tie of longitudinal direction</li> </ul>	System B (4)	$\frac{M_{slh-r}}{(4.8.2.4-5(2))}$

#### Notes:

Numbers in parentheses indicate section number.

- (1)  $M_{slh-p}$  applies to stiffeners attached to plate panels within the range of  $0.3\ell_{tk}$  from transverse bulkheads / front and aft walls of tank, while  $M_{slh-r}$  applies to stiffeners attached to plate panels within the range of  $0.3b_{tk}$  from longitudinal bulkheads / tank side walls. Definitions of  $\ell_{tk}$  and  $b_{tk}$  are specified in Table 4.8.2-13 and Table 4.8.2-14.
- (2) Notwithstanding (1) above, where large sloping plates (such as plates consisting of top side tanks) are arranged between tank top plates and longitudinal bulkheads /tank side walls, tank top plates may be excluded from the members to be assessed.
- (3) See Fig. 10.9.3-1
- (4) See Fig. 10.9.3-2



## 10.9.4 Webs of Girders

 $\bigcirc$ 

(a)

## 10.9.4.1

The web thickness  $t_w$  of girders on which sloshing loads act is to be not less than the value obtained from the following formula.

(b) Vertical girder

(c) Horizontal girder,

cross-tie

$$t_w = \frac{b}{2} \sqrt{\frac{P_{slh} \times 10^{-3}}{1.15 C_a \sigma_Y}} \ (mm)$$

 $P_{slh}$ : Equivalent pressure  $(kN/m^2)$  for the plate panels, as specified in Table 10.9.4-1

 $\underline{C_a}$ : Coefficient of axial force effect as specified in 10.9.2.1

 $\bigcirc$ 

Bulkhead

b: Length (mm) of the shorter side of the plate panel

Table 10.9.4-1 Equivalent Pressure for Each Member to be Assessed

Member to be assessed	$\underline{P_{slh}}$
- Horizontal girders attached to transverse bulkheads / front and aft walls of	
<u>tanks</u>	
<ul> <li>Horizontal girders attached to transverse wash bulkheads</li> </ul>	$\underline{P_{slh-p}}$
<ul> <li>Vertical girders attached to longitudinal bulkheads / tank side walls</li> </ul>	<u>(4.8.2.4-4(1))</u>
<ul> <li>Vertical girders attached to longitudinal wash bulkheads</li> </ul>	
<ul> <li>Cross-ties in transverse direction</li> </ul>	
- Horizontal girders attached to longitudinal bulkheads / tank side walls	
<ul> <li>Horizontal girders attached to longitudinal wash bulkheads</li> </ul>	
<ul> <li>Vertical girders attached to transverse bulkheads / front and aft walls of</li> </ul>	<u>P<sub>slh=r</sub></u>
<u>tanks</u>	(4.8.2.4-4(2))
<ul> <li>Vertical girders attached to transverse wash bulkheads</li> </ul>	
<ul> <li>Cross-ties in longitudinal direction</li> </ul>	
Notes:	
Numbers in parentheses indicate section number.	

## 10.9.5 Corrugated Bulkheads

## <u>10.9.5.1</u>

- 1 The thickness of flanges and webs of corrugated bulkheads is to be not less than the value specified in 10.9.2.1.
- 2 The section modulus of 1/2 pitch of vertically corrugated bulkheads is to be not less than the value obtained from the following formula.

$$Z = \frac{M_{slh}}{\sigma_V} \times 10^3 \, (cm^3)$$

M<sub>slh</sub>: Equivalent bending moment (kN-m), as specified in Table 10.9.5-1

3 Notwithstanding -1 and -2 above, horizontally corrugated bulkheads are to be as deemed appropriate by the Society.

Table 10.9.5-1 Equivalent Bending Moment for Each Member to be Assessed

Member to be assessed	$M_{slh}$		
Corrugated transverse bulkheads	$\frac{M_{sth-p}}{(4.8.2.4-6)}$		
Corrugated longitudinal bulkheads	$\frac{M_{sth-r}}{(4.8.2.4-6)}$		
Notes: Numbers in parentheses indicate section number.			

## Chapter 11 STRUCTURES OUTSIDE CARGO REGION

#### 11.2 Bow Construction

#### **11.2.1** General

Title of Paragraph 11.2.1.2 has been amended as follows.

#### 11.2.1.2 Net Scantlings Approach

Unless clearly indicated to be gross scantlings, the scantlings specified in this 11.2 are to be net scantlings.

#### 11.3 Superstructures and Deckhouses

#### 11.3.2 Superstructures

#### 11.3.2.3 Superstructure End Bulkheads

Sub-paragraph -2 has been amended as follows.

2 The section modulus of the stiffener on superstructure end bulkhead (gross scantlings) is not to be less than that obtained by the following formula:

$$Z_{gr} = 350 s P_{GW} \ell^2 \times 10^{-3} \times 10^{-6} \text{ (cm}^3)$$

s and  $P_{GW}$ : As specified in -1(1) above.

 $\ell$ : Distance (m) between the decks at the position. However, when the value is less than 2m,  $\ell$  is taken as 2m.

#### 11.5 Stern Construction

#### 11.5.1 Stern Frames

#### 11.5.1.1 General

Sub-paragraph -3 has been amended as follows.

- 1 The requirements in this 11.5.1 apply only to stern frames without rudder posts.
- 2 Stern frames may be fabricated from steel plates or made of cast steel with a hollow section. For applicable material specifications and steel grades (*See 3.2*). Stern frames of other material or construction will be specially considered.
- Cast steel and fabricated stern frames are to be strengthened by adequately spaced horizontal plates with gross thicknesses not less than 80 % of the required thickness for stern frame  $t_1$ . Where the radius of curvature is large, a centreline stiffener is to be provided, and  $t_1$  is to be according to Table 11.5.1-1 or Table 11.5.1-2. Abrupt changes of section are to be avoided in castings, and all sections are to have an adequate tapering radius.
- 4 The scantlings specified in this 11.5.1 are to be gross scantlings.

## Part 2-1 CONTAINER CARRIERS

## Chapter 4 LOADS

## 4.2 Loads to be Considered in Longitudinal Strength

## 4.2.2 Maximum Load Condition

## 4.2.2.2 Vertical Still Water Bending Moment and Vertical Still Water Shear Force

Table 4.2.2-3 has been amended as follows.

Table 4.2.2-3 Vertical Wave Bending Moments  $M_{WV-h}$  and  $M_{WV-s}$  for Torsional Strength Assessments

101 TOTSIONA	i bucingui risse.	SSITCHES
The vertical wave bending moment in hogging condition	$M_{WV-h}$ (kN-m)	$M_{WV-h} = 1.5C_R C_{Vh} L_C^3 C C_W \left(\frac{B}{L_C}\right)^{0.8} C_{NL-h}$
The vertical wave bending moment in sagging condition	$M_{WV-s}$ (kN-m)	$M_{WV-s} = 1.5C_R C_{Vs} L_C^{\ 3} C C_W \left(\frac{B}{L_C}\right)^{0.8} C_{NL-s}$
Notes: $C_R$ , $C_{NL-h}$ , $C_{NL-s}$ , $C$ : As specified in Table 4.2.2-1.		

 $C_{Vh}$ ,  $C_{Vs}$ : Distribution coefficient, as given by the following formulae, to be taken as 0 in the ranges of  $x/L_C < 0$  and  $x/L_C > 1$  (See Fig. 4.2.2-3)

$$C_{Vh} = 3.165 \cos \left(0.1\pi \left(\frac{x}{L_C} + 3.5\right)\right) \sin^2 \left(\pi \frac{x}{L_C}\right)$$

$$C_{Vs} = -3.165 \cos \left(0.1\pi \left(\frac{x}{L_C} + 3.5\right)\right) \sin^2 \left(\pi \frac{x}{L_C}\right)$$

## 4.2.2.6 Horizontal Bending Moment and Torsional Moment

Table 4.2.2-6 has been amended as follows.

Table 4.2.2-6 Horizontal Bending Moment and Torsional Moment when Assessing Torsional Strength by Whole Ship Analysis

Strength by whole Ship Analysis			
Horizontal wave bending	$M_{WH1}$ (kN-m)	$M_{WH1} = 0.32C_R C_1 C_{H1} L_C^2 T_{SC} \sqrt{\frac{L_C - 35}{L_C}}$	
moments	$M_{WH2}$ (kN-m)	$M_{WH2} = 0.32C_R C_1 C_{H2} L_C^2 T_{SC} \sqrt{\frac{L_C - 35}{L_C}}$	
Scill 4 4 1 1	$M_{ST1}$ (kN-m)	$M_{ST1} = M_{ST\_max} \cdot C_{T1}$	
Still water torsional moments	$M_{ST2}$ (kN-m)	$M_{ST2} = M_{ST\_max} \cdot C_{T2}$	
	$M_{WT1}$ (kN-m)	$M_{WT1} = C_R C_{T1} \cdot [1.3C_1 L_C T_{SC} C_B (0.65T_{SC} + e) + 0.2C_1 L_C B^2 C_W]$	
Wave torsional moments	$M_{WT2}$ (kN-m)	$M_{WT2} = C_R C_{T2} \cdot [1.3C_1 L_C T_{SC} C_B (0.65T_{SC} + e) + 0.2C_1 L_C B^2 C_W]$	

Notes:

 $C_R$ : Coefficient considering the effect of ship operation, taken as 0.85

 $C_{H1}$ ,  $C_{H2}$ : Distribution coefficient, as given by the following formulae, to be taken as 0 in the ranges of  $x/L_C < 0$  and  $x/L_C > 1$  (See Fig. 4.2.2-6)

$$C_{H1} = -\cos\left(0.77\pi \left(\frac{x}{L_{C}} - 0.52\right)\right) \cdot \sin^{2}\left(\pi \frac{x}{L_{C}}\right) \cdot \left[\frac{1 - \exp(-6x/L_{C})}{1 - \exp(-3)}\right]$$

$$C_{H2} = -\sin\left(0.77\pi \left(\frac{x}{L_{C}} - 0.52\right)\right) \cdot \sin^{2}\left(\pi \frac{x}{L_{C}}\right) \cdot \left[\frac{1 - \exp(-6x/L_{C})}{1 - \exp(-3)}\right]$$

 $M_{ST\_max}$ : Largest value of the permissible maximum still water torsional moment in the longitudinal direction as described in loading manual (kN-m).

 $C_{T1}$ ,  $C_{T2}$ : Distribution coefficient, as given by the following formulae, to be taken as 0 in the ranges of  $x/L_C < 0$  and  $x/L_C > 1$  (See Fig. 4.2.2-7)

$$C_{T1} = -1.0 \left[ \sin \left( 2\pi \frac{x}{L_C} \right) + 0.1 \sin^2 \left( \pi \frac{x}{L_C} \right) \right] \cdot \exp \left( -0.35 \frac{x}{L_C} \right) \cdot \exp \left( -8 \left( 2\frac{x}{L_C} - 1 \right)^{10} \right)$$

$$C_{T2} = -0.5 \left[ -\sin \left( 3\pi \frac{x}{L_C} \right) + 0.65 \sin^3 \left( \pi \frac{x}{L_C} \right) \right] \cdot \exp \left( -0.4 \frac{x}{L_C} \right) \cdot \exp \left( -8 \left( 2\frac{x}{L_C} - 1 \right)^{10} \right)$$

e: Distance from baseline to shear centre at the midship cross section  $(m)^{(1)}$ 

<sup>(1)</sup> The position of the shear centre can be calculated by finding the point of action of the shearing force where a torsional moment does not occur in the cross section when the shearing force in the horizontal direction acts on the traverse section of the hull. For example, the position can be calculated by applying the requirements of Annex 5.2, Part 1 "Calculation Of Shear Flow CALCULATION OF SHEAR FLOW".

## 4.4 Loads to be Considered in Strength of Primary Supporting Structures

#### 4.4.2 Maximum Load Condition

#### **4.4.2.1** General

Table 4.4.2-1 has been amended as follows.

Table 4.4.2-1 Loads to be Considered in Maximum Load Condition

			Loading patterns			Difference between	
Structures to be assessed		Draught(m)	Vertical still water bending moment (kN-m)	Loaded to be considered	Equivalent design wave	external and internal pressure to be considered $(kN/m^2)$	
Double bottom	S1	$T_{SC}$	$M_{SV\;max}$	Container cargo	<i>HM</i> -1 / <i>HM</i> -2		
	S2	$T_{SC}$	$M_{SV\ min}$	Container cargo	DD 1D /	Double bottom: $P_{DB}$ Double side: $P_{DS}$	
Double side	S3	$T_{SC}$	$M_{SV\;min}$	None Container cargo	<i>BP-1P /</i> <i>BP-</i> 1S	<i>DS</i>	

#### **4.4.2.2** External Pressure

Table 4.4.2-2 has been amended as follows.

Table 4.4.2-2 External and Internal Pressure to be Considered

Structures to be assessed		$P_{DB} (kN/m^2)^{(1)} (2)$	$P_{DS} (kN/m^2)^{(1)}(2)$
Double bottom	S1	$P_{exs} + P_{exw} - P_{in\underline{s1}}$	$P_{exs} + P_{exw}$
	S2	$P_{exs} + P_{exw} - P_{in\_s1}$	$P_{exs} + P_{exw}$
Double side	S3	$P_{exs} + P_{exw} - P_{in\_s3}$	$P_{exs} + P_{exw}$

Notes:

 $P_{exs}$ ,  $P_{exw}$ : Hydrostatic and Hydrodynamic pressures  $(kN/m^2)$  acting on bottom shell in the case of  $P_{DB}$ , and the Those values acting on side shell in the case of  $P_{DS}$ . Each value is to be calculated in accordance with 4.6.2.4, Part 1.

 $P_{in\_S1}$ . The values considering the effect of container cargo  $(kN/m^2)$ , as given by the following formula: however, the said

values are to be taken as 0 when the number of bays in the cargo hold is only one.

$$\frac{P_{in\_S1} = 0.15\rho g T_{SC}}{P_{in\_S2\_S3} = 0.3\rho g T_{SC}}$$

(1) Load calculation points is are to be in accordance with 7.3.1.5, Part 1 for all loading conditions.

(2) When calculating loads,  $T_{LC} = T_{SC}$ .

## 4.6 Loads to be Considered in Fatigue

# 4.6.3 Loads to be Considered in Torsional Fatigue Strength Assessment by Whole Ship Analysis

#### 4.6.3.2 Hull Girder Loads

Table 4.6.3-1 has been amended as follows.

Table 4.6.3-1 Vertical Wave Bending Moments  $M_{WV-h}$  and  $M_{WV-s}$ 

10010 11010 1 10110001 11011	2 2 3 1 1 2 1 1 2 1 1	$\frac{1}{2} \frac{1}{V} \frac{1}{V} \frac{1}{V} = \frac{1}{V} \frac{1}{V} \frac{1}{V} \frac{1}{V} = \frac{1}{V} \frac{1}{V} \frac{1}{V} = \frac{1}{V} \frac{1}{V} \frac{1}{V} = \frac{1}{V} \frac{1}{V} \frac{1}{V} = \frac{1}{V} \frac{1}{V} \frac{1}{V} \frac{1}{V} \frac{1}{V} = \frac{1}{V} \frac{1}{V} \frac{1}{V} \frac{1}{V} \frac{1}{V} \frac{1}{V} \frac{1}{V} = \frac{1}{V} \frac{1}$
The vertical wave bending moment in hogging condition	$M_{WV-h}$ (kN-m)	$M_{WV-h} = 1.5C_{F\_WV}C_{Vh}L_{C}^{3}CC_{W}\left(\frac{B}{L_{C}}\right)^{0.8}$
The vertical wave bending moment in sagging condition	$M_{WV-s}$ (kN-m)	$M_{WV-s} = 1.5C_{F\_WV}C_{Vs}L_c^{\ 3}CC_W\left(\frac{B}{L_c}\right)^{0.8}$

Notes:

 $C_{F\ WV}$ : Fatigue coefficient, as given by the following formula.

 $C_{F\_WV} = C_{F1\_WV}C_{F2\_WV}C_{F3\_WV}$ 

 $C_{F1\_WV}$ : Coefficient considering speed effects, to be taken as 1.2.

 $C_{F2\_WV}$ : Conversion coefficient for the exceedance probability level to be considered in fatigue strength assessment, to be taken as 0.22

 $C_{F3\_WV}$ : Coefficient considering the effects of the loading condition, to be taken as 0.82.

 $C_{Vh}$ ,  $C_{Vs}$ : Distribution coefficient as given by the following formulae, to be taken as 0 in the ranges of  $x/L_C < 0$  and  $x/L_C > 1$  (See Fig. 4.6.3-1)

$$\begin{split} C_{Vh} &= 1.044 \sin \left(\pi \frac{x}{L_C}\right) \sin^2 \left(0.45 \pi \frac{x}{L_C} + 0.35 \pi\right) \sin^2 \left(0.6 \pi \frac{x}{L_C} + 0.21 \pi\right) \\ C_{Vs} &= -1.044 \sin \left(\pi \frac{x}{L_C}\right) \sin^2 \left(0.45 \pi \frac{x}{L_C} + 0.35 \pi\right) \sin^2 \left(0.6 \pi \frac{x}{L_C} + 0.21 \pi\right) \end{split}$$

C: Wave parameter, as specified in **Table 4.2.2-1**.

Table 4.6.3-3 has been amended as follows.

Table 4.6.3-3 Wave Torsional Moments  $M_{WT1}$  and  $M_{WT2}$ 

Wave torsional moment	$M_{WT1}$ (kN-m)	$M_{WT1} = C_{F\_WT}C_{T1} \cdot [1.3C_1L_CT_{SC}C_B(0.65T_{SC} + e) + 0.2C_1L_CB^2C_W]$
Wave torsional moment	$M_{WT2}$ (kN-m)	$M_{WT2} = C_{F\_WT}C_{T2} \cdot [1.3C_1L_CT_{SC}C_B(0.65T_{SC} + e) + 0.2C_1L_CB^2C_W]$

Notes:

 $C_{F\ WT}$ : Fatigue coefficient, as given by the following formula:

 $C_{F_{WT}} = C_{F1\_WT}C_{F2\_WT}C_{F3\_WT}$ 

 $C_{F1\ WT}$ : Coefficient considering speed effects, to be taken as 1.03.

 $C_{F2\ WT}$ : Conversion coefficient for the exceedance probability level to be considered, to be taken as 0.23

 $C_{F3\ WT}$ : Coefficient considering the effects of the loading condition, to be taken as 0.82.

 $C_{T1}$ ,  $C_{T2}$ : Distribution coefficients, as given by the following formulae, to be taken as 0 in the ranges of  $x/L_C < 0$  and  $x/L_C > 1$ :

$$C_{T1} = -1.1 \left[ \sin \left( 2\pi \frac{x}{L_C} \right) + 0.1 \sin^2 \left( \pi \frac{x}{L_C} \right) \right] \exp \left( -0.7 \frac{x}{L_C} \right) \exp \left( -8 \left( 2 \frac{x}{L_C} - 1 \right)^{10} \right)$$

$$C_{T2} = -0.5 \left[ -\sin \left( 3\pi \frac{x}{L_C} \right) + 0.8 \sin^3 \left( \pi \frac{x}{L_C} \right) \right] \exp \left( -0.6 \frac{x}{L_C} \right) \exp \left( -8 \left( 2 \frac{x}{L_C} - 1 \right)^{10} \right)$$

e: Distance from baseline to shear centre at the midship section (m), as specified in Table 4.2.2-6.

## Chapter 5 LONGITUDINAL STRENGTH

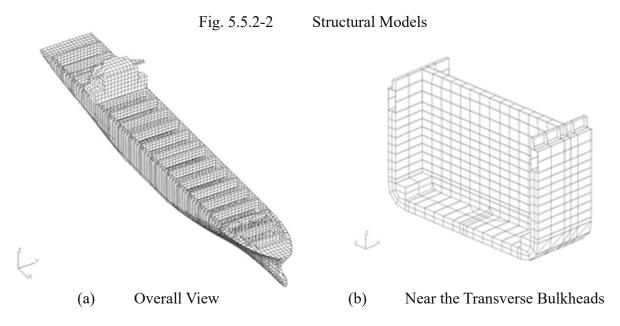
## 5.5 Torsional Strength

## 5.5.2 Torsional Strength Assessments Based on Finite Element Analysis

#### 5.5.2.3 Structural Models

Sub-paragraph -5 has been amended as follows.

- 5 The meshing of elements is to be in accordance with the following (1) and (2) (See Fig. 5.5.2-2):
- (1) The meshing of elements is to  $b\underline{B}e$  performed so as to reproduce the structural responses of the whole ship accurately. In principle, the meshing of elements is to be in accordance with the following (a) to (c):
  - (a) The size of mesh is not to be greater than floor space in general.
  - (b) The aspect ratio of shell elements is to be kept as close to 1 as possible.
  - (c) Variation of the mesh size and the use of triangular elements are to be kept to a minimum.
- (2) Separate finite element models that uses a finer mesh than usual may be used by applying the boundary conditions obtained from the structural model.



Paragraph 5.2.2.4 has been amended as follows.

#### 5.5.2.4 Load Conditions and Boundary Conditions

- 1 As load conditions, vertical still water bending moments, vertical wave bending moments, horizontal wave bending moments, and still water torsional moments caused by unbalanced loading of containers and wave torsional moments carrier are to be considered.
- When applying the load, it is not to affect the <u>axial</u> stress <u>caused by hull girder loads</u> at the point of reference. <del>Torsional moments, especially, are to be applied in accordance with the following (1) to (3) as standard:</del>
- (1) Torsional moments acting on hull girders are to be applied to structural model as a series of bulkhead torsional moments resulting in a stepped curve. An approximated torsional step

moment curve is shown in Fig. 5.5.2-3.

(2) Torsional moments applied to bulkheads are the net change in torsional moment over the effective range of the bulkhead. The effective range of a bulkhead is the distance between the midpoints of the two adjacent bulkheads. The torsional moments at bulkhead i (kN-m) are specified as the following formulae: (See Fig. 5.5.2-4).

$$\begin{array}{l} \delta M_{WT1t} = M_{WT1} |_{\frac{1}{2}(x_t + x_{t+1})} M_{WT1} |_{\frac{1}{2}(x_{t-1} + x_t)} \\ \delta M_{WT2t} = M_{WT2} |_{\frac{1}{2}(x_t + x_{t+1})} M_{WT2} |_{\frac{1}{2}(x_{t-1} + x_t)} \end{array}$$

x<sub>z</sub>: x-coordinate of bulkhead i

- (3) The torsional moment at each bulkhead is to be reproduced by two equivalent shear forces on each side. An example of a method for applying shear force is shown in Fig. 5.5.2-5.
- 3 Torsional moments are, as a standard, to be applied in accordance with the following (1) to (3):
- (1) Torsional moments acting on hull girders are to be applied to structural models as a series of bulkhead torsional moments resulting in a stepped curve. An approximated torsional step moment curve is shown in Fig. 5.5.2-3.
- (2) Torsional moments applied to bulkheads are the net change in torsional moment over the effective range of the bulkhead. The effective range of a bulkhead is the distance between the midpoints of the two adjacent bulkheads. The torsional moments at bulkhead *i* are specified as the following formulae (*See Fig. 5.5.2-4*):

$$\frac{\delta M_{WT1i} = M_{WT1}|_{\frac{1}{2}(x_i + x_{i+1})} - M_{WT1}|_{\frac{1}{2}(x_{i-1} + x_i)}}{\delta M_{WT2i} = M_{WT2}|_{\frac{1}{2}(x_i + x_{i+1})} - M_{WT2}|_{\frac{1}{2}(x_{i-1} + x_i)}}$$

 $x_i$ : x-coordinate of bulkhead i

- (3) The torsional moment at each bulkhead is to be reproduced by two equivalent shear forces on each side. An example of a method for applying shear force is shown in Fig. 5.5.2-5.
- When analysing the vertical and horizontal bending moments applied, a method applying unit moments is to be used as the standard. Stresses corresponding to the moments prescribed in 4.4.2 are to be calculated based on the stresses obtained through structural analysis with unit moments applied.
- Boundary conditions is to constrain the displacement of the structure at positions where the reaction force is considered to be small. Standard boundary conditions are to be given in Table 5.5.2-2.

Table 5.2.2-2 has been amended as follows.

Table 5.5.2-2 Boundary Conditions at Model Ends

			Translation			Rotation	
	Location	X direction	Y direction	Z direction	Around the <u>x</u> ¥-axis	Around the <u>y</u> ¥-axis	Around the <u>z</u> <b>Z</b> -axis
Boundary conditions of	Aft end of $L_c$	Fix	Fix	Fix	-	-	-
torsional moment (See Fig. 5.5.2-26 (a))	Fore end of $L_c$	-	Fix	Fix	-	-	-
Boundary conditions of	Aft end of $L_c$	Fix	Fix	Fix	1	-	1
vertical bending moment (See Fig. 5.5.2-37)	Fore end of $L_c$	-	Fix	Fix	-	-	-
	Aft end of $L_c$	Fix	Fix	<b>=</b> Fix	1	-	1
Boundary conditions of horizontal bending moment	Fore end of $L_c$	-Fix	<del>Fix</del>	-Fix	1	-	ı
(See Fig. 5.5.2-4 <u>8</u> )	Transverse bulkheads of $L_c$	<del>Fix</del> -	<del>Fix</del> -	Fix	-	-	-

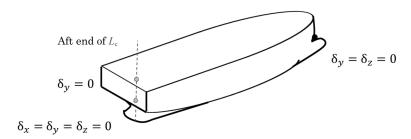
Notes:

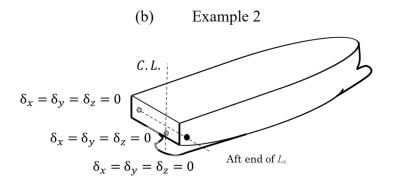
The position of a constraint is to be the position in which the reaction force is thought to be small.

This table may not be applied for some load applying methods and stress calculation methods

Fig. 5.5.2-2 has been renumbered to Fig. 5.5.2-6, and Fig. 5.5.2-7 and Fig.5.5.2-8 have been added as follows.

Fig. 5.5.2-<del>2</del>6 Boundary Conditions of Torsional Moment
(a) Example 1





<sup>[-]</sup> means no constraint applied (free).

Fig. 5.5.2-4 and Fig. 5.5.2-5 have been deleted.

Fig. 5.5.2-4 Boundary Conditions of Vertical Bending Moment

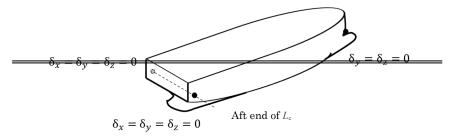


Fig. 5.5.2-7 Boundary Conditions and Loads Application of Vertical Bending Moment

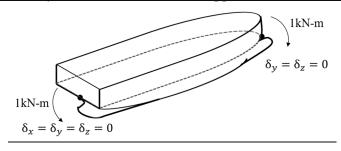


Fig. 5.5.2-5 Boundary Conditions of Horizontal Bending Moment

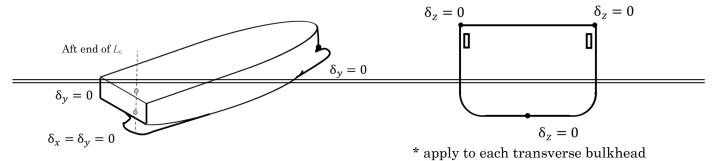
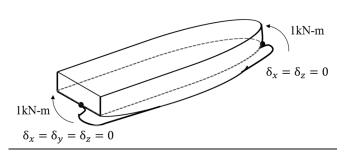
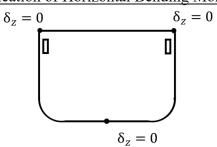


Fig. 5.5.2-8 Boundary Conditions and Loads Application of Horizontal Bending Moment





\* apply to each transverse bulkhead

## 5.5.2.9 Buckling Strength Assessment

Sub-paragraphs -1 and -2 have been amended as follows.

- 1 The requirements in **8.6.2.1**, **Part 1** are applied correspondingly for buckling assessments. However, the buckling permissible utilisation factor is to be taken as 0.9 for bottom shell plating, bilge strakes and longitudinal stiffeners attached to these members.
- 2 Notwithstanding the requirement in -1 above, bilge strakes longitudinally stiffened and longitudinal stiffeners attached to the bilge strakes may be verified in accordance with the following requirements (1) or (2) according to net thickness of bilge strakes and bilge radius:
- (1) In case where the bilge strake net thickness is not less than 14.5 mm and the bilge radius is not greater than 8 m, the following-requirements (a) and (b) are to be applied.
  - (a) The evaluation stress determined in accordance with 5.5.2.7 is not greater than 0.9 times the specified minimum yield stress of the steel used or  $3\frac{2}{2}$ 0 N/mm<sup>2</sup>, whichever is smaller.
  - (b) The following formula is satisfied:

$$\sqrt{11 \cdot \left(\frac{t}{1000R}\right)^2 + \left(\frac{\pi t}{1000S}\right)^4} + \left(\frac{\pi t}{1000S}\right)^2 \ge 0.014$$

- t: Bilge strake net plate thickness (mm)
- S: Spacing (m) of stiffeners. To be taken as the girth length.
- R: Bilge radius (m)
- (2) If the bilge strake net thickness is less than 14.5 mm and the bilge radius is greater than 8 m, the evaluated stress determined in accordance with 5.5.2.7 is not to be greater than 0.9 times the buckling strength obtained by using non-linear analysis, etc and other methods.

## Chapter 7 STRENGTH OF PRIMARY SUPPORTING STRUCTURES

#### 7.1 General

## 7.1.1 Application

#### 7.1.1.1

Sub-paragraph -2 has been renumbered to Sub-paragraph -3, and Sub-paragraph -2 has been added as follows.

- 1 The requirements of this Chapter apply to ships of less than 150 m in length  $L_c$ .
- 2 Notwithstanding -1 above, strength assessments for deck girders with respect to deck loads and green sea loads are to be carried out in accordance with this Chapter.
- For the double bottom and double-side skin structure, the requirements of the double hull structure specified in 7.3, Part 1 are to be applied. For other girder members that can be regarded as simple girders, the requirement of the simple girder specified in 7.2, Part 1 are to be applied.

#### 7.2 Double Hull Structure

#### **7.2.1** General

Paragraph 7.2.1.1 has been amended as follows.

#### 7.2.1.1 Handling of Partial Bulkheads in the Hold

In applying 7.3, Part 1, the length between the watertight bulkheads is to be assessed as the length of the cargo hold regardless of whether there are partial bulkheads in the middle of the hold. When assessing in consideration of the influence of the partial bulkheads in the middle of the hold, the strength is to be assessed by the cargo hold analysis specified in Chapter 8. Girders near partial bulkheads are to ensure sufficient strength to account for shear force effects.

#### Chapter 8 STRENGTH ASSESSMENT BY CARGO HOLD ANALYSIS

#### 8.6 Strength Assessment

## **8.6.1** Yield Strength Assessment and Buckling Strength Assessment

Paragraph 8.6.1.3 has been added as follows.

#### 8.6.1.3 Strength Assessments of Transverse Bulkheads in Flooded Condition

Where strength assessments are performed in the flooded condition, girders attached to transverse bulkheads are to satisfy the criteria for yield strength assessment and buckling strength assessment. Platings of bulkheads, however, only need to satisfy the criteria for yield strength assessment.

## Chapter 9 FATIGUE

## 9.3 Torsional Fatigue Strength Assessment

## 9.3.4 Boundary Conditions and Load Conditions

## 9.3.4.1 Boundary Conditions

- 1 Boundary conditions are to be set to appropriately reproduce the structural responses of the whole ship model in consideration of the extent of model and loads to be considered, etc.
- 2 Boundary conditions which do not generate torsional deformation are to be given when calculating stress due to horizontal bending moments.
- 3 The boundary conditions for torsional moment is to constrain the translational and rotational displacements of the positions where the reaction force in the port and starboard model is thought to be small.
- 4 The standard boundary conditions are in accordance with the following (1) to (3):
- (1) The boundary conditions for the standard torsional moment are as shown in Fig. 9.3.4-1.
- (2) The boundary conditions for the standard vertical bending moment are as shown in **Fig. 9.3.4-2**.
- (3) The boundary conditions for the standard horizontal bending moment are as shown in **Fig. 9.3.4-3**.

Fig. 9.3.4-2 has been amended as follows.

Fig. 9.3.4-2 Boundary Conditions of Vertical Bending Moments and Load Conditions

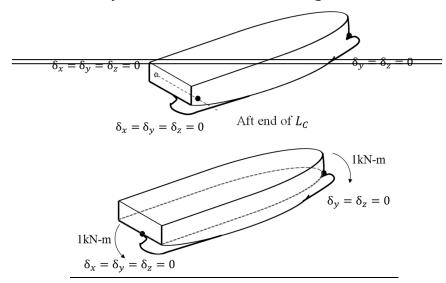
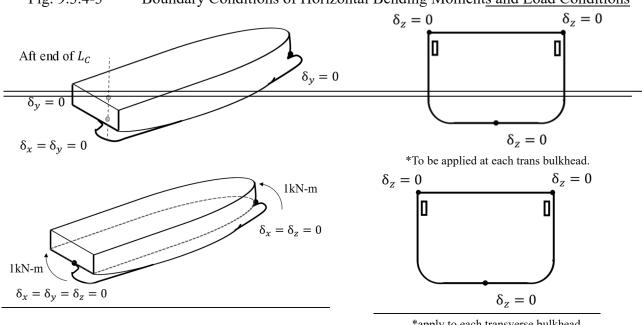


Fig. 9.3.4-3 has been amended as follows.

Fig. 9.3.4-3 Boundary Conditions of Horizontal Bending Moments and Load Conditions



\*apply to each transverse bulkhead.

#### 9.3.4.2 **Load Conditions**

Sub-paragraph -4 has been added as follows.

1 The loads to be used in the torsional fatigue strength assessments are to be in accordance with **4.6.3**.

2 Stresses due to vertical bending moment, horizontal bending moment and torsional moment are calculated based on structural analysis using the whole ship model.

3 Torsional moments are to be applied to structural models in accordance with the following (1) to (3):

- Torsional moments acting on hull girders are to be applied to structural model as a series of (1) bulkhead torsional moments resulting in a stepped curve. An approximated torsional step moment curve is shown in Fig. 9.3.4-4.
- Torsional moments applied to bulkheads are the net change in torsional moment over the (2) effective range of the bulkhead. The effective range of a bulkhead is the distance between the midpoints of the two adjacent bulkheads. The torsional moments at bulkhead i(kN-m) are specified as the following formulae: (See Fig. 9.3.4-5)

$$\begin{split} \delta M_{WT1i} &= M_{WT1}|_{\frac{1}{2}(X_i + X_{i+1})} - M_{WT1}|_{\frac{1}{2}(X_{i-1} + X_i)} \\ \delta M_{WT2i} &= M_{WT2}|_{\frac{1}{2}(X_i + X_{i+1})} - M_{WT2}|_{\frac{1}{2}(X_{i-1} + X_i)} \end{split}$$

 $X_i$ : X-coordinate of bulkhead i

- The torsional moment at each bulkhead is to be reproduced by two equivalent shear forces on (3) each side. An example of a method for applying shear force is shown in Fig. 9.3.4-6.
- When analysing the vertical and horizontal bending moments applied, a method applying unit moments is to be used as the standard. Stresses corresponding to the moments prescribed in 4.6.3.2 are to be calculated based on the stresses obtained through structural analysis with unit moments applied. (See Fig.9.3.4-2 and Fig.9.3.4-3)

## Part 2-2 BOX-SHAPED BULK CARRIERS

## Chapter 7 PRIMARY SUPPORTING STRUCTURE STRENGTH

#### 7.1 General

## 7.1.1 Application

#### 7.1.1.1

Sub-paragraph -2 has been renumbered to Sub-paragraph -3, and Sub-paragraph -2 has been added as follows.

- 1 The requirements of this Chapter apply to ships of less than 150 m in ship length  $L_C$ .
- 2 Notwithstanding -1 above, strength assessments for deck girders with respect to deck loads and green sea loads are to be carried out in accordance with this Chapter.
- **23** For the double bottom and double-side skin construction, the requirements of the double hull structure specified in **7.3**, **Part 1** are to be applied. For other girder members that can be regarded as simple girders, the requirement of the simple girder specified in **7.2**, **Part 1** are to be applied.

# Chapter 8 STRENGTH ASSESSMENT BY CARGO HOLD ANALYSIS

#### 8.5 Strength Assessment

## 8.5.1 Yield Strength Assessment and Buckling Strength Assessment

Paragraph 8.5.1.3 has been added as follows.

#### 8.5.1.3 Strength Assessments of Plane-type Transverse Bulkheads

- 1 Where strength assessments are performed in the maximum load condition, 8.6.2.1-2, Part 1 may be applied instead of the assessment under compressive loads in shorter side direction specified in 8.6.2.1-1, Part 1, for plate panels of plane-type transverse bulkheads of vertically stiffened system (See Table 8.5.1-1).
- 2 In the application of -1 above, the yield strength assessments specified in 8.6.1, Part 1 need not be carried out for the said plane-type transverse bulkheads.
- 3 In the application of -1 above, where the yield strength assessments and buckling strength assessments specified in An2.7, Annex 8.6A, Part 1 "Strength Assessment Considering Effect of Surrounding Structures" are carried out, such yield strength and buckling strength assessments need not be required for the following (1) to (3).
- (1) Plate panels within the rigidity reduction range
- (2) Buckling panels which include elements sharing nodes included in the elements of (1) above
- (3) Elements which include (1) and (2) above

Table 8.5.1-1 has been amended as follows.

reduction ranges are excluded.

Table 8.5.1-1 Relationship between the application of Part 1 and Part 2

M. J. d. J.	Maximum loc	ad condition
Member to be assessed	Equivalent design wave HM and FM	Equivalent design wave BR and BP
Side shell (Annex 8.6A, Part 1 is applied for equivalent design wave BR and BP)  Members other than side shell (Annex 8.6A, Part 1 is applied for equivalent design wave BR and BP)	Yield strength assessment:     As specified in 8.6.1, Part 1     Buckling strength assessment:     As specified in 8.6.2.1-1, Part 1	-An2.2 to An2.6 of Annex 8.6A, Part 1 are appliedPermissible utilization factor (buckling): 0.8 - Yield strength assessment: N.4 -An2.7 of Annex 8.6A, Part 1 is applied Yield strength assessment: As specified in 8.6.1, Part 1 Buckling strength assessment: As specified in 8.6.2.1 1, Part 1.
<del>Cross deck</del>	- Yield strength assessment:  As specified in 8.6.1, Part 1: - Buckling strength assessment:  As specified in 8.5.1.2.	Yield strength assessment:     As specified in 8.6.1, Part 1.     Buckling strength assessment:     As specified in 8.6.2.1-1, Part 1.

Paragraph 2	Member to be assessed	Maximum load condition	
		Equivalent design waves $HM$ and $FM$	Equivalent design waves BR and BP
<u>8.5.1.1</u>	Side shells within rigidity reduction ranges	• Yield strength assessment:  As specified in 8.6.1, Part 1 • Buckling strength assessment:  As specified in 8.6.2.1-1, Part 1	• An2.2 to An2.6, Annex 8.6A, Part 1 apply • Permissible utilisation factor (buckling): 0.8 • Yield strength assessment: NA
	Other members <sup>(1)</sup>		• An2.7, Annex 8.6A, Part 1 applies • Yield strength assessment:  As specified in 8.6.1, Part 1. • Buckling strength assessment:  As specified in 8.6.2.1-1, Part 1
8.5.1.2	Cross decks	• Yield strength assessment:  As specified in 8.6.1, Part 1  • Buckling strength assessment:  As specified in 8.5.1.2	• Yield strength assessment:  As specified in 8.6.1, Part 1  • Buckling strength assessment:  As specified in 8.6.2.1-1, Part 1
8.5.1.3	Plane-type transverse bulkheads within rigidity reduction ranges	<ul> <li>An2.2 to An2.6, Annex 8.6A, Part 1 apply</li> <li>Permissible utilisation factor (buckling): 0.8</li> <li>Yield strength assessment: NA</li> </ul>	
	Other members <sup>(1)</sup>	<ul> <li>• An2.7, Annex 8.6A, Part 1 applies</li> <li>• Yield strength assessment: As specified in 8.6.1, Part 1</li> <li>• Buckling strength assessment: As specified in 8.6.2.1-1, Part 1</li> </ul>	

(1) Where 8.5.1.1 and 8.5.1.3 are applied simultaneously for the equivalent design waves BR and BP, members within rigidity

# Annex 1.1 ADDITIONAL REQUIREMENTS FOR BULK CARRIERS IN CHAPTER XII OF THE SOLAS CONVENTION

#### An3 Transverse Watertight Bulkheads in Cargo Holds

#### An3.2 Load Conditions

#### An3.2.1

Table An6 has been amended as follows.

Table An6 Pressure  $P_{bs}$  and Force  $F_{bs}$  Acting on the Corrugated Bulkheads by Due to Bulk Cargo

The pressure $P_{bs}$ ( $kN/m^2$ ) at each point of the bulkhead	$P_{bs} = \rho_c g K_{c-f}(z_C - z)$
The force $F_{bs}$ (kN) acting on the corrugated bulkhead	$F_{bs} = \rho_c g S_1 \frac{(z_C - h_{DB} - h_{LS})^2}{2} K_{c-f}$

#### Notes:

 $\rho_c$ : Bulk cargo density  $(t/m^3)$ 

 $K_{c-f}$ : Coefficient, as given by the following formula:

$$K_{c-f} = \tan^2\left(45 - \frac{\psi}{2}\right)$$

$$K_{c-f} = \tan^2\left(45^\circ - \frac{\psi}{2}\right)$$

 $\psi$ : As specified in **Table An2**.

 $z_C$ : Distance (m) from the baseline to the horizontal plane corresponding to the top of the cargo when levelled out (m) (See Fig. An1)

 $h_{DB}$ : Height (m) of the double bottom

 $S_1$ : Spacing (m) of corrugation

 $h_{LS}$ : Height (m) of the lower stool from the inner bottom plating

## **An4** Allowable Hold Loading on Double Bottom

## An4.3 Strength Criteria

#### An4.3.1

Sub-paragraphs -1 and -3 have been amended as follows.

1 Shear capacity of double bottoms  $C_h$  and  $C_e$  are to comply with the following formulae:

$$\begin{array}{l} C_{\mathbf{k}} = Z \cdot A_{\mathbf{DB,h}} \cdot (kN) \cdot \underline{C}_{h} = P_{FD-db} \cdot A_{DB,h}(kN) \\ C_{\mathbf{e}} = Z \cdot A_{\mathbf{DB,e}} \cdot (kN) \cdot \underline{C}_{e} = P_{FD-db} \cdot A_{DB,e}(kN) \end{array}$$

The variables in the above formulae are to be in accordance with the following -2 to -4.

- 2 (Omitted)
- The load  $P_{FD-db}$  (N/mm<sup>2</sup>) (kN/m<sup>2</sup>) acting on the double bottom form the flooded hold condition is to be obtained from **Table An11**. In this such cases, the flooding head  $z_F$  (m) under consideration is to be the vertical distance from the baseline in the absence of trim or heel, according to **Table An12**.

#### Table An11 has been amended as follows.

Table An11 Load  $P_{FD-db}$  Acting on Double Bottoms  $\underline{\text{In}}\underline{\text{When}}$  Flooding Occurs

Flooding case	Load $P_{FD-db}(\frac{N/mm^2}{2})$			
$z_F > z_C$	$P_{FD-db} = \rho g[(z_C - h_{DB})(perm - 1) - E + (z_F - h_{DB})] + \rho_c g(z_C - h_{DB})$			
$z_F \leq z_C$	$P_{FD-db} = \rho_c g(z_C - h_{DB}) - \rho g[E - (z_F - h_{DB})perm]$			

Notes:

 $z_F$ : As specified in 4.2.1.5 1. Table An12

 $z_C$ ,  $z_{\overline{b}}$ . As specified in Table 4.2.1 3. And

 $\rho_c$ : As specified in Table 4.2.1 1. Bulk cargo density  $(t/m^3)$ . The density for steel is to be used for steel mill products<sup>(1)</sup>.

perm: As specified in Table 4.2.1 3An1.2.1(7). 0 for steel mill products.

 $V_C$ : Cargo volume  $(m^3)$  in for each cargo hold as given by the following formula:  $V_C = \frac{FW}{\rho_C}$ 

F: Coefficient to be taken as 1.1. However, it is to be taken as 1.05 for steel mill products.

W: Cargo mass (t) loaded in each hold

E: Ship immersion (m) for flooded hold condition as given by the following formula;

 $E = z_F - 0.1D$ 

<sup>(1)</sup> The load in the cargo hold is determined by the sum of the weight of the steel products and the weight of the flooded load, which is obtained by determining the volume of steel products in the cargo hold using the actual density of the steel mill products being loaded, and it is to be assuminged that seawater enters the void spaces remaining in the cargo hold at a flooding rate of 0.95; in cases where the flooding rate for the volume of steel products being considered is assumed to be 0.

#### Part 2-3 ORE CARRIERS

#### **Chapter 10 ADDITIONAL STRUCTURAL REQUIREMENTS**

Section 10.5 has been renumbered to Section 10.6, and 10.5 has been added as follows.

#### 10.5 Tank Structures for Sloshing

#### **10.5.1 General**

#### **10.5.1.1 Application**

- 1 For the members of ballast tank structures which satisfy the following (1) to (3), the scantlings specified in this 10.5 are considered to be satisfied when obtained using the sloshing loads specified in 4.8.2.4, Part 1.
- (1) Ballast tanks with volumes of not less than  $100 m^3$
- (2) Ballast tanks designed for possible to loading at filling ratios of not less than 20 % and not more than 90 %
- Where the period of longitudinal oscillation of ballast tanks is within the range of  $\pm 20$  % of pitch period and within  $\pm 1.5$  seconds from the same period, and where the period of transverse oscillation of ballast tanks is within the range of  $\pm 20$  % of roll period and within  $\pm 1.5$  seconds from the same period.
- 2 In applying -1(3) above, where only one of the conditions is applicable, only the sloshing load due to the relevant ship motion need be considered.
- In applying -1(3) above, the tank natural periods are to be calculated for each 10 % of the filling ratio, and only the sloshing load due to the filling ratio corresponding to the conditions of -1(3) above need be considered.

#### 10.5.1.2 Scantling Approach

The required scantlings specified in this **10.5** are to be net scantlings.

#### **10.5.2** Plates

#### 10.5.2.1

- The plate thickness on which sloshing loads act is to be not less than the value obtained from **10.9.2. Part 1**.
- Equivalent pressure for members to be assessed is to be in accordance with **Table 10.5.2-1**. The density of seawater  $(1.025 \ t/m^3)$ , however, is to be considered instead of maximum design cargo density when calculating such pressure.

Table 10.5.2-1 Equivalent Pressure for Plate Panels

Member	P <sub>slh</sub>
- Transverse bulkheads - Transverse wash bulkheads - Tank top plates near transverse bulkheads (1)	P <sub>slh-p</sub> (4.8.2.4-4(1), Part 1)
<ul> <li>Longitudinal bulkheads</li> <li>Side shells</li> <li>Tank top plates near longitudinal bulkheads / side shells<sup>(1)</sup></li> <li>Sloping plates below longitudinal bulkheads<sup>(2)</sup></li> </ul>	<u>P<sub>slh-r</sub></u> (4.8.2.4-4(2), Part 1)
Notes:	

Numbers in parentheses indicate section number.

- (1)  $P_{sth-p}$  is to be applied to the plate panels within the range of  $0.3\ell_{tk}$  from transverse bulkheads. In addition,  $P_{sth-r}$  is to be applied to the plate panels within the range of  $0.3\ell_{tk}$  from longitudinal bulkheads. The definitions of  $\ell_{tk}$  and  $\ell_{tk}$  are as specified in Table 4.8.2-13 and Table 4.8.2-14, Part 1.
- (2) For sloping plates below longitudinal bulkheads, sloshing loads are to be calculated using the same parameters as those used for longitudinal bulkheads.

#### 10.5.3 Stiffeners

#### <u> 10.5.3.1</u>

- 1 The section modulus of stiffeners attached to plates on which sloshing loads act is to be not less than the value obtained from the formula specified in 10.9.3, Part 1.
- 2 Equivalent bending moments for members to be assessed are to be in accordance with **Table 10.5.3-1**. The density of seawater  $(1.025 \ t/m^3)$  is to be considered instead of maximum design cargo density when calculating such moments.

Table 10.5.3-1 Equivalent Bending Moment for Each Member to be Assessed

Member to be assessed	Stiffened system	$M_{slh}$			
	Longitudinal	$M_{slh-p}(4.8.2.4-5(1), Part 1)$ $M_{slh-r}(4.8.2.4-5(2), Part 1)$			
<ul> <li>Stiffeners attached to tank top plates<sup>(1)</sup></li> </ul>	<u>Transverse</u>	$M_{slh-p}$ (4.8.2.4-5(2), Part 1) $M_{slh-r}$ (4.8.2.4-5(1), Part 1)			
<ul> <li>Stiffeners attached to transverse bulkheads</li> <li>Stiffeners attached to transverse wash bulkheads</li> <li>Stiffeners attached to vertical girders which are attached to</li> </ul>	System A <sup>(2)</sup>	$M_{slh-p}$ (4.8.2.4-5(1), Part 1)			
longitudinal bulkheads     Stiffeners attached to horizontal girders which are attached to transverse bulkheads     Stiffeners attached to cross-tie of transverse direction	System B <sup>(3)</sup>	$M_{slh=p}(4.8.2.4-5(2), Part 1)$			
<ul> <li>Stiffeners attached to longitudinal bulkheads</li> <li>Stiffeners attached to side shells</li> <li>Stiffeners attached to sloping plates below longitudinal bulkheads</li> </ul>	System A (2)	<u>M<sub>slh-r</sub></u> (4.8.2.4-5(1), Part 1)			
<ul> <li>Stiffeners attached to vertical girders which are attached to transverse bulkheads</li> <li>Stiffeners attached to horizontal girders which are attached to longitudinal bulkheads</li> <li>System B (3)</li> <li>(4.8.2.4-5(2), Part 1)</li> </ul>					
Notes: Numbers in parentheses indicate section number.					
<ol> <li>M<sub>sth=p</sub> is to be applied to the stiffeners attached to the plate panels within the range of 0.3ℓ<sub>tk</sub> from transverse bulkheads. And M<sub>sth=r</sub> is to be applied to the stiffeners attached to the plate panels within the range of 0.3b<sub>tk</sub> from longitudinal bulkheads / side shells.</li> <li>See Fig. 10.9.3-1, Part 1</li> </ol>					

#### 10.5.4 Webs of Girders

(3) See Fig. 10.9.3-2, Part 1

#### 10.5.4.1

- The web thickness  $t_w$  of girders on which sloshing loads act is to be not less than the value obtained from the formula specified in 10.9.4, Part 1.
- 2 Equivalent pressure for members to be assessed is to be in accordance with **Table 10.5.4-1**.

Table 10.5.4-1 Equivalent Pressure for Each Member to be Assessed

1able 10.5.4-1 Equivalent Hessure for Each Member to be Assessed			
Member to be assessed	<u>P<sub>slh</sub></u>		
<ul> <li>Horizontal girders attached to transverse bulkheads / transverse wash bulkheads</li> <li>Vertical girders attached to longitudinal bulkheads</li> <li>Vertical girders attached to side shells</li> <li>Cross-ties in transverse direction</li> </ul>	<u>P<sub>slh-p</sub></u> (4.8.2.4-4(1), Part 1)		
<ul> <li>Horizontal girders attached to longitudinal bulkheads</li> <li>Vertical girders attached to transverse bulkheads / transverse wash bulkheads</li> <li>Horizontal girders attached to side shells</li> </ul>	<u>P<sub>slh-r</sub></u> (4.8.2.4-4(2), Part 1)		
Notes: Numbers in parentheses indicate section number.			

#### 10.<u>56</u> Other

### 10.<u>56.1</u> Special Requirements for Ship Intended for the Carriage of Cargoes Having Moisture Contents Which Exceed Transportable Moisture Limit

#### 10.<del>5</del>6.1.1

The hull structural members of ships intended for the carriage of cargoes having moisture contents which exceed transportable moisture limit are to be in accordance with the following (1) or (2).

- (1) For ships intended for the carriage of nickel ore with a moisture content that exceeds the transportable moisture limit, the requirements specified in "Guidelines for the Safe Carriage of Nickel Ore"
- (2) For ships intended for the carriage of cargoes other than nickel ore, evaluation methods deemed appropriate by the Society

## Part 2-5 GENERAL CARGO SHIPS AND REFRIGERATED CARGO SHIPS

#### Chapter 4 LOADS

#### 4.3 Loads to be Considered in Strength of Primary Supporting Structures

#### 4.3.2 Maximum Load Condition

Table 4.3.2-2 has been amended as follows.

Table 4.3.2-2 External and Internal Pressure to be Considered

Structures to be assessed		$P_{DB}(kN/m^2)^{(1)}(2)$	$P_{DS}(kN/m^2)^{(1)}\frac{(2)}{(2)}$	
Double bottom	<i>S</i> 1	$P_{exs} + P_{exw}$	$P_{exs} + P_{exw}$	
D 11 '1	S2	$P_{exs} + P_{exw} - P_{in\_s2}$	$P_{exs} + P_{exw}$	
Double side	S3	$P_{exs} + P_{exw} - P_{in\_s3}$	$P_{exs} + P_{exw}$	

Notes:

 $P_{exs}$ ,  $P_{exw}$ : Hydrostatic and Hydrodynamic pressure  $(kN/m^2)$  act on bottom shell in case of  $P_{DB}$ . Those values act on side shell

in case of  $P_{DS}$ . Each value is calculated in accordance with **4.6.2.4**, Part 1.

 $P_{in s2}$ ,  $P_{in s3}$ : loads considering the effect of cargo  $(kN/m^2)$ , as given by the formulae:

 $\begin{aligned} P_{in\_S2} &= 0.5 \rho g T_{SC} \\ P_{in\_S3} &= \rho g T_{SC} \end{aligned}$ 

(1) Load calculation points <u>ware to be</u> in accordance with **7.3.1.5**, **Part 1** for all loading conditions.

When calculating loads,  $T_{LC} = 0.7T_{SC}$  for S1 and  $T_{LC} = T_{SC}$  for S2 and S3.

#### 4.4 Loads to be Considered in Additional Structural Requirements

#### 4.4.2 Maximum Load Condition

#### **4.4.2.1 Steel Coils**

1 (Omitted)

2 The total load  $F_{SC}$  (kN) of the steel coil acting on the hull is to be calculated by the following formula. However, it is to not be less than 0.

 $F_{SC} = F_{SCs} + F_{SCd}$ 

 $F_{SCs}$ : Static load (kN), as specified in **Table 4.4.2-1**.

 $F_{SCd}$ : Dynamic load (kN), as specified in **Table 4.4.2-2**.

Table 4.4.2-1 has been amended as follows.

Table 4.4.2-1 Static Load of Steel Coil  $F_{SCs}$ 

10010 11 12 1 2 1001 2 11 1 3 1 3 1 3 1 3 1 3 1 3 1 3 1 3 1					
Members	$n_2$ and $n_3$	$F_{SCS}$ $(kN)$			
I	$n_2 \le 10$ and $n_3 \le 5$	$C_{SC1}W_{SC}\frac{n_1n_2}{n_3}g$			
Inner bottom plating	$n_2 > 10 \text{ or } n_3 > 5$	$C_{SC1}W_{SC}n_1rac{\ell}{\ell_{st}}g$			
	$n_2 \le 10$ and $n_3 \le 5$	$C_{SC2}W_{SC}\frac{n_2}{n_3}g\cdot\cos\alpha$			
Hopper tank sloping	$n_2 > 10 \text{ or } n_3 > 5$	$C_{SC2}W_{SC}\frac{\ell}{\ell_{st}}g\cdot\cos\alpha$			
Longitudinal bulkheads and side frames	NA	0			

Notes:

 $n_1$ : Number of loading stages of steel coil

 $n_2$ : The load point per panel (the number of dunnages for a single panel), as specified in **4.4.2.2-3**.

 $n_3$ : Number of dunnage threads supporting one row of steel coils

 $W_{SC}$ : Mass of one steel coil (t)  $C_{SC1}$ : Coefficient as follows:

 $C_{SC1} = 1.4$  for multi-tiered stacking secured by a single-tiered loading secured with one or more key coils

 $C_{SC1} = 1.0$  for single-multi-tiered stacking loading or single-tired loading without key coils

 $C_{SC2}$ : Coefficient, as follows:

 $C_{SC2} = 3.2$  for single-tiered stacking or multi-tiered stacking in which the key coil is arranged in the second or third

position from the bilge tank sloping or inner hull

 $C_{SC2} = 2.0$  for all other cases

 $\ell$ : Distance between floors (m) (see Fig. 4.4.2-2)

 $\ell_{st}$ : Steel coil length (m) (see **Fig. 4.4.2-2**)

 $\alpha$ : The angle between the inner bottom plating and the hopper tank sloping (rad)

Table 4.4.2-2 has been amended as follows.

Table 4.4.2-2 Dynamic Load  $F_{SCd}$ 

	racio il 112 2 Dynamic Doud 1304			
Members		Load in waves $F_{SCd}$ (kN)		
Inner bottoms	$\frac{F_{SCS}}{g}C_{WDZ}a_{Ze-SC}$ $\frac{F_{SCS}}{g}C_{WDZ}a_{Ze-SC}$			
Hopper tank sloping	$\frac{\frac{F_{SCS}}{\cos\alpha}\cos( \theta - \alpha )}{F_{SCS}}\cos( \theta - \alpha )$			
T	$n_2 \le 10 \text{ and } n_3 \le 5$ $C_{SC3}W_{SC}\frac{n_1n_2}{n_3}g\sin\theta$			
Longitudinal bulkheads	$n_2 > 10 \text{ or } n_3 > 5$ $C_{SC3}W_{SC}n_1 \frac{\ell}{\ell_{SC}}g\sin\theta$			
Side frames	$C_{SC3}W_{SC}\frac{n_1}{n_4}g\sin\theta$			

Notes:

Coefficient of each load condition, specified in Table 4.4.2-8, Part 1

 $a_{Ze-SC}$ : Envelope acceleration in vertical direction  $(m/s^2)$  at the centre of gravity of steel coil in the cargo hold to be considered, as calculated in accordance with **4.2.4.1**, **Part1**<sup>(1)</sup>

 $\alpha$ ,  $n_1$ ,  $n_2$ ,  $n_3$ ,  $W_{SC}$ : As specified in **Table 4.4.2-1** 

 $\theta$ : Roll angle (rad), as specified in 4.2.2, Part  $\mathbf{1}^{(2)}$ .

 $C_{SC3}$ : Coefficient, as follows:

 $C_{SC3} = 4.0$  for single-tiered stacking or multi-tiered stacking in which the key coil is arranged in the second or third position from the ship side

 $C_{SC3} = 2.5$  for all other cases

 $n_4$ : The number of side frames that support a single steel coil.

- (1) The centre of gravity of steel coil to be considered is in accordance with **Table 4.4.2-3**.
- (2) The parameters (GM,  $z_G$ , etc.) required to calculate the ship motions and acceleration is in accordance with the values in the full load condition. The values in **Table 4.2.2-1** may be used if the parameters is not available.

### Table 4.4.2-3 The Centre of Gravity of Steel Coil (Omitted)

- 3 In applying -2 above, the number of load points per panel by dunnage  $n_2$  and the distance between the load points of dunnage at both ends of each panel  $\ell_{lp}$  are to be in accordance with the following (1) to (2).
- (1) For steel coil arrangements that do not consider floor position, as specified in **Fig. 4.4.2-2** and **Table 4.4.2-4**.
- (2) For steel coil arrangements that do consider floor position, as specified in the following (a) to (b). (See Fig. 4.4.2-3)
  - (a) The number of load points per panel by dunnage  $n_2$  is to be  $n_2 = n_3$ .
  - (b) The distance between the load points of the dunnage at both ends of each panel  $\ell_{lp}$  is to be the distance between the dunnage at both ends supporting a row of steel coils.

Fig. 4.4.2-2 Loading of Steel Coils on the Inner Bottom without Taking into Consideration the Floor Position

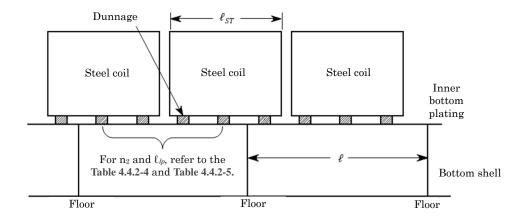


Fig. 4.4.2-3 Loading of Steel Coils on the Inner Bottom Taking into Consideration the Floor Position
(Omitted)

Table 4.4.2-4 Number of Load Points Per Panel According to Dunnage  $n_2$  (Omitted)

Table 4.4.2-5 has been amended as follows.

Table 4.4.2-5 Distance Between Load Points of Dunnage on Both Ends of Each Panel  $\ell_{lp}$  (m)

			8	
	$n_3$			
$n_2$	2	3	4	5
1		Actual wid	th of dunnage	
2	$0.5\ell_{st}$	$0.33\ell_{st}$	$0.25\ell_{st}$	$0.2\ell_{st}$
3	$1.2\ell_{st}$	$0.67\ell_{st}$	$0.50\ell_{st}$	$0.4\ell_{st}$
4	$1.7\ell_{st}$	$1.20\ell_{st}$	0.75ℓ <sub>st</sub>	$\frac{0.4\ell_{st}}{0.6\ell_{st}}$
5	$2.4\ell_{st}$	$1.53\ell_{st}$	$1.20\ell_{st}$	$0.8\ell_{st}$
6	$2.9\ell_{st}$	$1.87\ell_{st}$	$1.45\ell_{st}$	$1.2\ell_{st}$
7	$3.6\ell_{st}$	$2.40\ell_{st}$	$1.70\ell_{st}$	$1.4\ell_{st}$
8	$4.1\ell_{st}$	$2.73\ell_{st}$	$1.95\ell_{st}$	$1.6\ell_{st}$
9	$4.8\ell_{st}$	$3.07\ell_{st}$	$2.40\ell_{st}$	$1.8\ell_{st}$
10	$5.3\ell_{st}$	$3.60\ell_{st}$	2.65ℓ <sub>st</sub>	$2.0\ell_{st}$

#### 4 (Omitted)

#### Chapter 10 ADDITIONAL STRUCTURAL REQUIREMENTS

#### 10.1 Ships Carrying Steel Coils

#### 10.1.2 Inner Bottom Plating and Longitudinals

Paragraph 10.1.2.1 has been amended as follows.

#### **10.1.2.1** Inner Bottom Plating

The thickness of the inner bottom plating is not to be less than that obtained from the following formulae.

$$t = K_1 \sqrt{\frac{F_{SC}}{C_a \sigma_Y} \times 10^3 (mm)}$$

Where:

 $\sigma_{\rm Y}$ : Specified minimum yield stress ( $N/mm^2$ )

 $F_{SC}$ : Load (kN) acting on the inner bottom according to **4.4.2.1-2**.

 $K_1$ : Coefficient.

$$K_{1} = \sqrt{\frac{1.7 \frac{S}{1000} \ell K_{2} - 0.73 \left(\frac{S}{1000}\right)^{2} K_{2}^{2} - \left(\ell - \ell_{lp}\right)^{2}}{2\ell_{lp} \left(2 \frac{S}{1000} + 2\ell K_{2}\right)}}$$

 $K_2$ : Coefficient

$$K_2 = -\frac{s}{1000\ell} + \sqrt{\left(\frac{s}{1000\ell}\right)^2 + 1.37 \left(\frac{1000\ell}{s}\right)^2 \left(1 - \frac{\ell_{lp}}{\ell}\right)^2 + 2.33}$$

 $C_a$ : Axial force effect coefficient according to **6.3.2.1**, Part 1.

s: Distance between stiffeners (mm)

 $\ell$ : Distance between floors (m)

 $\ell_{lp}$ : The distance between the load points of the dunnage at both ends of each panel (m) according to 4.4.2.1-3

Paragraph 10.1.2.2 has been amended as follows.

#### **10.1.2.2** Inner Bottom Longitudinals

The section moduli and the web thicknesses of inner bottom longitudinals are not to be less than that obtained from the following formulae.

$$Z = K_3 \frac{F_{SC} \ell_{bdg}}{8C_s \sigma_Y} \times 10^3 (cm^3), t_w = \frac{0.5 F_{SC}}{d_{shr} \tau_Y} \times 10^3 (mm)$$

Where:

 $\sigma_Y$ : Specified minimum yield stress ( $N/mm^2$ )

 $\tau_{Y}$ : Allowable shear stress ( $N/mm^{2}$ )

 $\sigma_V/\sqrt{3}$ 

 $F_{SC}$ : Load (kN) acting on a longitudinal frame with inner bottom according to **4.4.2.1-2**,  $\ell$  is to be substituted by  $\ell_{bda}$ .

 $K_3$ : Coefficient according to **Table 10.1.2-1**  $K_3 = 2\frac{\ell_{\text{bath}}}{2}/3$  for  $n_2 > 10$ 

 $n_2$ : Load point per panel (the number of dunnages on one panel) according to **4.4.2.1-3** 

 $C_s$ : Coefficient related to the influence of axial force according to **Table 6.4.2.1**, Part 1

 $\ell_{bda}$ : Effective bending span (m) of stiffener according to 3.6.1.2, Part 1

 $d_{shr}$ : Effective shear depth (mm) of stiffener according to 3.6.4.2, Part 1

 $\ell_{lp}$ : Distance (m) between the load points of the dunnage at both ends for each panel according to **4.4.2.1-3** 

Table 10.1.2-1 has been amended as follows.

		Table 10.1.2-1	Coefficient $K_3$				
$n_2$		2	3	4	5		
<i>K</i> <sub>3</sub>	$\ell_{bdg}$	$t_{bdg} - rac{{\ell_{lp}}^2}{i_{bag}}$	$\ell_{bdg} - \frac{2\ell_{lp}^{2}}{3\ell_{bdg}}$	$\ell_{bdg} - \frac{5\ell_{lp}^2}{9\ell_{bdg}}$	$\ell_{bdg} - rac{{\ell_{lp}}^2}{2\ell_{bdg}}$		
$n_2$	6	7		9	10		
K <sub>3</sub>	$\ell_{bdg} - \frac{7\ell_{lp}^{2}}{15\ell_{bdg}}$	$\ell_{bdg} - \frac{4\ell_{lp}^2}{9\ell_{bdg}}$	$\ell_{bdg} - rac{3{\ell_{lp}}^2}{7{\ell_{bdg}}}$	$\ell_{bdg} - \frac{5\ell_{lp}^{2}}{12\ell_{bdg}}$	$\ell_{bdg} - \frac{11\ell_{lp}^{2}}{27\ell_{bdg}}$		
<u>n_2</u>	<u>1</u>	<u>2</u>	<u>3</u>	<u>4</u>	<u>5</u>		
<u>K<sub>3</sub></u>	<u>1.0</u>	$1.0 - \left(\frac{\ell_{lp}}{\ell_{bdg}}\right)^2$	$1.0 - \frac{2}{3} \left( \frac{\ell_{lp}}{\ell_{bdg}} \right)^2$	$1.0 - \frac{5}{9} \left( \frac{\ell_{lp}}{\ell_{bdg}} \right)^2$	$1.0 - \frac{1}{2} \left( \frac{\ell_{lp}}{\ell_{bdg}} \right)^2$		
<u>n_2</u>	<u>6</u>	<u>7</u>	<u>8</u>	<u>9</u>	<u>10</u>		
<u>K<sub>3</sub></u>	$1.0 - \frac{7}{15} \left( \frac{\ell_{lp}}{\ell_{bdg}} \right)^2$	$1.0 - \frac{4}{9} \left( \frac{\ell_{lp}}{\ell_{bdg}} \right)^2$	$1.0 - \frac{3}{7} \left( \frac{\ell_{lp}}{\ell_{bdg}} \right)^2$	$1.0 - \frac{5}{12} \left( \frac{\ell_{lp}}{\ell_{bdg}} \right)^2$	$1.0 - \frac{11}{27} \left( \frac{\ell_{lp}}{\ell_{bdg}} \right)^2$		

#### **10.1.2.3** Inner Bottom Longitudinals with Struts

Sub-paragraphs -2 and -3 have been amended as follows.

1 (Omitted)

2 The section modulus is not to be less than that obtained from the following formula. However, it is necessary to consider the conditions that correspond to either the following (1) or (2). It is presumed that the load condition is a steel coil loaded directly above the longitudinal and that a concentrated load is acting at the position of the dunnage.

$$Z = \frac{M}{C_S \sigma_V} \times 10^3 (cm^3)$$

(1) Equidistant load points (See Fig. 10.1.2-2)

$$M_{B} = \frac{W}{2 \ell \ell_{fs}^{2}} \left[ \sum_{k=1}^{n} \{a_{1} + (k-1)\ell_{1}\} \ell \ell_{fs}^{2} + \sum_{k=1}^{n} \{a_{1} + (k-1)\ell_{1}\}^{3} \right]$$

$$M_{Cm} = \frac{W\{a_{1} + (m-1)\ell_{1}\}}{2 \ell \ell_{fs}^{2}} \left[ \sum_{k=1}^{n} \{a_{1} + (k-1)\ell_{1}\}^{3} - 3 \ell \ell_{fs}^{2} \sum_{k=1}^{n} \{a_{1} + (k-1)\ell_{1} - \frac{2}{3} \ell \ell_{fs} \} \right] - \sum_{k=1}^{m} W(m-k)\ell_{1}$$

(2) Non-equidistant load points (See Fig. 10.1.2-3)

$$M_B = \sum_{k=1}^{n} \frac{W a_k \left( \frac{\ell \ell_{fs}^2}{2 \ell_{fs}^2} + \underline{-a_k}^2 \right)}{2 \ell \ell_{fs}^2}$$

$$M_{Cm} = \sum_{k=1}^{n} \frac{W\left(\frac{\ell \ell_{fs}}{2} - a_{k}\right)^{2} \left(2\frac{\ell \ell_{fs}}{2} + a_{k}\right)}{2\ell \ell_{fs}^{3}} a_{m} - \sum_{k=1}^{m} W(a_{m} - a_{k})$$

Where:

M: Bending moment (kN-m) and is to be the larger of  $M_B$  and  $M_C$ 

n: Maximum total number of load points between girders and struts

 $M_B$ : Bending moment at fixed end (kN-m)

 $M_{Cm}$ : Bending moment at the m-th load point from the strut support point (kN-m)

 $M_C$ : Bending moment (kN-m) at the load point is to be the largest of the following values  $M_{C1}$ ,  $M_{C2}$ ,  $M_{C3}$ , and  $M_{Cn}$ 

 $a_m$ : Distance from strut support point to the m-th load point (m)

 $a_1$ : The distance (m) from the strut support point to the first load point, and the value when the dunnage is arranged so that the values from  $M_{C1}$  to  $M_{Cm}$  and  $M_B$  are maximum values

 $\ell_1$ : Distance between load points (m)

 $\ell_{fs}$ : Distance between girders and struts (m)

W: The load of the steel coil that each dunnage is responsible for (kN)

$$W = \frac{F_{SC}}{n_{3}n_{2}}$$

 $F_{\underline{es_{SC}}}$ : Total load by steel coil acting on the plate panel (kN) (See 4.4.2.1-2),  $\ell$  is to be substituted by  $\ell_{bdg}$ .

n<sub>2</sub>: Number of dunnage threads supporting one row of steel coils (see 4.4.2.1-2)

 $\underline{n_2}$ : The load point per panel (the number of dunnages for a single panel), as specified in **4.4.2.1-3**.

3 The web thickness is not to be less than that obtained from the following formulae. However, it is necessary to consider under the conditions that correspond to either the following (1) or (2). It is presumed that the load condition is a steel coil loaded directly above the longitudinal and that a concentrated load is acting at the dunnage position.

$$t_w = \frac{F}{d_{shr}\tau_Y} \times 10^3 \; (mm)$$

(1) Equidistant load points (See Fig. 10.1.2-2)

$$R_{A} = \frac{W}{2 ! \ell_{fs}^{3}} \left[ \sum_{k=1}^{n} \{a_{1} + (k-1)\ell_{1}\}^{3} - 3 ! \ell_{fs}^{2} \sum_{k=1}^{n} \{a_{1} + (k-1)\ell_{1} - \frac{2}{3} ! \ell_{fs}^{2} \} \right]$$

$$R_{B} = \frac{W}{2 ! \ell_{fs}^{3}} \left[ 3 ! \ell_{fs}^{2} \sum_{k=1}^{n} \{a_{1} + (k-1)\ell_{1}\} - \sum_{k=1}^{n} \{a_{1} + (k-1)\ell_{1}\}^{3} \right]$$

(2) Load point position user determined (See Fig. 10.1.2-3)

$$R_{A} = \sum_{k=1}^{n} \frac{W\left(\ell \ell_{fs} - a_{k}\right)^{2} \left(2\ell \ell_{fs} + a_{k}\right)}{2\ell \ell_{fs}^{3}}$$

$$R_{B} = \sum_{k=1}^{n} \frac{Wa_{k}\left(3\ell \ell_{fs}^{2} + a_{k}^{2}\right)}{2\ell \ell_{fs}^{3}}$$

Where.

F: Shear force (kN) and is to be the larger of  $R_A$  and  $R_B$ 

 $R_A$ : Reaction force with simple support (kN)

 $R_B$ : Reaction force at fixed end (kN)

 $a_1$ : Distance (m) from the strut support point to the first load point, and the value when the dunnage is arranged so that the respective values of  $R_A$  and  $R_B$  are maximum values

Fig. 10.1.2-1 Inner Bottom Longitudinal with Struts

Coil load (concentrated load at the dunnage point)

R

Strut support point

Fig. 10.1.2-2 has been amended as follows.

Fig. 10.1.2-2 Load Conditions Between Girders and Struts (Equidistant Load Points)

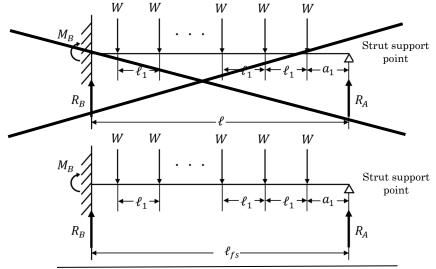
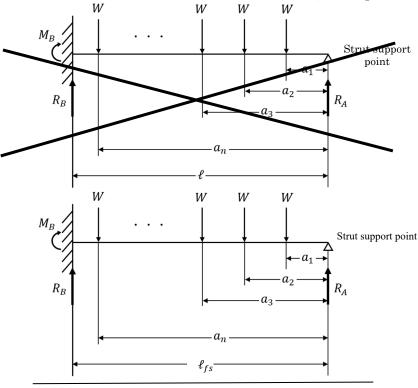


Fig. 10.1.2-3 has been amended as follows.

Fig. 10.1.2-3 Load Conditions Between Girders and Struts (Non-Equidistant Load Points)



### 10.1.3 Hopper Slant Plates and Longitudinal Frames with Hopper Slant Plates (Ships with Bilge Hoppers)

Paragraph 10.1.3.2 has been amended as follows.

#### 10.1.3.2 Longitudinal Frames with Hopper Slant Plates

The section moduli and web thicknesses of longitudinal frames with hopper slant plates are to be greater than or equal to the following values.

$$Z = K_3 \frac{F_{SC} \ell_{bdg}}{8C_s \sigma_Y} \times 10^3 (cm^3), t_w = \frac{0.5 F_{SC}}{d_{shr} \tau_Y} \times 10^3 (mm)$$

Where:

 $\sigma_{\rm Y}$ : Specified minimum yield stress (N/mm<sup>2</sup>)

 $\tau_Y$ : Allowable shear stress  $(N/mm^2)$ 

 $\sigma_Y/\sqrt{3}$ 

 $F_{SC}$ : Load (kN) acting on the longitudinal frame with hopper slant plate according to **4.4.2.1-2**,  $\ell$  is to be substituted by  $\ell_{bdg}$ .

 $K_3$ : Coefficient according to **10.1.2.2** 

 $C_s$ : Coefficient related to the influence of axial force according to **6.4.2.1**, Part 1

 $d_{shr}$ : Effective shear depth (mm) of stiffener according to 3.6.4.2, Part 1

### 10.1.4 Longitudinal Bulkheads and Longitudinal Frames with Longitudinal Bulkheads (Ships without Bilge Hopper and Ships with Double Side Shells)

Paragraph 10.1.4.2 has been amended as follows.

#### 10.1.4.2 Longitudinal Frames with Longitudinal Bulkheads

The section moduli and web thicknesses of longitudinal frames with longitudinal bulkheads are to be greater than or equal to the following values.

$$Z = K_3 \frac{F_{SC} \ell_{bdg}}{8C_s \sigma_Y} \times 10^3 (cm^3), t_w = \frac{0.5 F_{SC}}{d_{shr} \tau_Y} \times 10^3 (mm)$$

Where:

 $\sigma_{\rm Y}$ : Specified minimum yield stress (N/mm<sup>2</sup>)

 $\tau_Y$ : Allowable shear stress  $(N/mm^2)$ 

 $\sigma_Y/\sqrt{3}$ 

 $F_{SC}$ : Load (kN) acting on the longitudinal frame with longitudinal bulkhead according to **4.4.2.1-2**,  $\ell$  is to be substituted by  $\ell_{bdg}$ .

 $K_3$ : Coefficient according to **10.1.2.2** 

 $C_s$ : Coefficient related to the influence of axial force according to **6.4.2.1**, Part 1

 $d_{shr}$ : Effective shear depth (mm) of stiffener, according to 3.6.4.2, Part 1

#### 10.1.5 Side Frames (Ships Without Bilge Hoppers and Single-Side Ships)

Paragraph 10.1.5.1 has been amended as follows.

#### **10.1.5.1** Side Frames

The section moduli and web thicknesses of side frames are to be greater than or equal to the following values.

$$Z = 1.2 \frac{F_{SC} \ell_{1bdg}}{8\sigma_Y} \times 10^3 (cm^3), \ t_w = 2.0 \frac{0.5 F_{SC}}{d_{shr} \tau_Y} \times 10^3 (mm)$$

 $\sigma_{\rm Y}$ : Specified minimum yield stress  $(N/mm^2)$ 

 $\tau_Y$ : Allowable shear stress ( $N/mm^2$ )

 $\sigma_{\rm v}/\sqrt{3}$ 

 $F_{SC}$ : Load (kN) acting on the longitudinal frame with longitudinal bulkhead side frame according to 4.4.2.1-2

 $\ell_{1bdg}$ : Effective bending span (m) of the side frame. Where a bracket is provided, the end of the effective bending span is to be taken to the position where the depth of the side frame and the bracket is equal to  $2h_w$  (See Fig. 6.4.3-2, Part 1).

 $d_{shr}$ : Effective shear depth (mm) of stiffener according to 3.6.4.2, Part 1

## Part 2-6 VEHICLES CARRIERS AND ROLL-ON/ROLL-OFF SHIPS

#### Chapter 4 LOADS

#### 4.3 Loads to be Considered in Strength of Primary Supporting Structures

#### 4.3.2 Maximum Load Condition

Table 4.3.2-2 has been amended as follows.

Table 4.3.2-2 External and Internal Pressure to be Considered

Structures to be assessed		$P_{DB}(kN/m^2)^{(1)(2)}$	$P_{DS}(kN/m^2)^{(1)(2)}$	
Double bottom	<i>S</i> 1	$P_{exs} + P_{exw}$	$P_{exs} + P_{exw}$	
	S2	$P_{exs} + P_{exw} - P_{in\_s2}$	$P_{exs} + P_{exw}$	
Double side	S3	$P_{exs} + P_{exw} - P_{in\_s3}$	$P_{exs} + P_{exw}$	

#### Notes:

 $P_{exs}$ ,  $P_{exw}$ : Hydrostatic and Hydrodynamic pressure  $(kN/m^2)$  act on bottom shell in case of  $P_{DB}$ . Those values act on side shell in case of  $P_{DS}$ . Each value is calculated in accordance with **4.6.2.4**, **Part 1**.

 $P_{in\_s2}$ ,  $P_{in\_s3}$ : The values considering the effect due to cargo  $(kN/m^2)$ , as given by the following formulae:

$$\begin{aligned} P_{in\_s2} &= 0.5 \rho g T_{SC} \\ P_{in\_s3} &= \rho g T_{SC} \end{aligned}$$

(1) Load calculation points is are to be in accordance with 7.3.1.5, Part 1 for all loading conditions.

2) When calculating loads,  $T_{LC} = 0.7T_{SC}$  for S1 and  $T_{LC} = T_{SC}$  for S2 and S3.

#### Chapter 8 STRENGTH ASSESSMENT BY CARGO HOLD ANALYSIS

### 8.5 Racking Strength Assessment

#### 8.5.3 Yield Strength Assessment

Paragraph 8.5.3.2 has been amended as follows.

#### 8.5.3.2 Strength Criteria

1 Members to be assessed within the evaluation are to satisfy the following formula:

$$\lambda_y \leq \lambda_{yperm}$$

 $\lambda_{\nu}$ : Yield utilisation factor, as given by the following formulae:

For shell elements, 
$$\lambda_y = \frac{\sigma_{eq}}{188/K}$$
  
For rod elements,  $\lambda_y = \frac{|\sigma_a|}{188/K}$ 

 $\lambda_{vnerm}$ : Permissible utilisation factor, be taken as 1.0

2 Notwithstanding -1 above, yield strength assessments of intersections of pillars and deck transverses are considered to be satisfied for the structural types specified in 8.5.1.2-1(2) or similar structural types when the following formula is satisfied:

$$t_{is-gr}\sigma_{Y-is} \geq 1.7 \cdot t_{dt-gr}\sigma_{Y-dt}$$

 $\underline{t_{is-ar}}$ : Thickness (mm) (gross scantling) of intersections of pillars and deck transverses.

 $\underline{\sigma_{Y\_is}}$ : Specified minimum yield stress ( $N/mm^2$ ) of intersections of pillars and deck transverses.

 $\underline{t_{dt-gr}}$ : Smallest thickness (mm) (gross scantling) of the web of the deck transverses at the same height of the intersection.

 $\sigma_{Y\_dt}$ : Smallest specified minimum yield stress  $(N/mm^2)$  of the web of the deck transverses at the same height of the intersection.

23 Where members do not satisfy the criteria specified in -1 and -2 above, a detailed fatigue strength assessment are to be performed for the members and so as to ensure a sufficient fatigue lifetime.

### **Chapter 9 FATIGUE**

Section 9.5 has been added as follows.

#### 9.5 Screening Assessment

#### **9.5.1 General**

#### **9.5.1.1** General

- This 9.5 specifies a method for assessing the fatigue strength of structural details including those specified in 9.2 using coarse mesh models instead of the very fine mesh models specified in 9.4, Part 1.
- When the fatigue damage calculated based upon the method specified in this 9.5 does not satisfy the criteria, fatigue strength assessment using a very fine mesh model is to be carried out.
- 3 The hot spot to be assessed, stress concentration factor derivation method, etc. is to be submitted beforehand to the Society for approval, when the fatigue strength is assessed according to the method specified in this 9.5.

#### 9.5.1.2 Application

The method specified in this 9.5 is applied to general structural details to be assessed, as specified in Table 9.2.1-1 and Table 9.2.1-2.

#### 9.5.1.3 Confirmation of Calculation Method and Accuracy of Analysis

- 1 The analysis method and analysis program are to have the following functions.
- (1) The effects of bending, shear, axial, and torsional deformation are to be effectively taken into consideration.
- (2) The behaviour of the 3-D structural model is to be represented effectively under reasonable boundary conditions.
- (3) The method and program are to be confirmed to have sufficient analytical accuracy.
- 2 The analysis method is to be approved in advance by the Society. The submission of details regarding the analysis method and verification of accuracy may be required when deemed necessary by the Society.

#### 9.5.2 Finite Element Modelling

#### **9.5.2.1** General

Coarse mesh models used in screening assessments are to be in accordance with **8.2**.

#### 9.5.2.2 Corrosion Models

Coarse mesh models used for screening assessments are to be made using  $tt_{n25}$  (mm). The corrosion addition to be considered is to be in accordance with the requirement in 3.3.4, Part 1.

#### 9.5.3 Loading Conditions and Fractions of Time to be Considered

#### **9.5.3.1** General

Loading conditions and fractions of time are to be in accordance with 9.3.1.

#### 9.5.4 Boundary Conditions and Load Conditions

#### **9.5.4.1** General

Boundary conditions and load conditions to be considered are to be in accordance with 9.4.

#### 9.5.5 **Hot Spot Stress**

#### 9.5.5.1 General

- Hot spot stress may be obtained by using the stress acting on element midpoints of shell elements irrespective of the hot spot type used in the screening assessment.
- The hot spot stress range and mean stress are to be obtained from the following formulae when conditions "i1" and "i2" for the same equivalent design wave for the same loading condition (j) are considered. The orthogonal direction to the weld line is represented by the x-direction and the parallel direction is represented by the y-direction.

 $\Delta \sigma_{HS\ ort.i(i)} = K_{SCF} \cdot \Delta \sigma_{adi\ x.i(i)}$ 

 $\Delta \sigma_{HS \ par.i(i)} = K_{SCF} \cdot \Delta \sigma_{adi \ v.i(i)}$ 

 $K_{SCF}$ : Stress concentration factor as given in -3 below.

 $\Delta \sigma_{adj \times i(i)}$ : Stress range (N/mm<sup>2</sup>) in the x-direction in the x-y coordinate system of the equivalent design wave "i" for loading condition (j) according to the following formula:

 $\Delta \sigma_{adi\ x,i(i)} = \left| \sigma_{adi\ x,i1(i)} - \sigma_{adi\ x,i2(i)} \right|$ 

 $\sigma_{adi\ xi1(i)}$ : Surface stress  $(N/mm^2)$  in the x-direction in the x-y coordinate system of the equivalent design wave "i1" for

loading condition "j" on element midpoint of element in

way of hot spot

Surface stress  $(N/mm^2)$  in the x-direction in the x-y

coordinate system of the equivalent design wave "i2" for loading condition "j" on element midpoint of element in

way of hot spot

 $\Delta \sigma_{adi,vi(i)}$ : Stress range (N/mm<sup>2</sup>) in the y-direction in the x-y coordinate system of the equivalent design wave "i" for loading condition "j" according to the following formula:

 $\Delta \sigma_{adj,y,i(j)} = \left| \sigma_{adj,y,i1(j)} - \sigma_{adj,y,i2(j)} \right|$ 

Surface stress  $(N/mm^2)$  in the y-direction in the x-y  $\sigma_{adi\ v,i1(i)}$ :

coordinate system of the equivalent design wave "i1" for loading condition "j" on element midpoint of element in

way of hot spot

Surface stress  $(N/mm^2)$  in the y-direction in the x-y  $\sigma_{adi\ v,i2(i)}$ :

coordinate system of the equivalent design wave "i2" for loading condition "j" on element midpoint of element in

way of hot spot

 $\sigma_{mean\_ort,i(j)} = K_{SCF} \cdot \frac{\sigma_{adj\_x,i1(j)} + \sigma_{adj\_x,i2(j)}}{2}$   $\sigma_{mean\_par,i(j)} = K_{SCF} \cdot \frac{\sigma_{adj\_y,i1(j)} + \sigma_{adj\_y,i2(j)}}{2}$ 

- $K_{SCE}$  for a representative hot spot of a representative transverse section is to be obtained by the following. The application method of an obtained  $K_{SCF}$  to other transverse sections is to be approved in advance by the Society.
- Hot spot stress range for a representative hot spot of a representative transverse section is to be obtained by fatigue strength assessment using a very fine mesh model.
- Nominal stress range is to be obtained in the same location as assessed in (1) above by finite element analysis using a coarse mesh model
- $K_{SCF}$  is to be obtained as the ratio of stress range obtained by (1) and (2) above.

#### 9.5.6 **Fatigue Strength Assessment**

#### 9.5.6.1 General

- 1 This 9.5.6 specifies requirements for the fatigue strength assessment method using the hot spot stresses obtained in **9.5.5**.
- The fatigue strength assessment in this **9.5** is based on Miner's linear cumulative damage rule.
- Total cumulative fatigue damage is to be calculated for all loading conditions by using each fatigue damage in the in-air environment where the coating is effective and in the corrosive environment where the coating is not effective, considering the ratio of period of each environment.

#### Reference Stress for Fatigue Strength Assessment

Hot spot stress ranges used in screening assessments,  $\Delta \sigma_{FS(i)}$ , are  $\Delta \sigma_{FS,ort(i)}$  and  $\Delta \sigma_{FS,ngr(i)}$ , and fatigue damage is to be calculated for each stress range.

where.

$$\Delta \sigma_{FS\_ort,(j)} = \max_{i} (\Delta \sigma_{FS\_ort,i(j)})$$

$$\Delta \sigma_{FS\_par,(j)} = \max_{i} (\Delta \sigma_{FS\_par,i(j)})$$

 $\Delta \sigma_{FS, ort, i(i)}$ : Hot spot stress range  $(N/mm^2)$  for screening assessment according to the hot spot stress in the direction orthogonal to the weld line, as obtained from the following formula:

$$\Delta \sigma_{FS\_ort,i(j)} = f_{mean\_ort,i(j)} \cdot \Delta \sigma_{HS\_ort,i(j)}$$

 $\Delta \sigma_{FS,par,i(i)}$ : Hot spot stress range  $(N/mm^2)$  for screening assessment according to the hot spot stress in the direction parallel to the weld line, as obtained from the following formula:

$$\Delta \sigma_{FS\_par,i(j)} = 0.72 \cdot f_{mean\_par,i(j)} \cdot \Delta \sigma_{HS\_par,i(j)}$$

f<sub>mean ort.i(i)</sub>, f<sub>mean nar.i(i)</sub>: Correction factor for mean stress effect, as obtained by the following formulas for each combination of  $\Delta \sigma_{HS \ ort,i(i)}$ ,  $\sigma_{mean \ ort,i(i)}$  and  $\Delta \sigma_{HS \ par,i(i)}$ ,  $\sigma_{mean \ nari(i)}$  respectively.

$$\int_{mean,i(j)} \frac{\sigma_{mean\_par,i(j)} \text{ respectively.}}{1.0,0.8 + 0.2 \frac{\sigma_{mCor,i(j)}}{2\Delta\sigma_{HS,i(j)}}} : \sigma_{mCor,i(j)} \ge 0$$

$$\int_{mean,i(j)} e^{-max} \left[ 0.6,0.8 + 0.2 \frac{\sigma_{mCor,i(j)}}{2\Delta\sigma_{HS,i(j)}} \right] : \sigma_{mCor,i(j)} < 0$$
Where,  $\sigma_{mCor,i(j)}$  is to be obtained as follows:

Where,  $\sigma_{mCori(i)}$  is to be obtained as follows

$$\begin{cases} \sigma_{mCor,i(j)} = \sigma_{mean,i(j)} : \sigma_{max} \leq \sigma_{YEq} \\ \hline \sigma_{mCor,i(j)} = \sigma_{YEq} - \sigma_{max} + \sigma_{mean,i(j)} : \sigma_{max} > \sigma_{YEq} \\ \hline \sigma_{max} = \max_{i(j)} (\Delta \sigma_{HS,i(j)} + \sigma_{mean,i(j)}) \end{cases}$$

$$\sigma_{YEg} = \max(315, \sigma_Y)$$

 $\Delta \sigma_{HS \ ort \ i(i)}$ ,  $\Delta \sigma_{HS \ nor \ i(i)}$ : As given in **9.5.5.1-2**.  $\sigma_{mean\_ort,i(j)}$ ,  $\sigma_{mean\_par,i(j)}$ : As given in 9.5.5.1-2.

#### 9.5.6.3 Fatigue Damage Calculation and Fatigue Strength Assessment Criterion

The cumulative fatigue damage D is to be obtained from the following formula:

$$\underline{D} = \sum_{i} \alpha_{(j)} \cdot \underline{D_{(j)}}$$

 $\underline{\alpha_{(i)}}$ : Fraction of time of loading condition (i) in the fatigue design life, as given in **Table** 

 $\underline{D_{(i)}}$ : Cumulative fatigue damage for the fatigue design life for loading condition (j)calculated by the following formula:

$$\underline{D_{(j)}} = \frac{T_{DF} - T_C}{T_{DF}} \underline{D_{air,(j)}} \cdot + \frac{T_C}{T_{DF}} \underline{D_{cor,(j)}}$$

Dair (i), Dcor (i): Cumulative fatigue damage in the in-air environment and corrosive environment for the fatigue design life for loading

$$\underline{D_{air,(j)}} = \frac{N_{DF}}{K_{2,air}} \frac{\Delta \sigma_{FS,(j)}^{m}}{(\ln N_{R})^{m/\xi}} \cdot \underline{\mu_{(j)} \cdot \Gamma\left(1 + \frac{m}{\xi}\right)}$$

$$\underline{D_{cor,(j)}} = \frac{N_{DF}}{K_{2,cor}} \frac{\Delta \sigma^m_{FS,(j)}}{(\ln N_R)^{m/\xi}} \cdot \Gamma \left( 1 + \frac{m}{\xi} \right)$$

$$\frac{N_{DF}\colon \text{ Total number of cycles in the fatigue design life } T_{DF} \ .}{N_{DF} = \frac{60\times60\times24\times365.25}{4\text{log}L_c}\cdot f_D\cdot T_{DF}}$$

 $f_D$ : Ship's operation rate, taken as 0.85  $\Delta \sigma_{FS,(j)}$ : Fatigue stress range  $(N/mm^2)$  at the reference probability level of exceedance of

m: Inverse of the slope of the S-N curve, taken as m = 3

 $N_R$ : Number of cycles corresponding to the reference probability of exceedance of  $10^{-2}$ , taken as  $N_R = 100$ 

 $\xi$ : Weibull shape parameter, taken as  $\xi = 1$ 

 $\Gamma(x)$ : Complete Gamma function

 $K_{2.air}$ : Constant of the design S-N curve for in-air environment, taken as  $1.52 \times 10^{12}$ 

 $K_{2 cor}$ : Constant of the design S-N curve for corrosive environment, taken as  $7.60 \times 10^{11}$ 

 $\Delta \sigma_a$ : Stress range (N/mm<sup>2</sup>) corresponding to the intersection of the two segments of design S-N curve at  $N = 10^7$  cycles, taken as  $\Delta \sigma_a = 53.4$ 

 $\mu_{(i)}$ : Coefficient taking into account the change of inverse slope of the S-N curve, m,

· For in-air environment

$$\underline{\mu_{(j)}} = 1 - \frac{\left\{ \gamma \left( 1 + \frac{m}{\xi}, \nu_{(j)} \right) - \nu_{(j)}^{-\Delta m/\xi} \cdot \gamma \left( 1 + \left( \frac{m + \Delta m}{\xi} \right), \nu_{(j)} \right) \right\}}{\Gamma \left( 1 + \frac{m}{\xi} \right)}$$

$$\underline{\nu_{(j)}} = \left(\frac{\Delta \sigma_q}{\Delta \sigma_{FS_s(j)}}\right)^{\xi} \ln N_R$$

For corrosive environment

$$\underline{\mu_{(j)}=1}$$

y(a, x): Incomplete Gamma function

 $\Delta m$ : Change in inverse slope of S-N curve at  $N = 10^7$  cycles, taken as  $\Delta m = 2$ .

- Fatigue strength assessment criterion is to be in accordance with 9.5.5, Part 1.
- Correction factor for post-weld treatment by grinding is not to be considered.

#### Part 2-7 TANKERS

#### Chapter 2 GENERAL ARRANGEMENT DESIGN

#### 2.1 Structural Arrangements

#### 2.1.1 Arrangement and Separation

Paragraph 2.1.1.5 has been deleted, and Paragraph 2.1.1.6 has been renumbered to Paragraph 2.1.1.5.

#### 2.1.1.5 Arrangements of Swash Bulkheads

Where the length or breadth of a cargo oil tank exceeds, 15 m or  $0.1L_{\text{c}}$  (m), whichever is greater, swash bulkheads are to be provided in cargo oil tanks. However, in accordance with the requirement in 4.2.1 (2), Part S, this requirement may be dispensed with if special consideration is given to sloshing.

#### 2.1.1.65 Length of Deep Tanks

The length of deep tanks is not to exceed  $0.2L_f$  (m).

Paragraph 2.1.2 has been deleted.

#### 2.1.2 Primary Supporting Members

#### 2.1.2.1 Arrangements of Primary Supporting Members

- 1 For tankers with double hull structures having no longitudinal bulkheads on the centreline or tankers with double hull structures having longitudinal bulkheads only on the centreline, the arrangement of primary members in the double bottom and double side hull of the eargo tank area are to be determined in the following (1) to (5).
- (1) The height of the double bottom in eargo oil spaces is not to be less than B/20 (m).
- (2) The width of the double side hull is not to be less than D/9 (m).
- (3) In double bottoms in eargo oil spaces, girders are to be provided at a spacing not exceeding  $0.9\sqrt{\ell_{\pm}}$  (m) and floors are to be provided at a spacing not exceeding  $0.55\sqrt{B}$  (m) or  $0.75\sqrt{D}$  (m), whichever is smaller.
- (4) In the double side hull, stringers are to be provided at a spacing not exceeding  $1.1\sqrt{\ell_{\pi}}$  (m).
- (5) Transverses in the double side hull, eargo oil tanks and deep tanks are to be provided in line with the floors in the double bottom.
- 2 If the spacing of girders and floors in the double bottom and stringers and transverses in the double side hull according to -1 above are smaller than the values shown in (1) and (2) in tankers without partial

loading conditions (such as half-loading or alternate loading), the spacing may be increased to the values given in (1) and (2).

- (1) Girders in the double bottom and stringers in the double side hull: 4.1 (m)
- (2) Floors in the double bottom and transverses in the double side hull: 2.8 (m)

#### Chapter 7 PRIMARY SUPPORTING MEMBER STRUCTURAL STRENGTH

#### 7.1 General

#### 7.1.1 Application

#### 7.1.1.1

Sub-paragraph -2 has been renumbered to Sub-paragraph -3, and Sub-paragraph -2 has been added as follows.

- 1 The requirements in this Chapter apply to ships of less than 150 m in length  $L_C$ .
- 2 Notwithstanding -1 above, strength assessments for deck girders with respect to deck loads and green sea loads are to be carried out in accordance with this Chapter.
- **23** For the double bottom and double-side skin structure, the requirements of the double hull structure specified in **7.3**, **Part 1** are to be applied. For other girder members that can be regarded as simple girders, the requirement of the simple girder specified in **7.2**, **Part 1** are to be applied.

#### Chapter 8 STRENGTH ASSESSMENT BY CARGO HOLD ANALYSIS

#### **8.3** Structural Models

#### 8.3.1 General

Title of Paragraph 8.3.1.2 has been amended as follows.

#### 8.3.1.2 In Way of Lower End of <u>Vertically</u> Corrugated Bulkhead

Sub-paragraphs -1 and -2 have been amended as follows.

- In applying 8.3.3.1-2, Part 1, plating and primary supporting members around the lower end of <u>vertically</u> corrugated bulkhead are to be modelled by shell elements with the mesh-sized of  $100 \text{ } mm \times 100 \text{ } mm$  or less.
- 2 In applying -1 above, members for reinforcement directly under <u>vertically</u> corrugated bulkheads across the inner bottom plating are to be appropriately modelled. Shell elements are to be used for modelling if necessary.

#### 8.5 Strength Assessments

#### 8.5.1 Yield Strength Assessments

#### **8.5.1.1** Reference Stress

Sub-paragraph -1 has been amended as follows.

1 In applying 8.6.1.1, Part 1, the averaged stresses of multiple elements within the range deemed appropriate by the Society may be taken as the reference stress for locations where the

mesh size specified in **8.3.1.2** is applied. A range that web depth of the vertically corrugated bulkhead is divided into three parts (that is, approximately  $300 \text{ } mm \times 300 \text{ } mm$ ) is to be taken as the standard range.

## Part 2-9 SHIPS CARRYING LIQUEFIED GASES IN BULK (INDEPENDENT PRISMATIC TANKS TYPE A/B)

#### Chapter 4 Loads

#### 4.2 Loads to be Considered in Local Strength

#### 4.2.2 Maximum Load Condition

#### 4.2.2.1 Lateral Loads

Sub-paragraph -1 has been amended as follows.

In applying 4.4.2, Part 1, the parameters required to calculate the dynamic pressure due to cargo are to be the values in the loading condition where cargo is loaded in the cargo hold to be considered and the draught is at a minimum (e.g. one-tank-loaded condition). The radius of gyration (m) around X-axis is to be taken as 0.38B but the value calculated based upon the weight distribution according to the loading condition to be considered may be used.

#### 4.2.3 Loads to be Considered in Tank Type A

Paragraphs 4.2.3.1 to 4.2.3.3 have been amended as follows.

#### 4.2.3.1 Loads to be Considered in Tank Boundary plates

The load  $P_{in}$   $(kN/m^2)$  to be considered for tank boundary plates is to be in accordance with the following formula. However,  $P_2$  is used only where  $P_h$  is given specified in **4.13.2-3**, **Part N**.

$$P_{in} = \max(P_1, P_2, P_3)$$

$$P_1 = P_{ls} + P_{ld}$$

$$P_2 = P_{ls=2}$$

$$P_2 = P_{ls=2}/0.95$$

 $P_3 = P_{heel}$ 

 $P_{ls}$ : Static pressure and design vapor pressure  $(kN/m^2)$ , as specified in **4.4.2.4**, Part 1=

 $P_{ld}$ : Dynamic pressure  $(kN/m^2)$ , in accordance with **4.4.2.4**, **Part 1**.

 $P_{ls\_2}$ : Static pressure and design vapor pressure in harbour condition  $(kN/m^2)$ . As for  $P_{ls}$  specified in **4.4.2.4**, **Part 1**, the value obtained by replacing the design vapor pressure with  $P_h$  specified in **4.13.2-3**, **Part N**.

 $P_{heel}$ : Maximum static pressure in 30-degree static heel condition  $(kN/m^2)$ 

#### 4.2.3.2 Loads Considering Increase of Pressure due to a Fire

The Load  $P_f$   $(kN/m^2)$  considering increase of pressure due to a fire is to be in accordance with the following formula. However,  $P_{f2}$  is used only where  $P_h$  is given specified in **4.13.2-3**, **Part N**.

$$\begin{split} P_f &= \max(P_{f1}, P_{f2}) \\ P_{f1} &= P_{ls\_f1} + P_{ld} \\ \frac{P_{f2} &= P_{ls\_f2}}{P_{f2}} P_{f2} = P_{ls\_f2}/0.95 \end{split}$$

 $P_{ls\_f1}$ : Static pressure and design vapor pressure  $(kN/m^2)$ , the value is obtained by multiplying the design vapor pressure  $P_0$  by 1.2 in  $P_{ls}$  specified in **4.4.2.4**, **Part** 

1.

 $P_{ld}$ : Dynamic pressure  $(kN/m^2)$  in accordance with the requirements of **4.4.2.4**, **Part 1**.

 $P_{ls\_f2}$ : Static pressure and design vapor pressure in harbour condition  $(kN/m^2)$ . As for  $P_{ls}$  specified in **4.4.2.4**, **Part 1**, the value obtained by replacing the design vapor pressure with 1.2 times the value of  $P_h$  specified in **4.13.2-3**, **Part N**.

#### 4.2.3.3 Loads to be Considered in Centreline Bulkheads

The load  $P_{CL}$   $(kN/m^2)$  to be consider for the centreline bulkhead is to be in accordance with the following formula:

 $P_{CL} = \max(P_{CL1}, P_{CL2}, P_{CL3})$ 

 $P_{CL1} = P_{ld}$ 

 $P_{CL2} = P_{heel}$ 

 $P_{CL3} = P_{ope}$ 

 $P_{ld}$ : Dynamic pressure  $(kN/m^2)$  in accordance with the requirements of **4.4.2.4**, **Part 1**.

 $P_{heel}$ : Static pressure  $(kN/m^2)$  from maximum difference between liquid levels of both sides of cargo tank in 30-degree static heel condition.

 $P_{ope}$ : Internal pressure  $(kN/m^2)$  of cargo tank according to operational restrictions, to be taken as follows. Where operations are limited, operational limitations are to be specified in the loading manual.

Where asymmetric loading of both side of cargo tanks is not allowed in harbour condition,

 $P_{ope} = 0.4 P_{CL1}$ 

Where asymmetric loading of both side of cargo tanks is allowed in harbour condition,

 $\frac{P_{one} - P_{ls}}{P_{one}} = P_{ls}/0.95$ 

Where asymmetric loading of both side of cargo tanks is allowed in maximum load condition,

 $P_{ope} = P_{ls} + P_{ld}$ 

#### **Chapter 6 LOCAL STRENGTH**

#### **6.1** Independent Prismatic Tanks

#### 6.1.1 General

#### 6.1.1.1 Application

The scantlings of the plates and stiffeners which subject to liquid cargo are to be in accordance with **6.1**.

#### 6.1.2 Design Load Scenarios and Applied Loads for Assessment Target Members

#### 6.1.2.1

The Design Load Scenarios and Applied Loads for Assessment Target Members/Compartments are to be in accordance with **Table 6.1.2-1**.

Table 6.1.2-1 has been amended as follows.

Table 6.1.2-1 Design Load Scenarios and Applied Loads for Assessment Target Members/Compartments

		Wiemoe	18/Compai	timents		
				Applied lo	ad	
Members/compart ments to be	Design Load Scenarios			_	R	eference
assessed		Lateral load	Load type	Load component	Lateral load (P)	Hull girder load $(M_{V-HG}, M_{H-HG})$
Tank casing	Maximum load condition (normal)	Internal pressure	Liquid cargo	Static/dynamic loads	4.2.3.1	
Tank casing	Harbour condition (normal)	Internal pressure	Liquid cargo	Static load	4.2.3.1	
Tank casing	30-degree static listed condition	Internal pressure	Liquid cargo	Static load	4.2.3.1	
Tank casing	Maximum load condition (Pressure rising due to a fire)	Internal pressure	Liquid cargo	Static/dynamic loads	4.2.3.2	
Tank casing	Harbour condition (Pressure rising due to a fire)	Internal pressure	Liquid cargo	Static load	4.2.3.2	-
<u>eC</u> entre line bulkhead <sup>(1)</sup>	Maximum load condition (normal)	Internal pressure	Liquid cargo	Static/dynamic loads	4.2.3.3	
Centre line bulkhead <sup>(1)</sup>	30 degree static listed condition	Internal pressure	Liquid cargo	Static load	4.2.3.3	
Centre line bulkhead <sup>(1)</sup>	Condition under operational restrictions	Internal pressure	Liquid cargo	Static load	4.2.3.3	

#### Note:

<sup>(1)</sup> Where openings which cannot be closed (excluding vapour spaces at the centreline bulkhead) are installed, the requirements need not be applied.

Paragraph 6.1.3 has been amended as follows.

#### 6.1.3 Plates and Stiffeners

#### 6.1.3.1

- The plates and stiffeners of the independent prismatic tanks are to satisfy the requirements of **6.3** and **6.4**, **Part 1** respectively for design load scenarios and applied loads specified in **Table 6.1.1-1**. In applying **6.3** and **6.4**, **Part 1**, the following requirements (1) to (3) are to be complied with.
- (1) <u>\*F</u>or design load scenarios considering pressure rising due to a fire in those in **Table 6.1.2-1**, the plate and stiffeners are to be evaluated by using formulae for flooded conditions. And, for other design load scenarios, the evaluations are to be carried out by using formulae for maximum load conditions.
- (2) However,  $t\underline{T}$  he coefficients  $C_a$  and  $C_s$  related to the influence of the axial force  $\frac{may}{may}$  are to be taken as 1.0 respectively.
- (3) In applying 6.4, Part 1,  $C_{Safety}$  is to be 1.1, and  $\sigma_Y/1.33$  or  $\sigma_B/2.66$ , whichever is less, is to be used instead of  $\sigma_Y$ , where  $\sigma_B$  is the specified minimum tensile strength at room temperature, as specified in 8.5.1.1-2.
- 2 In addition to -1 above, where high density cargoes are partially loaded into cargo tanks, strength assessments are to be carried out taking into account the cargo density and cargo loading height.
- <u>3</u> In structures where the membrane or axial force due to internal pressure cannot be neglected, the requirements in -1 above is to be applied with necessary modification.

#### Chapter 8 STRENGTH ASSESSMENT BY CARGO HOLD ANALYSIS

#### 8.5 Strength Assessment

#### 8.5.2 Buckling Strength Assessment

Paragraph 8.5.2.2 has been amended as follows.

#### 8.5.2.2 Side Shell Plating with Transverse Stiffness

In applying Annex 8.6, Part 1 "BUCKLING STRENGTH ASSESSMENT BASED ON CARGO HOLD ANALYSIS", for the side shell with stiffened in transverse direction which is surrounded by the bilge hopper tanks and top side tanks, the stress act on the members in the shorter side direction of the panels may be averaged regardless of the definition of the reference stress  $\sigma_y$ , which is in the shorter side direction of the panels specified in A3.2.2, Annex 8.6.

2 In the application of -1 above, where the requirements for hull girder ultimate strength specified in 5.4, Part 1 are satisfied, for the strength assessments performed in the equivalent design waves HM-1, FM-1, AV-1P and AV-1S of the maximum load condition, the stress under the shorter side direction of the panels acting on the said members is to be considered up to the value obtained by the following formula:

$$\sigma_{y\_limit} = \frac{\pi^2 E}{12(1 - \nu^2)} \left(\frac{t_p}{b}\right)^2$$

E: Young's modulus ( $N/mm^2$ ), taken as  $2.06 \times 10^5$ .

v: Poisson's ratio, taken as 0.3.

 $\underline{t_p}$ : Thickness (mm) of the plate panel.

b: Length (mm) of the shorter side of the plate panel.

#### Chapter 10 ADDITIONAL STRUCTURAL REQUIREMENTS

#### 10.1 Tank Structures for Sloshing

#### **10.1.1** General

#### **10.1.1.1 Application**

1 For the members of cargo tank structures of independent prismatic tanks type A/B which satisfy the following (1) to (3), the scantlings specified in this 10.1 are considered to be satisfied when obtained using the sloshing loads specified in 4.8.2.4, Part 1.

- (1) Independent prismatic tanks with volumes of not less than  $100 m^3$
- (2) Independent prismatic tanks designed for possible to loading at filling ratios of not less than 20 % and not more than 90 %
- (3) Where the period of longitudinal oscillation of independent prismatic tanks is within the range of  $\pm 20$  % of pitch period and within  $\pm 1.5$  seconds from the same period, and where the period of transverse oscillation of independent prismatic tanks is within the range of  $\pm 20$  % of roll period and within  $\pm 1.5$  seconds from the same period.
- 2 In applying -1(3) above, where only one of the conditions is applicable, only the sloshing load due to the relevant ship motion need be considered.
- <u>3</u> In applying -1(3) above, tank natural periods are to be calculated for each 10 % of the filling ratio, and only the sloshing load due to the filling ratio corresponding to the conditions of -1(3) above need be considered.

#### 10.1.1.2 Scantling Approach

The required scantlings specified in this **10.1** are to be net scantlings. Corrosion additions for independent prismatic tanks are to be in accordance with **3.3.4.3**, **Part 1**.

#### **10.1.2** Plates

#### **10.1.2.1** Tank Type A

The thickness of plates on which sloshing loads act is to be not less than the value obtained from the following formula.

$$t = \frac{b}{2} \sqrt{\frac{P_{slh} \times 10^{-3}}{1.15 C_a \sigma_{perm}}} \ (mm)$$

b: Length (mm) of the shorter side of the plate panel.

 $P_{s,lh}$ : Equivalent pressure  $(kN/m^2)$  for the plate panels, as specified in **Table 10.1.2-1**.

 $C_a$ : Coefficient of axial force effect, to be taken as 1.0.

 $\sigma_{nerm}$ : Permissible stress (N/mm<sup>2</sup>), as specified in **Table 10.1.2-2**.

Table 10.1.2-1 Equivalent Pressure for Plate Panels

	Member to be assessed	<u>P<sub>slh</sub></u>
- - -	Front and aft walls of tank Transverse wash bulkheads Tank top plates near front and aft walls of tank <sup>(1)</sup>	<u>P<sub>slh-p</sub></u> (4.8.2.4-4(1), Part 1)
	Tank side walls  Centreline bulkheads  Longitudinal wash bulkheads  Tank top plates near tank side walls <sup>(1)</sup> Sloping plates above and below tank side walls <sup>(2)</sup>	<u>P<sub>slh-r</sub></u> (4.8.2.4-4(2), Part 1)

#### Notes:

Numbers in parentheses indicate section number.

- (1)  $P_{slh-p}$  applies to plate panels within the range of  $0.3\ell_{tk}$  from the front and aft walls of tanks, while  $P_{slh-r}$  applies to plate panels within the range of  $0.3b_{tk}$  from centreline bulkheads / tank side walls. Definitions of  $\ell_{tk}$  and  $b_{tk}$  are specified in Table 4.8.2-13, Part 1 and Table 4.8.2-14, Part 1.
- (2) Notwithstanding (1) above, where large sloping plates are arranged between tank top plates and tank side walls, tank top plates may be excluded from the members to be assessed.

Table 10.1.2-2 Permissible Stress (Tank Type A)

<u>Member</u>	$\sigma_{perm}$	
- Ferrite steels	$min\ (0.79\sigma_{Y}, 0.53\sigma_{B})$	
Notes: $\sigma_V$ :Specified minimum yield stress $(N/mm^2)$		
$\frac{\sigma_V}{\sigma_R}$ : Specified minimum yield stress (N/mm²) $\frac{\sigma_R}{\sigma_R}$ : Specified minimum tensile stress at room temperature (N/mm²), to be taken as:		
For <i>KL</i> 24, taken as 400		
For <i>KL</i> 27, taken as 420		
For <i>KL</i> 33, taken as 440		
For <i>KL</i> 37, taken as 490		

#### **10.1.2.2** Tank Type B

The thickness of plates on which sloshing loads act is to be not less than the value obtained from the formula specified in 10.1.2.1. However, permissible stress  $\sigma_{perm}$  (N/mm<sup>2</sup>) is to be in accordance with Table 10.1.2-3.

Table 10.1.2-3 Permissible Stress (Tank Type B)

Member to be assessed	$\sigma_{perm}$	
- Nickel steels, carbon manganese steels	$\min (0.83\sigma_{Y}, 0.5\sigma_{B})$	
- Austenitic steels and aluminum alloys	min (0.83σ <sub>V</sub> , 0.4σ <sub>B</sub> )	
Notes: $\sigma_Y$ , $\sigma_B$ : As specified in <b>Table 10.1.2-2</b>		

#### 10.1.3 Stiffeners

#### 10.1.3.1

The section modulus of stiffeners attached to plates on which sloshing loads act is to be not less than the value obtained from the following formula.

$$\underline{Z = \frac{M_{slh}}{C_s \sigma_{perm}} \times 10^3 (cm^3)}$$

M<sub>slh</sub>: Equivalent bending moment (kN-m), as specified in **Table 10.1.3-1**.

 $\underline{C_s}$ : Coefficient of axial force effect, to be taken as 1.0.

 $\sigma_{perm}$ : Permissible stress (N/mm<sup>2</sup>), as specified in **Table 10.1.2-2** or **Table 10.1.2-3**.

Table 10.1.3-1 Equivalent Bending Moment for Each Member to be Assessed

Table 10.1.5-1 Equivalent Bending Women for Each Member to be Assessed		
Member to be assessed	Stiffened system	$\underline{M_{slh}}$
- Stiffeners attached to tank top plates <sup>(1)(2)</sup>	Longitudinal	$\frac{M_{slh-p}(4.8.2.4-5(1), Part 1)}{M_{slh-r}(4.8.2.4-5(2), Part 1)}$
	<u>Transverse</u>	$\frac{M_{slh-p}(4.8.2.4-5(2), Part 1)}{M_{slh-r}(4.8.2.4-5(1), Part 1)}$
Stiffeners attached to front and aft walls of tank     Stiffeners attached to transverse wash bulkheads     Stiffeners attached to vertical girders which are attached to centreline bulkheads	System A <sup>(3)</sup>	$\underline{M_{slh-p}}$ (4.8.2.4-5(1), Part 1)
Stiffeners attached to vertical girders which are attached to tank     side walls     Stiffeners attached to horizontal girders which are attached to     front and aft walls of tank     Stiffeners attached to cross-tie of transverse direction	System B <sup>(4)</sup>	<u>M<sub>slh=p</sub>(4.8.2.4-5(2), Part 1)</u>
<ul> <li>Stiffeners attached to tank side walls</li> <li>Stiffeners attached to longitudinal wash bulkheads</li> <li>Stiffeners attached to sloping plates above and below tank side walls</li> </ul>	System A (2)	<u>M<sub>slh=r</sub></u> (4.8.2.4-5(1), Part 1)
Stiffeners attached to vertical girders which are attached to front and aft walls of tank     Stiffeners attached to horizontal girders which are attached to tank side walls     Stiffeners attached to horizontal girders which are attached to centreline bulkheads	System B (3)	<u>M<sub>slh-r</sub></u> (4.8.2.4-5(2), Part 1)

#### Notes:

Numbers in parentheses indicate section number.

- (1)  $M_{slh-p}$  applies to stiffeners attached to plate panels within the range of  $0.3\ell_{tk}$  from the front and aft walls of tanks, while  $M_{slh-r}$  applies to stiffeners attached to plate panels within the range of  $0.3b_{tk}$  from centreline bulkheads / tank side walls. Definitions of  $\ell_{tk}$  and  $b_{tk}$  are specified in Table 4.8.2-13 and Table 4.8.2-14, Part 1.
- (2) Notwithstanding (1) above, where large sloping plates are arranged between tank top plates and tank side walls, stiffeners attached to the tank top plates may be excluded from the members to be assessed.
- (3) See Fig. 10.9.3-1, Part 1
- (4) See Fig. 10.9.3-2, Part 1

#### 10.1.4 Webs of Girders

#### <u>10.1.4.1</u>

The web thickness  $t_w$  for girders on which sloshing loads act is to be not less than the value obtained from the following formula.

$$\underline{t_w = \frac{b}{2} \sqrt{\frac{P_{slh} \times 10^{-3}}{1.15 C_a \sigma_{perm}} (mm)}}$$

 $P_{slh}$ : Equivalent pressure  $(kN/m^2)$  for plate panels, as specified in **Table 10.1.4-1**  $C_a$ : Coefficient of axial force effect, to be taken as 1.0  $E_a$ : Length  $E_a$ : Length  $E_a$ : Length  $E_a$ : Description of the shorter side of the plate panel

 $\sigma_{perm}$ : Permissible stress (N/mm<sup>2</sup>), as specified in **Table 10.1.2-2** or **Table 10.1.2-3** 

Table 10.1.4-1 Equivalent Pressure for Each Member to be Assessed

Member to be assessed	$\underline{P_{sth}}$
<ul> <li>Horizontal girders attached to front and aft walls of tanks / transverse wash bulkheads</li> <li>Vertical girders attached to tank side walls / centreline bulkheads / longitudinal wash bulkheads</li> <li>Cross-ties in transverse direction</li> </ul>	<u>P<sub>slh-p</sub></u> (4.8.2.4-4(1), Part 1)
<ul> <li>Horizontal girders attached to tank side walls / centreline bulkheads /         longitudinal wash bulkheads</li> <li>Vertical girders attached to front and aft walls of tanks / transverse wash bulkheads</li> <li>Cross-ties in longitudinal direction</li> </ul>	<u>P<sub>slh-r</sub></u> (4.8.2.4-4(2), Part 1)
Note: Numbers in parentheses indicate section number.	

## Part 2-10 SHIPS CARRYING LIQUEFIED GASES IN BULK (INDEPENDENT TANKS OF TYPE C)

#### Chapter 4 LOADS

#### 4.3 Loads to be Considered in Strength of Primary Supporting Structures

#### 4.3.2 Maximum Load Condition

Table 4.3.2-2 has been amended as follows.

Table 4.3.2-2 External and Internal Pressure to Be Considered

		$P_{DB}(kN/m^2)^{(1)}$ (2)	$P_{DS}(kN/m^2)^{(1)}$
5 11 1	<i>S</i> 1	$P_{exs} + P_{exw}$	$P_{exs} + P_{exw}$
Double bottom	<i>S</i> 2	$P_{exs} + P_{exw}$	$P_{exs} + P_{exw}$

#### Notes:

 $P_{exs}$ ,  $P_{exw}$ : Hydrostatic and Hydrodynamic pressure  $(kN/m^2)$  act on bottom shell in case of  $P_{DB}$ . Those values act on side shell in case of  $P_{DS}$ . Each value is calculated in accordance with **4.6.2.4**, **Part 1**.

<sup>(1)</sup> Load calculation points for calculating each component of loads such as  $P_{exs}$  is are to be in accordance with 7.3.1.5, Part 1 for all loading conditions.

<sup>(2)</sup> When calculating loads,  $T_{LC} = T_{SC}$ .

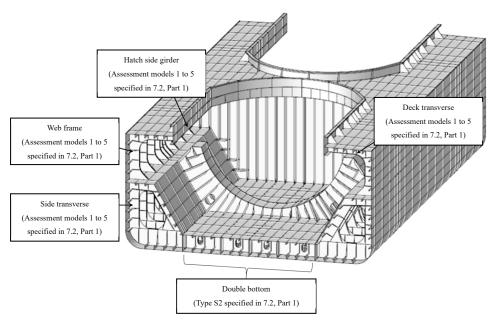
### Chapter 7 PRIMARY SUPPORTING STRUCTURE STRENGTH

#### 7.1 General

### 7.1.1 Application

Fig. 7.1.1-1 has been amended as follows.

Fig. 7.1.1-1 Application Example of Liquified Gas Bulk Carriers with Independent Tanks Type C



Note:

Where bottom structure is single bottom, the requirement of the simple girder specified in 7.2, Part 1 are is to be applied.

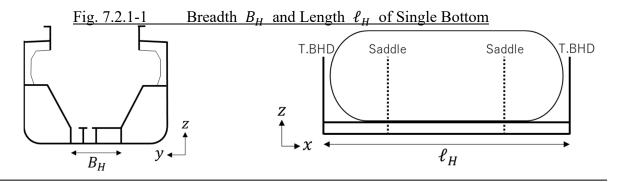
Section 7.2 has been added as follows.

#### 7.2 General

#### 7.2.1 Application

#### 7.2.1.1

Where the ship is single bottom and the aspect ratio  $\ell_H/B_H$  is 2.0 or more, to be in accordance with **7.2.2**.  $\ell_H$  and  $B_H$  are defined in **Fig. 7.2.1-1**. Where the aspect ratio is small, it is to be as deemed appropriate by the Society.



#### 7.2.2 Evaluation Members to be Assessed

#### **7.2.2.1** Floors

Strength assessments are to be carried out in accordance with the requirements for simple girders specified in 7.2 of Part 1. The full length  $\ell$  of the floor is to be  $B_H$  shown in Fig. 7.2.1-1.

#### **7.2.2.2 Girders**

Girder scantlings are to be of the same degree as floor scantlings.

## Part 2-11 SHIPS CARRYING LIQUEFIED GASES IN BULK (MEMBRANE TANKS)

#### Chapter 4 Loads

- 4.2 Loads to be Considered in Local Strength
- 4.2.2 Maximum Load Condition
- 4.2.2.1 Lateral Loads

Sub-paragraph -1 has been amended as follows.

In applying 4.4.2.4, Part 1, the parameters  $(GM, z_G)$  etc.) required to calculate the dynamic pressure due to cargo are to be the values in the loading condition that cargo is loaded only in the cargo hold to be considered and the draught is at a minimum (e.g. one-tank-loaded condition). The radius of gyration (m) around x-axis is to be taken as 0.38B, but a value calculated based upon the weight distribution according to the loading condition to be considered may be used.

### **Chapter 10 ADDITIONAL STRUCTURAL REQUIREMENTS**

Section 10.2 has been added as follows.

#### **10.2** Tank Structures for Sloshing

#### **10.2.1 General**

#### 10.2.1.1 Application

- 1 For the members of hull structures acting as boundaries of cargo tanks which satisfy the following (1) to (3), the scantlings specified in this 10.2 are considered to be satisfied when obtained using the sloshing loads specified in 4.8.2.4, Part 1.
- (1) Cargo tanks with volumes of not less than  $100 m^3$
- (2) Cargo tanks designed for possible loading at filling ratios of not less than 20 % and not more than 90 %
- (3) Where the period of longitudinal oscillation of cargo tanks is within the range of  $\pm 20$  % of pitch period and within  $\pm 1.5$  seconds from the same period, and where the period of transverse oscillation of cargo tanks is within the range of  $\pm 20$  % of roll period and within  $\pm 1.5$  seconds from the same period.
- 2 In applying -1(3) above, where only one of the conditions is applicable, only the sloshing load due to the relevant ship motion need be considered.
- In applying -1(3) above, the tank natural periods are to be calculated for each 10 % of the filling ratio, and only the sloshing load due to the filling ratio corresponding to the conditions of -1(3) above need be considered.

#### 10.2.1.2 Scantling Approach

The required scantlings specified in this **10.2** are to be net scantlings.

#### **10.2.2** Plates

#### 10.2.2.1

The thickness of plates on which sloshing loads act is to be not less than the value obtained from the formula specified in 10.9.2, Part 1. Equivalent pressure is to be in accordance with Table 10.2.2-1.

Table 10.2.2-1 Equivalent Pressure for Plate Panels

Table 10.2.2-1 Equivalent Hessure for Hater anels		
Members to be assessed	$\underline{P_{slh}}$	
<ul> <li>Transverse bulkheads</li> <li>Tank top plates near transverse bulkheads</li> </ul>	<u>C<sub>m</sub>P<sub>slh-p</sub></u> (4.8.2.4-4(1), Part 1)	
<ul> <li>Longitudinal bulkheads</li> <li>Inner deck sloping plates</li> <li>Bilge hopper plating</li> </ul>	C <sub>m</sub> P <sub>slh-r</sub> (4.8.2.4-4(2), Part 1)	
Notes: Numbers in parentheses indicate section number.  C <sub>m</sub> : Coefficient, to be taken as 0.85		

#### 10.2.3 Stiffeners

#### 10.2.3.1

The section modulus of stiffeners attached to plates on which sloshing loads act is to be not

less than the value obtained from the formula specified in 10.9.3, Part 1. Equivalent bending moments are to be in accordance with Table 10.2.3-1.

Table 10.2.3-1 Equivalent Bending Moment for Each Member to be Assessed

Member to be assessed	Stiffened system	<u>M<sub>slh</sub></u>
- Stiffeners attached to transverse bulkheads	<u>Vertical</u>	$C_m M_{slh-p}$ (4.8.2.4-5(1), Part 1)
	<u>Horizontal</u>	$C_m M_{slh-p}$ (4.8.2.4-5(2), Part 1)
- Stiffeners attached to longitudinal bulkheads - Stiffeners attached to inner deck sloping plates - Stiffeners attached to bilge hopper plating	Longitudinal	<u>C<sub>m</sub>M<sub>slh-r</sub></u> (4.8.2.4-5(1), Part 1)
Notes: Numbers in parentheses indicate section number. $C_m$ : Coefficient, to be taken as 0.85		

#### EFFECTIVE DATE AND APPLICATION

- **1.** The effective date of the amendments is 1 July 2023.
- **2.** Notwithstanding the amendments to the Rules, the current requirements apply to the following ships:
  - (1) ships for which the date of contract for construction is before the effective date; or
  - (2) sister ships of ships subject to the current requirements for which the date of contract for construction is before 1 January 2025.

# GUIDANCE FOR THE SURVEY AND CONSTRUCTION OF STEEL SHIPS

Part C

**Hull Construction and Equipment** 

2023 AMENDMENT NO.1

Notice No.28 30 June 2023

Resolved by Technical Committee on 25 January 2023

Notice No.28 30 June 2023 AMENDMENT TO THE GUIDANCE FOR THE SURVEY AND CONSTRUCTION OF STEEL SHIPS

"Guidance for the survey and construction of steel ships" has been partly amended as follows:

### Part C HULL CONSTRUCTION AND EQUIPMENT

### Part 1 GENERAL HULL REQUIREMENTS

#### Amendment 1-1

#### C10 ADDITIONAL STRUCTURAL REQUIREMENTS

Section C10.9 has been added as follows.

#### C10.9 Tank Structures for Sloshing

#### C10.9.1 General

#### C10.9.1.1 Application

- It is recommended that the design of tank structure satisfy the following (1) and (2).
- (1) The period of longitudinal oscillation of liquid cargo tanks is not within the range of  $\pm 20$  % of pitch period and not within  $\pm 1.5$  seconds from the same period.
- (2) The period of transverse oscillation of liquid cargo tanks is not within the range of  $\pm 20$  % of roll period and not within  $\pm 1.5$  seconds from the same period.

#### EFFECTIVE DATE AND APPLICATION (Amendment 1-1)

- **1.** The effective date of the amendments is 1 July 2023.
- 2. Notwithstanding the amendments to the Guidance, the current requirements apply to the following ships:
  - (1) ships for which the date of contract for construction is before the effective date; or
  - (2) sister ships of ships subject to the current requirements for which the date of contract for construction is before 1 January 2025.

#### Amendment 1-2

#### C13 RUDDERS

#### C13.2 Rudders

#### C13.2.1 General

#### C13.2.1.2 Materials

Sub-paragraph -2 has been amended as follows.

2 Rolled bar steel (*KSFR45KSFR440-M*) may be treated in the same way as *KSF45KSF440-M*.

#### EFFECTIVE DATE AND APPLICATION (Amendment 1-2)

- 1. The effective date of the amendments is 1 July 2023.
- 2. Notwithstanding the amendments to the Guidance, the current requirements apply to ships for which the date of contract for construction\* is before the effective date.
  - \* "contract for construction" is defined in the latest version of IACS Procedural Requirement (PR) No.29.

#### IACS PR No.29 (Rev.0, July 2009)

- 1. The date of "contract for construction" of a vessel is the date on which the contract to build the vessel is signed between the prospective owner and the shipbuilder. This date and the construction numbers (i.e. hull numbers) of all the vessels included in the contract are to be declared to the classification society by the party applying for the assignment of class to a newbuilding.
- 2. The date of "contract for construction" of a series of vessels, including specified optional vessels for which the option is ultimately exercised, is the date on which the contract to build the series is signed between the prospective owner and the shipbuilder. For the purpose of this Procedural Requirement, vessels built under a single contract for construction are considered a "series of vessels" if they are built to the same approved plans for classification purposes. However, vessels within a series may have design alterations from the original design provided:
  - (1) such alterations do not affect matters related to classification, or
  - (2) If the alterations are subject to classification requirements, these alterations are to comply with the classification requirements in effect on the date on which the alterations are contracted between the prospective owner and the shipbuilder or, in the absence of the alteration contract, comply with the classification requirements in effect on the date on which the alterations are submitted to the Society for approval.

The optional vessels will be considered part of the same series of vessels if the option is exercised not later than 1 year after the contract to build the series was signed.

- 3. If a contract for construction is later amended to include additional vessels or additional options, the date of "contract for construction" for such vessels is the date on which the amendment to the contract, is signed between the prospective owner and the shipbuilder. The amendment to the contract is to be considered as a "new contract" to which 1. and 2. above apply.
- 4. If a contract for construction is amended to change the ship type, the date of "contract for construction" of this modified vessel, or vessels, is the date on which revised contract or new contract is signed between the Owner, or Owners, and the shipbuilder.

#### Note:

This Procedural Requirement applies from 1 July 2009.